

The Arching of Beams  
By  
Farid A. Chouery<sup>1</sup>, PE, SE  
©2012, 2020 by Farid A. Chouery all rights reserved

**Abstract:**

The following solution was found using meditative techniques. The problem of arching of beams for dead load and live load is presented for infinite beams under an elastic media sitting on columns. The problem starts by solving the biharmonic equation using Fourier series. The solution presented is for a uniform load and gravity load. Other potential solution is architected at the end of the article. Charts and tables for simple span beam and fixed span beam is presented for an infinite elastic media and a finite elastic media with a given height. This problem has never been solved before.

**Introduction:**

The ancient problem of curved beams arching, explored by the Greeks and Romans, is applied, in this paper, to a straight beam instead of an arch. The following solution offers a new insight into the phenomenon of a straight beam behaving as an arch. Using finite element (FE), one observes the arching action when looking at the load distribution of a stress diagram. How, though, can FE be used to find the reductions in moment and shear due to the redistribution of the dead and live loads caused by arching? This dilemma is especially apparent in the analysis of concrete because FE does not accept reinforcing rebar; instead, 3-D analysis is needed. Additionally, modeling becomes very cumbersome if FE analysis must be performed for every beam in a structure. It would be much more productive if one knew the arching factors beforehand. Elasticity will allow the exploration of this phenomenon in all materials, including soils (for small strains).

In a previous paper, the problem was solved with an infinite soil or material on top of a beam in one bay, or infinite bay (see <http://www.facsystems.com/LAGGING.pdf>). If bricks are substituted for soil, steel channel for wood lagging, and columns for soldier piles, then it can be concluded that more arching reduction would result as the columns become thicker.

Additionally, the shear reduction factor is different than the bending moment reduction factor. The solution in that paper is very precise with less than 2% difference between the results calculated using the reduction factors and those calculated directly. Utilizing the substitutions above, we will explore a confined soil or material, on top of a beam, of which the material ends at a distance  $h$  above the beam. We will also assume a live load on top of the material. Future research will explore the arching of materials without a beam at the bottom, including wide flange steel beams.

This paper represents the latest iteration of a thought process initiated during my master's Thesis in which I proposed a correction to an error by W. D. Liam Finn, PhD, in his paper, "Boundary Value Problems of Soil Mechanics." My goal was to modify his method to prove the existence of

---

<sup>1</sup> Structural, Electrical and Foundation Engineer, FAC Systems Inc., 6738 19<sup>th</sup> Ave. NW, Seattle, WA

arching. The conclusion was discovering the arching factors for wood lagging in shoring walls. There was an error in differentiating the deflection, however, because the deflection, expressed as a Fourier series, was calculated by taking the fourth derivative of the series. This represented an error in the boundary condition because differentiating a Fourier series is not allowed. Furthermore, ignoring the load on top of the pile by taking a Fourier series on the lagging only was incorrect because the load on top of the pile affects the arching and should be considered. Regardless of the error, though, the master's Thesis mathematically proves the existence of arching.

The thesis error was corrected in the article cited above. It is evident from the collocation function method how to arrive at the solution by integrating the stress to equate the deflection. What is not clear, however, is how to utilize the Fourier series representation to obtain the exact deflection. Further research is needed. In the foregoing analysis, the exact solution would be easier to implement than the collocation function.

This paper, for which the master's thesis research and the web article paved the road, will evaluate the infinite bay solution instead of the one bay solution, which are similar when using the Fourier Transform instead of the Fourier series.

Keeping in mind that if there is no deflection, as in rigid beam, then there is no arching and the full load must be taken into account, we start with the phi function in elasticity as a solution to the biharmonic equation:

Where  $x$  and  $y$  are the coordinates axis and  $A, B, C$  and  $D$  are constant and at  $n = \phi$  is at  $n \rightarrow 0$  but divide by 2 as in the Fourier cosine series first term.  $\alpha_n$  is a function of  $n$ ,  $\alpha_n = \frac{n\pi}{b+t}$ . Where  $2b$  is the bay width and  $2t$  is the column width as in Fig. 1. Thus, the stress is

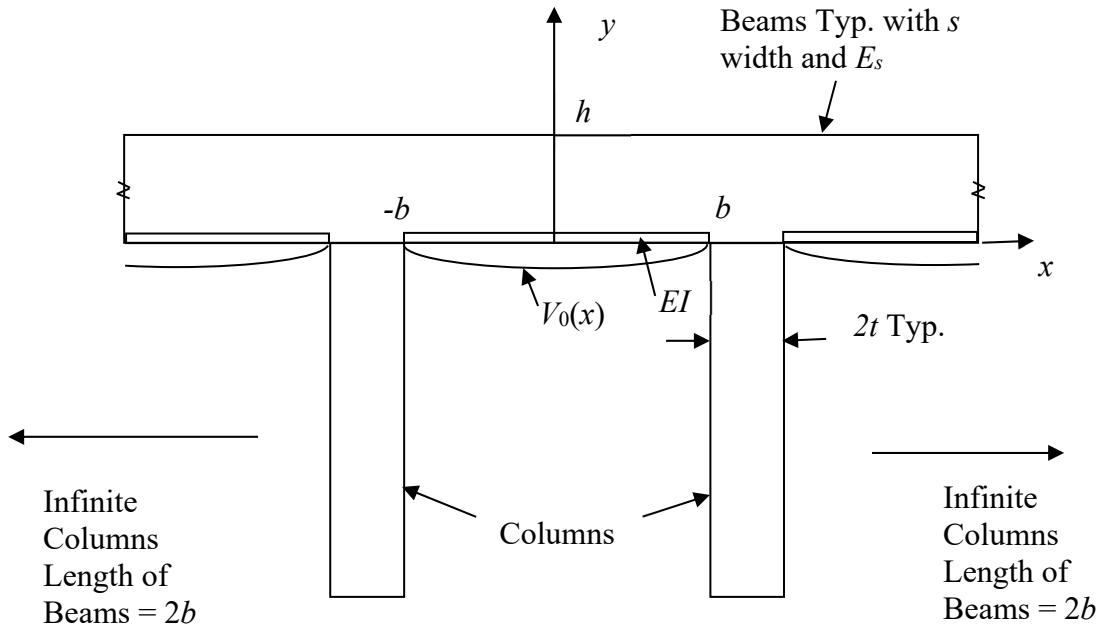


Fig. 1 Beams and Columns Diagram with Bottom Beam

At  $y = h$  there is the live load  $-q$  expressed as a Fourier series =  $\sum_{n=0}^{\infty} w_n \cos \alpha_n x$  ..... 3

Thus:

$$(A + B\alpha_n h)e^{\alpha_n h} + (C + D\alpha_n h)e^{-\alpha_n h} = -w_n \dots \quad 4$$

Or

$$A = -(C + D\alpha_n h)e^{-2\alpha_n h} - B\alpha_n h - w_n e^{-\alpha_n h} \dots \quad 5$$

Now

We will assume the shear is zero at  $y = h$ , thus from Eq. 6 yields:

Substituting Eq. 5 in Eq. 7 and solve for  $B$  yields:

$$B = (2C - D + 2D\alpha_n h)e^{-2\alpha_n h} + w_n e^{-\alpha_n h} \dots \quad 8$$

Substitute Eq. 8 in Eq. 5 yields:

We will assume at  $y = 0$  the shear is zero, thus from Eq. 6 yields

$$\begin{aligned}
& -(C + 2C\alpha_n h + 2D\alpha_n^2 h^2)e^{-2\alpha_n h} - w_n(\alpha_n h + 1)e^{-\alpha_n h} + (2C - D + 2D\alpha_n h)e^{-2\alpha_n h} \\
& \quad + w_n e^{-\alpha_n h} - C + D = 0 \\
& -(C + 2C\alpha_n h)e^{-2\alpha_n h} + 2Ce^{-2\alpha_n h} - C \\
& \quad = -D + 2D\alpha_n^2 h^2 e^{-2\alpha_n h} + De^{-2\alpha_n h} - 2D\alpha_n h e^{-2\alpha_n h} + w_n(\alpha_n h + 1)e^{-\alpha_n h} \\
& \quad - w_n e^{-\alpha_n h} \\
& [-(1 + 2\alpha_n h)e^{-2\alpha_n h} + 2e^{-2\alpha_n h} - 1]C \\
& \quad = [-1 + 2\alpha_n^2 h^2 e^{-2\alpha_n h} + e^{-2\alpha_n h} - 2\alpha_n h e^{-2\alpha_n h}]D + w_n \alpha_n h e^{-\alpha_n h}
\end{aligned}$$

Substitute Eq. 8 and Eq. 9 in Eq. 10 and solve for  $C$  yields:

Let

$$C = c_n D + w_{n3} w_n \dots \quad 12$$

Where

$$c_n = \frac{-1 + (1 - 2\alpha_n h + 2\alpha_n^2 h^2)e^{-2\alpha_n h}}{(1 - 2\alpha_n h)e^{-2\alpha_n h} - 1} = 1 + \frac{2\alpha_n^2 h^2 e^{-2\alpha_n h}}{(1 - 2\alpha_n h)e^{-2\alpha_n h} - 1}$$

At  $h = \infty$   $c_n = 1$

And

$$w_{n3} = \frac{\alpha_n h e^{-\alpha_n h}}{(1 - 2\alpha_n h)e^{-2\alpha_n h} - 1}$$

Substitute Eq. 12 in Eq. 9 and solve for  $A$  yields:

$$A = -\left( [1 + 2\alpha_n h][c_n D + w_{n3} w_n] + 2D\alpha_n^2 h^2 \right) e^{-2\alpha_n h} - w_n(\alpha_n h + 1) e^{-\alpha_n h}$$

Where

$$a_n = -[c_n(1 + 2\alpha_n h) + 2\alpha_n^2 h^2]e^{-2\alpha_n h}$$

At  $h = \infty$   $a_n = 0$

And

$$w_{n1} = -w_{n3}(1 + 2\alpha_n h)e^{-2\alpha_n h} - (1 + \alpha_n h)e^{-\alpha_n h}$$

Substitute Eq. 12 in Eq. 8 yields:

Where

$$b_n = [2c_n - 1 + 2\alpha_n h]e^{-2\alpha_n h}$$

At  $h = \infty$   $b_n = 0$

And

$$w_{n2} = 2w_{n3}e^{-2\alpha_n h} + e^{-\alpha_n h}$$

So,  $A$ ,  $B$  and  $C$  are functions of  $D$ . Now we seek to find  $D$ .

To find  $D$  we need to find the deflection, the strain in the  $y$  direction is:

Where  $\beta = \frac{1-\nu^2}{E_s}$  and  $\rho = \frac{\nu(1+\nu)}{E_s}$  for plain strain and  $\beta = \frac{1}{E_s}$  and  $\rho = \frac{\nu}{E_s}$  for plain stress,  $\nu$  is Poisson's ratio and  $E_s$  is the elastic modulus of the material on top of the beam and

$$\sigma_y = - \sum_{n=\varphi}^{\infty} \cos \alpha_n x [(A + B\alpha_n y)e^{\alpha_n y} + (C + D\alpha_n y)e^{-\alpha_n y}] - p \left(1 - \frac{y}{h}\right)$$

$$\begin{aligned}\sigma_x &= \frac{\partial^2 \phi}{\partial y^2} = \sum_{n=\varphi}^{\infty} \cos \alpha_n x [(A + 2B + B\alpha_n y) e^{\alpha_n y} + (C - 2D + D\alpha_n y) e^{-\alpha_n y}] \\ \sigma_x &= \sum_{n=\varphi}^{\infty} \cos \alpha_n x [(A + B\alpha_n y) e^{\alpha_n y} + (C + D\alpha_n y) e^{-\alpha_n y}] \\ &\quad + 2 \sum_{n=\varphi}^{\infty} \cos \alpha_n x [Be^{\alpha_n y} - De^{-\alpha_n y}] - pk_0 \left(1 - \frac{y}{h}\right)\end{aligned}$$

$$\begin{aligned}\sigma_x = & k_0 \sum_{n=\varphi}^{\infty} \cos \alpha_n x [(a_n D + k_0 w_{n1} w_n + (b_n D + k_0 w_{n2} w_n) \alpha_n y) e^{\alpha_n y} \\ & + (c_n D + k_0 w_{n3} w_n + D \alpha_n y) e^{-\alpha_n y}] \\ & + 2k_0 \sum_{n=\varphi}^{\infty} \cos \alpha_n x [(b_n D + k_0 w_{n2} w_n) e^{\alpha_n y} - D e^{-\alpha_n y}] - p k_0 \left(1 - \frac{y}{h}\right) \dots \dots . 17\end{aligned}$$

$$\begin{aligned}\tau_{xy} &= -\frac{\partial^2 \varphi}{\partial x \partial y} = \sum_{n=\varphi}^{\infty} \sin \alpha_n x [(A + B + B\alpha_n y)e^{\alpha_n y} + (-C + D - D\alpha_n y)e^{-\alpha_n y}] \\ \tau_{xy} &= \sum_{n=\varphi}^{\infty} \sin \alpha_n x [(a_n D + w_{n1} w_n + b_n D + w_{n2} w_n + (b_n D + w_{n2} w_n)\alpha_n y)e^{\alpha_n y} \\ &\quad + (-(c_n D + w_{n3} w_n) + D - D\alpha_n y)e^{-\alpha_n y}]\end{aligned}$$

$k_0 = \frac{\rho}{\beta}$  this is because we have the strain  $\epsilon_x$  is zero everywhere or the beam has infinite moment of inertia. This is necessary to include the contact shear as described in

<http://www.facsystems.com/Contact.pdf> Where the self-weight being  $p \left(1 - \frac{y}{h}\right)$ . We assume the contact surface of the live load on the mass media is a rough surface. If it is a smooth surface  $k_0 = 1.0$  but for the self-weight  $k_0 = \frac{\rho}{\beta}$ .

Thus:

Note: the summation sign and  $\cos \alpha_n x$  is not shown for simplicity.

But

$$\varepsilon_y = \frac{\partial V(x)}{\partial y}$$

Then

$$V(x) = \left[ -\beta A - \rho(A + 2B) \right] \frac{e^{\alpha_n y}}{\alpha_n} - (-\beta - \rho)B \frac{e^{\alpha_n y}}{\alpha_n} + (-\beta - \rho)Bye^{\alpha_n y} \\ + \left[ -\beta C - \rho(C - 2D) \right] \frac{e^{-\alpha_n y}}{-\alpha_n} - (-\beta - \rho)D \frac{e^{-\alpha_n y}}{\alpha_n} - (-\beta - \rho)Dye^{-\alpha_n y} + g(x) \quad ..... 20$$

At  $y = h \rightarrow \infty$  and  $w_n = 0$ ,  $V(x) = 0$  thus  $g(x) = 0$ . Note:  $A, B$  de-solves the equation to zero.

At  $y = 0$  we have:

At  $C = D$  and  $A = 0$  and  $B = 0$  at  $h \rightarrow \infty$  we have:

$$V_0(x) = \sum_{n=0}^{\infty} \frac{2\beta D}{\alpha_n} \cos \alpha_n x \text{ matching the equation in the article at}$$

<http://www.facsystems.com/LAGGING.pdf>

Substitute Eq. 12, 13 and 14 in Eq. 21 yields:

$$V_0(x) = - \sum_{n=\varphi}^{\infty} \frac{\cos \alpha_n x}{\alpha_n} \{ [\beta(a_n - b_n - c_n - 1) + \rho(a_n + b_n - c_n + 1)] D + \beta(w_{n1} - w_{n2} - w_{n3}) w_n \\ + \rho(w_{n1} + w_{n2} - w_{n3}) w_n \}$$

..... 20

$$V_0(x) = - \sum_{n=\varphi}^{\infty} \frac{\beta \cos \alpha_n x}{\alpha_n} \left\{ \left[ (a_n - b_n - c_n - 1) + \frac{\rho}{\beta} (a_n + b_n - c_n + 1) \right] D + (w_{n1} - w_{n2} - w_{n3}) w_n + \frac{\rho}{\beta} (w_{n1} + w_{n2} - w_{n3}) w_n \right\}$$

$$\begin{aligned} & \sum_{n=\varphi}^{\infty} v_n \cos \alpha_n x \\ &= - \sum_{n=\varphi}^{\infty} \frac{\beta \cos \alpha_n x}{\alpha_n} \left\{ \left[ (a_n - b_n - c_n - 1) + \frac{\rho}{\beta} (a_n + b_n - c_n + 1) \right] D + (w_{n1} - w_{n2} - w_{n3}) w_n \right. \\ & \quad \left. + \frac{\rho}{\beta} (w_{n1} + w_{n2} - w_{n3}) w_n \right\} \end{aligned}$$

$$-\frac{\alpha_n v_n}{\beta} = \left[ (a_n - b_n - c_n - 1) + \frac{\rho}{\beta} (a_n + b_n - c_n + 1) \right] D + (w_{n1} - w_{n2} - w_{n3}) w_n \\ + \frac{\rho}{\beta} (w_{n1} + w_{n2} - w_{n3}) w_n$$

$$V_0(x) = \sum_{n=0}^{\infty} v_n \cos \alpha_n x$$

where  $v_n = \frac{2}{b} \int_0^b v_0(\lambda) \cos \alpha_n \lambda d\lambda$  and  $v_0(\lambda) = V_0$  is the deflection

Thus

$$D = -\frac{\frac{\alpha_n v_n}{\beta} + (w_{n1} - w_{n2} - w_{n3})w_n + \frac{\rho}{\beta}(w_{n1} + w_{n2} - w_{n3})w_n}{(a_n - b_n - c_n - 1) + \frac{\rho}{\beta}(a_n + b_n - c_n + 1)} \dots \dots \dots 22$$

$\sigma_y$  @  $y = 0$  from Eq. 2 and Eq. 16 yields:

$$\sigma_y = -\sum_{n=\varphi}^{\infty} \cos \alpha_n x(A + C) = -\sum_{n=\varphi}^{\infty} \cos \alpha_n x[(a_n + c_n)D + (w_{n1} + w_{n3})w_n] - p$$

..... 23

Substitute Eq. 22 in 23 yields:

And  $p$  is the self-weight.

Where:

$$k_n = -\frac{a_n + c_n}{d_n}$$

and

$$W_n = -\frac{(a_n + c_n) \left[ (w_{n1} - w_{n2} - w_{n3})w_n + \frac{\rho}{\beta} (w_{n1} + w_{n2} - w_{n3})w_n \right]}{(a_n - b_n - c_n - 1) + \frac{\rho}{\beta} (a_n + b_n - c_n + 1)} + (w_{n1} + w_{n3})w_n$$

And

$$d_n = (a_n - b_n - c_n - 1) + \frac{\rho}{\beta} (a_n + b_n - c_n + 1)$$

Now from equation 10

$$A + B - C + D = 0 = a_n D + w_{n1}w_n + b_n D + w_{n2}w_n - c_n D - w_{n3}w_n + D = 0$$

$$a_n + b_n - c_n + 1 = 0$$

And

$$w_{n1} + w_{n2} - w_{n3} = 0$$

Thus

And

And

Since at  $y = 0$

$$\int_0^{b+t} \sigma_y dx = -p(b+t) - q(b+t)$$

Then the  $n = 0$  first coefficient drops and  $n$  starts at 1 for  $k_n$  factor since it does not involve  $w_n$ . For example, if  $q = 0$ , then the equation must hold for all  $n$ . We will use a parabola collocation function

$$\bar{v}_0(z) = -d_1 \delta [1 - (x/b)^2]$$

where  $\delta = 5sb^4p/(24EI)$  for dead loads, and  $\delta = 5sb^4q/(24EI)$  for live loads. The charts will be for dead load only and live load only and  $d_1$  is a dimensional constant and will vary with  $h$ . Thus,

$$v_n = -\frac{4d_1\delta}{b+t} \left( \frac{1}{b^2\alpha_n^3} \sin \alpha_n b - \frac{1}{b\alpha_n^2} \cos \alpha_n b \right) \dots \quad 28$$

It is clear that at  $n \rightarrow 0$   $v_n = -\frac{2}{3}d_1\delta b$  and at  $n = 0$  in the left term of Eq. 24 is zero and the first term drops. For the live load:

$$w_0 = -\frac{q}{b+t}(b+t-l)$$

Rearrange for the stress functions:

$$\begin{aligned}\sigma_y = & - \sum_{n=0}^{\infty} \cos \alpha_n x [(a_n D + w_{n1} w_n + (b_n D + w_{n2} w_n) \alpha_n y) e^{\alpha_n y} \\ & + (c_n D + w_{n3} w_n + D \alpha_n y) e^{-\alpha_n y}] - p \left(1 - \frac{y}{h}\right)\end{aligned}$$

Where:

The full stress functions will be addressed later. Now at  $y = 0$ .

where  $l$  is a small negligible distance.

Thus, the shear is:

Note:  $n$  starts at 1 since  $W_n$  as  $\alpha_n \rightarrow \infty$  is a finite number. Thus, the shear goes to infinity at  $n = 0$ , but, since the shear is finite and equal to  $p(b+t) + q(b+t)$ ,  $n = 0$  drops. Conversely, the constant of integration will de-solve it to have the shear as an odd function.

The shear reduction for dead load is:

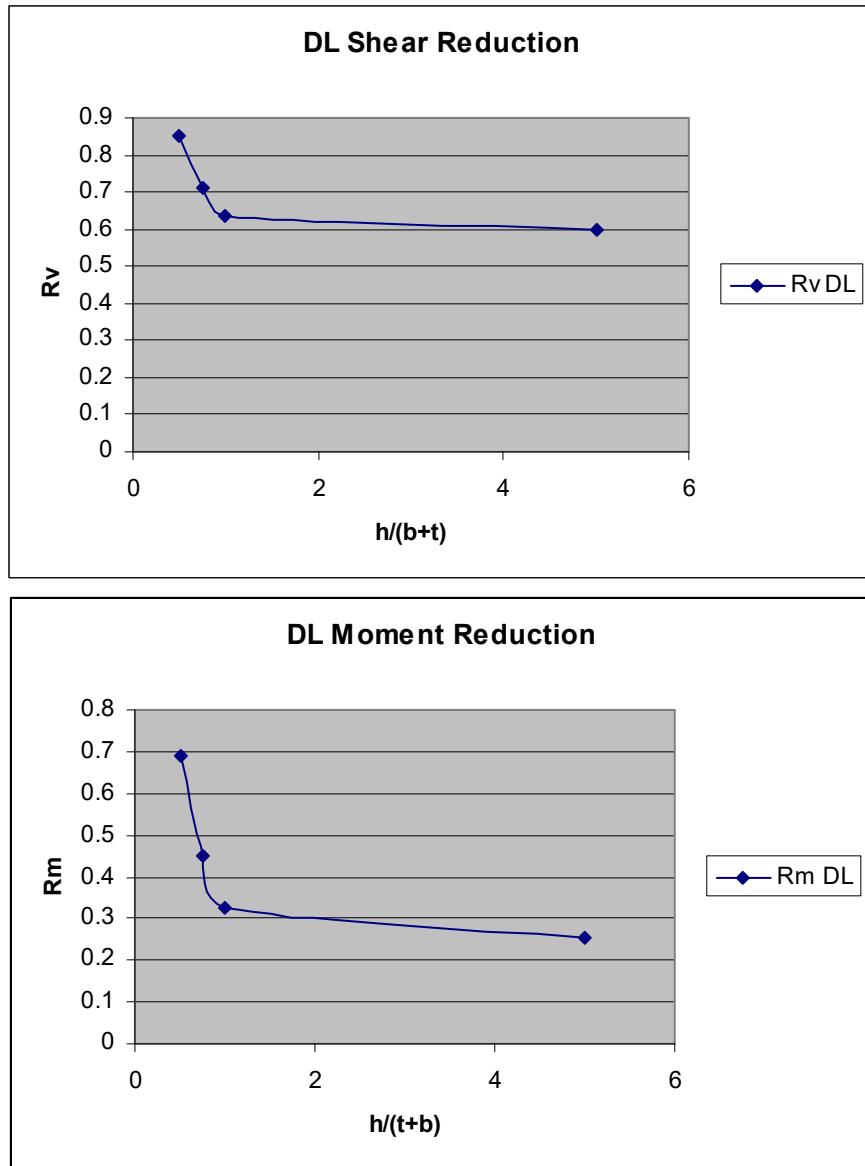
The shear reduction for live load is:

The moment is

Where  $C_1$  is a constant of integration, the moment reduction for dead load is:

The moment reduction for live load is:

See Fig 2 and 3 for arching reductions

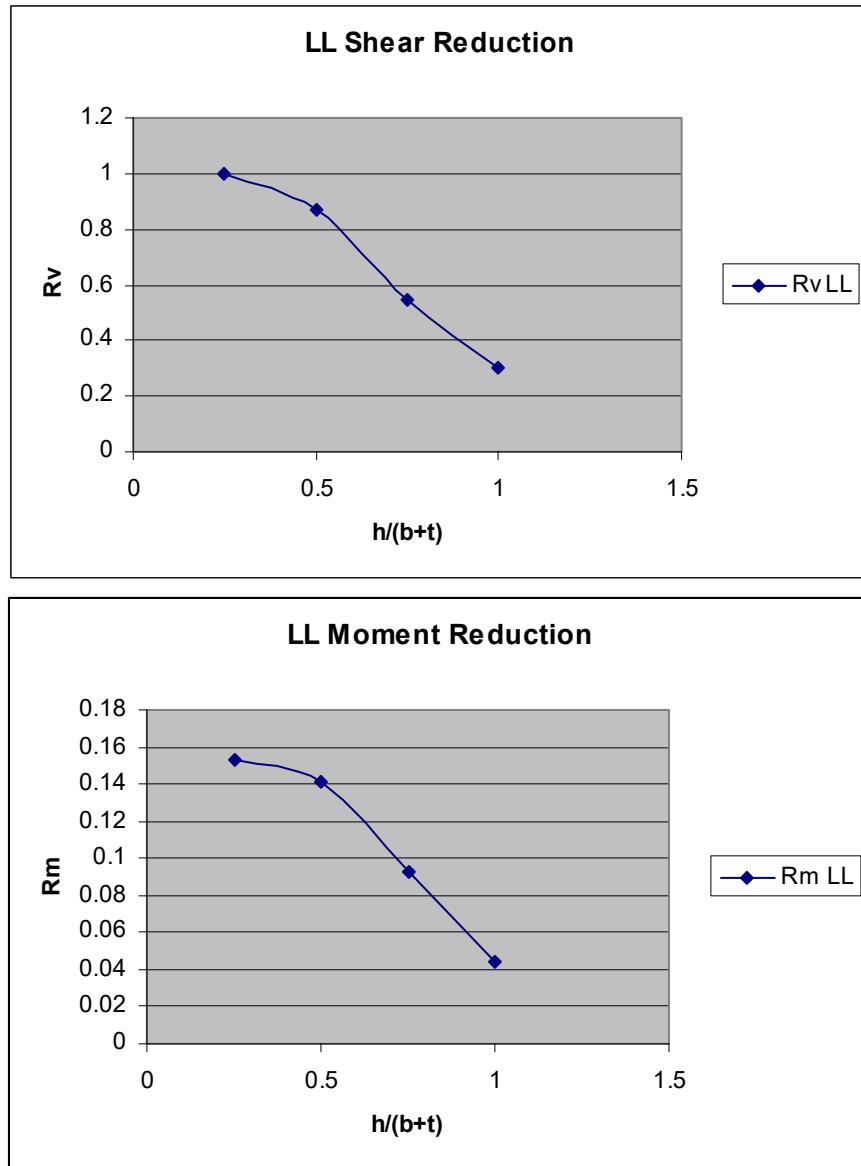


**Figure 2 – Two Plots of the Reduction Factors due to Arching for DL**

**Note:**

$$\begin{aligned}
 d_1 &= 0.282667 \\
 \frac{\delta}{\beta pb} &= 6.25 \\
 b/(b+t) &= 0.667 \\
 \nu &= 0.3 \text{ Soils}
 \end{aligned}$$

Note:  $h/(b+t)$  will change  $d_1$  and it is left constant for the purpose of demonstration  
 Charts are for demonstration only and not to be used until final charts  
 are derived from research



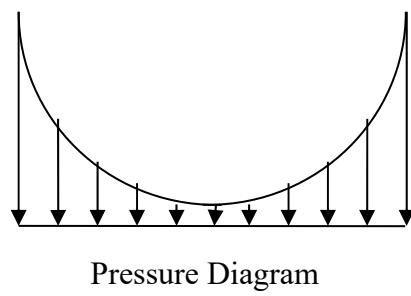
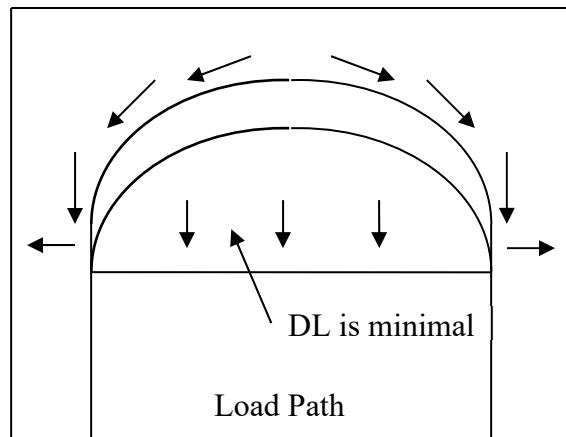
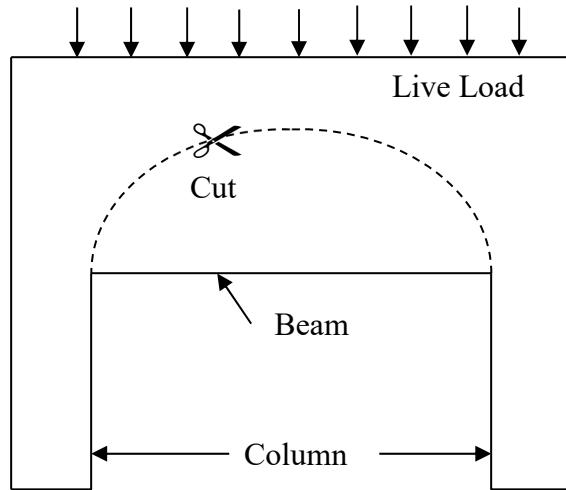
**Figure 3 – Two Plots of the Reduction Factors due to Arching for LL**

**Note:**

$$\begin{aligned}
 d_1 &= 0.01 \\
 \frac{\delta}{\beta q b} &= 6.25 \\
 b/(b+t) &= 0.667 \\
 V &= 0.3 \quad \text{Soils} \\
 (b+t-l)/(b+t) &= 0.85
 \end{aligned}$$

Note:  $h/(b+t)$  will change  $d_1$  and it is left constant for the purpose of demonstration  
 Charts are for demonstration only and not to be used until final charts  
 are derived from research

Figure 4 shows the mechanics underlying the loading.



**Figure 4 – When cut to an arch the load path and pressure diagram is exposed  
If the removed portion is put back the arch does not disappear**

### Closed form solution #1 simple span lagging with $h = \infty$ :

Starting with at  $C = D$  and  $A = 0$  and  $B = 0$  at  $h \rightarrow \infty$  we have:

$$V_0(x) = \sum_{n=0}^{\infty} \frac{2\beta D}{\alpha_n} \cos \alpha_n x \text{ matching the equation in the article at}$$

<http://www.facsystems.com/LAGGING.pdf>

We have the stress and reaction yields:

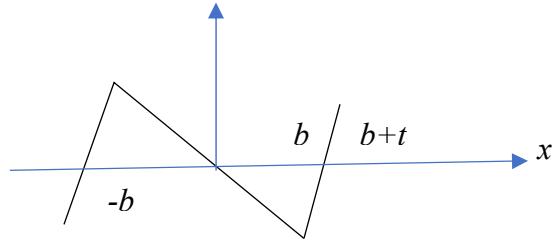
$$\sigma_y|_{y=0} = - \sum_{n=1}^{\infty} \frac{s\alpha_n v_n}{2\beta} \cos \alpha_n x + p \frac{b}{t} \quad b \leq x \leq b+t \dots\dots\dots 39$$

In the past we forced the deflection under the column to be zero which gave infinite stress at the corners or at  $x = \pm b$ . In here we do it from stress requirements. It can be seen by setting  $1/\beta = 0$  or water the stress it  $-p$  at the lagging in the region  $-b \leq x \leq b$  and  $pb/t$  reaction at the column. This is necessary so the summation of forces to be zero. Where,  $-p$  is in lb/ft and the stress is in strip  $s =$  width of lagging board in the z-direction or thickness of the plate. The reaction at the column is portrait as positive it does not mean tension the final solution at the column must be reversed or multiplied by -1 to show compression at the column resulting in increased stress at the corners. This will limit the solution since when  $t/b$  is large the stress at the column from the lagging is limited to the bearing area at the corners of the column and does not approach zero. However, the stress has an average of  $pb/t$  so the stress at the column is not uniform we assume smaller  $t$ . The stress  $-p$  at the column is not shown since the reaction will cancel it out with  $p$ .

It was found from the above article that at  $t/b > 0.5$  the deflection at the column can be zero for all  $0.5 < t/b < \infty$  and the solution is that of  $t/b = 0.5$  can be used as a conservative solution  $t/b > 0.5$ . The moment reduction is around 2.2% over and 15.5% under for shear reduction this percentage will be found in the following solution. The stress at  $x = 0$  matches very closely  $A = 16.53$  versus 16.5 gives zero pressure. The reason of this difference is the stress in the past article at  $x = b$  is infinitely negative or compressive and does not have a value but the stress. However, in the new solution it has a value and makes more sense and the deflection at the column corners is not quite zero. So, I would trust the new following solution. Note at  $x = 0$  both stress and moment are very close in both solutions.

The shear yields:

The coefficient of integration is zero since the shear is zero at  $x = 0$ . We can rewrite in Fourier sine series in the interval  $-b-t \leq x \leq b+t$  where the line in the left side of the equation has an active shear with slope  $-p$  in  $-b \leq x \leq b$  and a reactive shear at the columns with a slope  $pb/t$  for equilibrium. So, when differentiating over the entire  $x$  domain we obtain active pressure  $-p$  for arching and a reactive pressure  $pb/t$  for the column reaction. If  $b = t$  there is a load  $-p$  at the column there is a reaction equal  $p$  for that load, and they cancel each other out. This is necessary allowing downward deflection at lagging and upward deflection at columns. Thus, when multiplying the stress by the distances to obtain the forces in the  $y$  direction the summation of forces is zero see Figure 5, yields



**Figure 5, Saw tooth shear or rotation**

$$\text{the saw tooth is} = -\frac{2p}{t} \sum_{n=1}^{\infty} \frac{\sin \alpha_n b}{\alpha_n^2} \sin \alpha_n x$$

The moment yields

When  $v_n = 0$  or  $\beta = \infty$  as in water then at  $x = b$  the moment is zero and

The rotation yields

The coefficient of integration is zero since the rotation slope is zero at  $x = 0$ . We can rewrite in Fourier sine series in the interval  $-b-t \leq x \leq b+t$  where the line  $C2x$  has an active slope with moment  $C2$  in  $-b \leq x \leq b$  and a reactive slope at the columns with a moment  $-C2 b/t$  so there is a finite rotation value at  $x = \pm b$  and the rotation at  $x = t$  is  $C2b/t \times t = C2b$  under the column and it is a saw tooth function. This is necessary for compatibility when the moment of inertia is the same for the beam and the column allowing downward deflection at lagging and upward deflection at columns such that the involuntary parabolic deflection of the moment  $C2$  in  $-b \leq x \leq b$  is met with a reverse parabolic deflection in  $-b - 2t \leq x \leq -b$  and  $b \leq x \leq b + 2t$  and so on giving a wavy deflection with compatibility slope at  $b = t$ . Also, when differentiating over the entire  $x$  domain we obtain active moment  $C2$  for arching and a reactive moment for the column  $-C2$  when  $b = t$  and have equilibrium. Part of the reason of doing the saw tooth because in equation 24 we restricted the deflection to be zero at the columns where in here we did not let it be controlled by the moment on inertia, see Figure 5, yields.

$$\iiint \sigma_y = \frac{1}{EI} \left[ \sum_{n=1}^{\infty} \frac{sv_n}{2\beta \alpha_n^2} \sin \alpha_n x + \frac{2p}{t} \sum_{n=1}^{\infty} \frac{\sin \alpha_n b}{\alpha_n^4} \sin \alpha_n x + \frac{2C2}{t} \sum_{n=1}^{\infty} \frac{\sin \alpha_n b}{\alpha_n^2} \sin \alpha_n x \right] \dots 46$$

In here  $I$  is the same in the region  $-b \leq x \leq b$  then of the region  $-b - 2t \leq x \leq -b$  and  $b \leq x \leq b + 2t$  the columns have large moment of inertia and the rotation and deflection is practically zero in that region when the deflection is derived it will be shown it is practically zero at the column when the rotation compatibility is not required. This is also true when  $t$  is infinite the slope is zero in that region under the column with  $1/\beta = 0$  or water. We leave the solution generic knowing the solution in each region is different and require substituting the proper moment of inertia. This solution will not affect the deflection at the lagging and does not change the moments, shear and stress since we require the moment to be zero at  $x = \pm b$ .

Finally, the deflection yields

C4 and  $v_0/2$  cancel and solve for  $v_n$  yields:

Or

This can happen if the deflection is not defined at  $x = \pm b$ . So, we set  $n$  to start from 1 and  $v_0$  is found from C4 when the deflection set equal zero at  $\pm b$ .

$\beta$  is in 1/KSI and  $p$  is Kip/inch thus  $v_0$  dimension is in inch. Substitute  $v_n$  in the moment

$$\int \int \sigma_y = - \sum_{n=1}^{\infty} \frac{s}{2\beta \alpha_n} \frac{\frac{2p \sin \alpha_n b}{t \alpha_n^5} + \frac{2C2 \sin \alpha_n b}{t \alpha_n^3}}{EI + \frac{s}{2\beta \alpha_n^3}} \cos \alpha_n x + \frac{2p}{t} \sum_{n=1}^{\infty} \frac{\sin \alpha_n b}{\alpha_n^3} \cos \alpha_n x + C2 \dots 52$$

Or

$$\iint \sigma_y = - \sum_{n=1}^{\infty} \frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^3} + \frac{2C2}{t} \frac{\sin \alpha_n b}{\alpha_n}}{\frac{2\beta EI}{S} \frac{\alpha_n^3}{a_n^3} + 1} \cos \alpha_n x + \frac{2p}{t} \sum_{n=1}^{\infty} \frac{\sin \alpha_n b}{\alpha_n^3} \cos \alpha_n x + C2 \dots \dots \dots 53$$

At  $x = b$  the moment is zero and solve for C2 and the closed form solution is obtained.

$$0 = - \sum_{n=1}^{\infty} \frac{\frac{2p \sin \alpha_n b}{t} \frac{\alpha_n^3}{\frac{2\beta EI a_n^3}{s} + 1}}{\cos \alpha_n b} + \frac{2p}{t} \sum_{n=1}^{\infty} \frac{\sin \alpha_n b}{\alpha_n^3} \cos \alpha_n b + C2$$

$$- C2 \sum_{n=1}^{\infty} \frac{\frac{2 \sin \alpha_n b}{t} \frac{\alpha_n}{\frac{2\beta EI a_n^3}{s} + 1}}{\cos \alpha_n b} ..... 54$$

$$C2 = \frac{- \sum_{n=1}^{\infty} \frac{\frac{2p \sin \alpha_n b}{t} \frac{\alpha_n^3}{\frac{2\beta EI a_n^3}{s} + 1}}{\cos \alpha_n b} + \frac{2p}{t} \sum_{n=1}^{\infty} \frac{\sin \alpha_n b}{\alpha_n^3} \cos \alpha_n b}{\frac{\frac{2 \sin \alpha_n b}{t} \frac{\alpha_n}{\frac{2\beta EI a_n^3}{s} + 1}}{\cos \alpha_n b} - 1} ..... 55$$

Or

$$C2 = \frac{\frac{\sum_{n=1}^{\infty} \frac{\frac{2\beta EI a_n^3}{s} \frac{2p \sin \alpha_n b}{t} \frac{\alpha_n^3}{\frac{2\beta EI a_n^3}{s} + 1}}{\cos \alpha_n b}}{\frac{2\beta EI a_n^3}{s} + 1}}{\frac{\sum_{n=1}^{\infty} \frac{\frac{2 \sin \alpha_n b}{t} \frac{\alpha_n}{\frac{2\beta EI a_n^3}{s} + 1}}{\cos \alpha_n b} - 1}{\frac{2 \sin \alpha_n b}{t} \frac{\alpha_n}{\frac{2\beta EI a_n^3}{s} + 1}}} = \frac{\frac{\sum_{n=1}^{\infty} \frac{\frac{2\beta EI}{s} \frac{2p}{t} \sin \alpha_n b}{\frac{2\beta EI a_n^3}{s} + 1}}{\cos \alpha_n b}}{\frac{\sum_{n=1}^{\infty} \frac{\frac{2 \sin \alpha_n b}{t} \frac{\alpha_n}{\frac{2\beta EI a_n^3}{s} + 1}}{\cos \alpha_n b} - 1}{\frac{2 \sin \alpha_n b}{t} \frac{\alpha_n}{\frac{2\beta EI a_n^3}{s} + 1}}} ..... 56$$

$$\frac{C2}{0.5pb^2} = \frac{\frac{\sum_{n=1}^{\infty} \frac{\frac{2\beta EI}{sb^3} \frac{4b}{t} \sin \alpha_n b}{\frac{2\beta EI b^3 a_n^3}{sb^3} + 1}}{\cos \alpha_n b}}{\frac{\sum_{n=1}^{\infty} \frac{\frac{2 \sin \alpha_n b}{t} \frac{\alpha_n}{\frac{2\beta EI b^3 a_n^3}{sb^3} + 1}}{\cos \alpha_n b} - 1}{\frac{2 \sin \alpha_n b}{t} \frac{\alpha_n}{\frac{2\beta EI b^3 a_n^3}{sb^3} + 1}}} = \frac{\frac{\sum_{n=1}^{\infty} \frac{\frac{4b}{t} \sin \alpha_n b}{b^3 a_n^3 + A}}{\cos \alpha_n b}}{\frac{A \sum_{n=1}^{\infty} \frac{\frac{2b \sin \alpha_n b}{t} \frac{\alpha_n}{b^3 a_n^3 + A}}{\cos \alpha_n b} - 1}{2b \sin \alpha_n b}} ..... 57$$

$$\frac{C2}{0.5pb^2} = \frac{\frac{\sum_{n=1}^{\infty} \frac{\frac{4}{\Gamma} \sin \alpha_n b}{(\pi n)^3 + A}}{\cos \alpha_n b}}{\frac{A \sum_{n=1}^{\infty} \frac{\frac{2(1+\Gamma) \sin \alpha_n b}{\Gamma} \frac{\pi n}{(\pi n)^3 + A}}{\cos \alpha_n b} - 1}{2(1+\Gamma) \frac{\sin \alpha_n b}{\pi n}}} = \frac{\frac{\sum_{n=1}^{\infty} \frac{4 \sin \alpha_n b}{(\pi n)^3 + A}}{\cos \alpha_n b}}{\frac{A \sum_{n=1}^{\infty} \frac{2(1+\Gamma) \frac{\sin \alpha_n b}{\pi n}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} - \Gamma}{2(1+\Gamma) \frac{\sin \alpha_n b}{\pi n}}} - \Gamma$$

Finally, with  $\Gamma = t/b$ :

The following solution is in the region  $-b \leq x \leq b$

The Moment becomes at  $x = 0$ :

Substitute  $v_n$  in the shear we have,

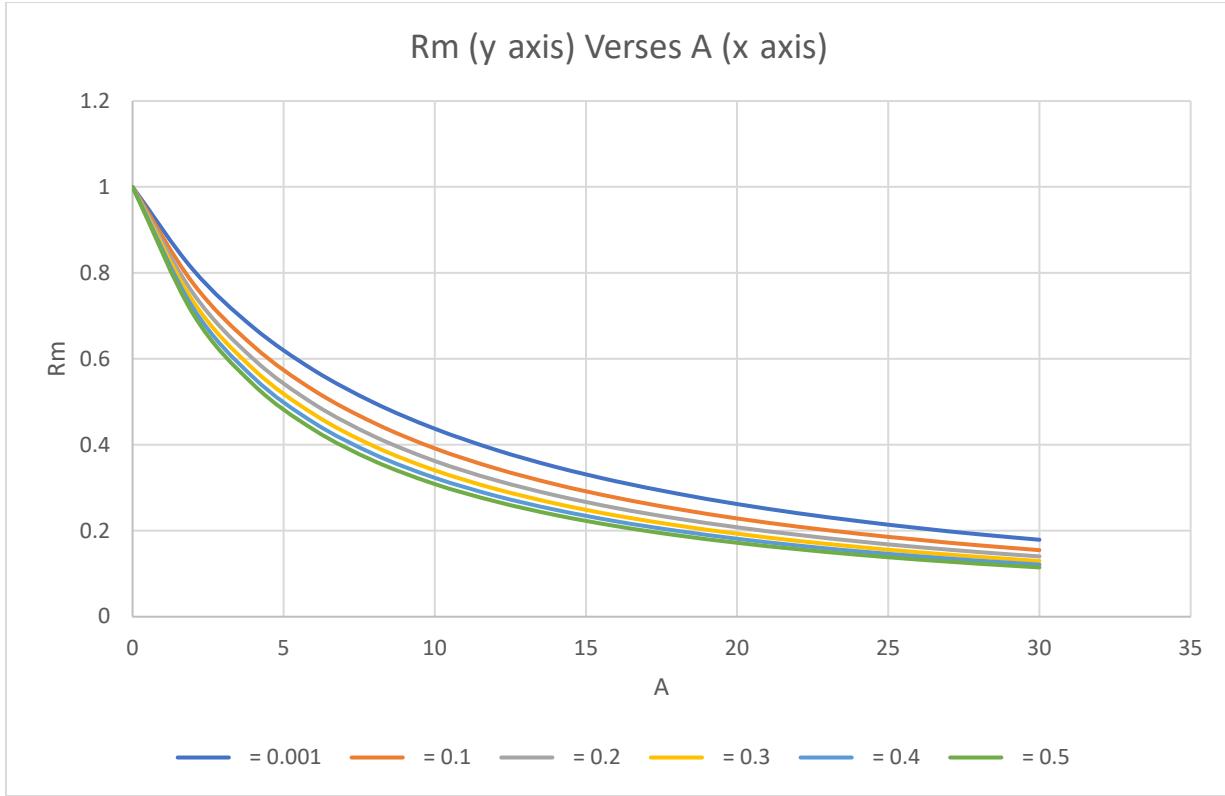
The Moment reduction becomes

Finally, the Moment reduction at  $x = 0$  becomes

Note: the maximum moment may not be at  $x = 0$  depending on  $A$

### Rm Table 1 for different t/b

A	= 0.001	= 0.1	= 0.2	= 0.3	= 0.4	= 0.5
0	1	1	1	1	1	1
2	0.807309	0.775713	0.752687	0.734192	0.718501	0.704682
4	0.672642	0.629316	0.599158	0.575732	0.556381	0.539728
6	0.573373	0.526407	0.494745	0.470694	0.45117	0.434613
8	0.49728	0.450234	0.41925	0.396083	0.3775	0.361901
10	0.437181	0.391665	0.36221	0.340437	0.323123	0.308697
12	0.388581	0.345297	0.317659	0.297405	0.281402	0.268142
14	0.348521	0.307729	0.281952	0.263184	0.248426	0.236251
16	0.314975	0.276715	0.252731	0.235355	0.221743	0.210551
18	0.286509	0.250709	0.228407	0.212311	0.199737	0.189426
20	0.26208	0.228617	0.20787	0.192939	0.1813	0.171777
22	0.240909	0.209639	0.190319	0.176445	0.165647	0.156827
24	0.222407	0.193178	0.175164	0.162248	0.152207	0.144017
26	0.206116	0.17878	0.16196	0.149911	0.140553	0.132928
28	0.191678	0.166093	0.150364	0.139103	0.130361	0.123246
30	0.178805	0.15484	0.140109	0.129565	0.121382	0.114727



Substitute  $v_n$  in the shear

The shear reduction becomes at  $x = b$ :

$$\frac{V}{-pb} = Rv = - \sum_{n=1}^{\infty} \frac{\frac{2b \sin \alpha_n b}{t} + \frac{2bC2}{tpb^2} \sin \alpha_n b}{\frac{2\beta EI b^3 a_n^3}{sb^3} + 1} \sin \alpha_n b + \frac{2b}{t} \sum_{n=1}^{\infty} \frac{\sin \alpha_n b}{\alpha_n^2 b^2} \sin \alpha_n b \dots \dots \dots 66$$

$$\frac{V}{-pb} = R\nu = - \sum_{n=1}^{\infty} \frac{\frac{2b \sin \alpha_n b}{t \alpha_n^2 b^2} + \frac{2bC2}{tpb^2} \sin \alpha_n b}{\frac{2\beta EI b^3 \alpha_n^3}{sb^3} + 1} \sin \alpha_n b + \frac{2b}{t} \sum_{n=1}^{\infty} \frac{\sin \alpha_n b}{\alpha_n^2 b^2} \sin \alpha_n b \dots \dots \dots 67$$

$$Rv = -A \sum_{n=1}^{\infty} \frac{\frac{2}{\Gamma}(1+\Gamma)^2 \frac{\sin \alpha_n b}{(\pi n)^2} + \frac{2C2}{\Gamma p b^2} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} \sin \alpha_n b + \frac{2}{\Gamma}(1+\Gamma)^2 \sum_{n=1}^{\infty} \frac{\sin \alpha_n b}{(\pi n)^2} \sin \alpha_n b$$

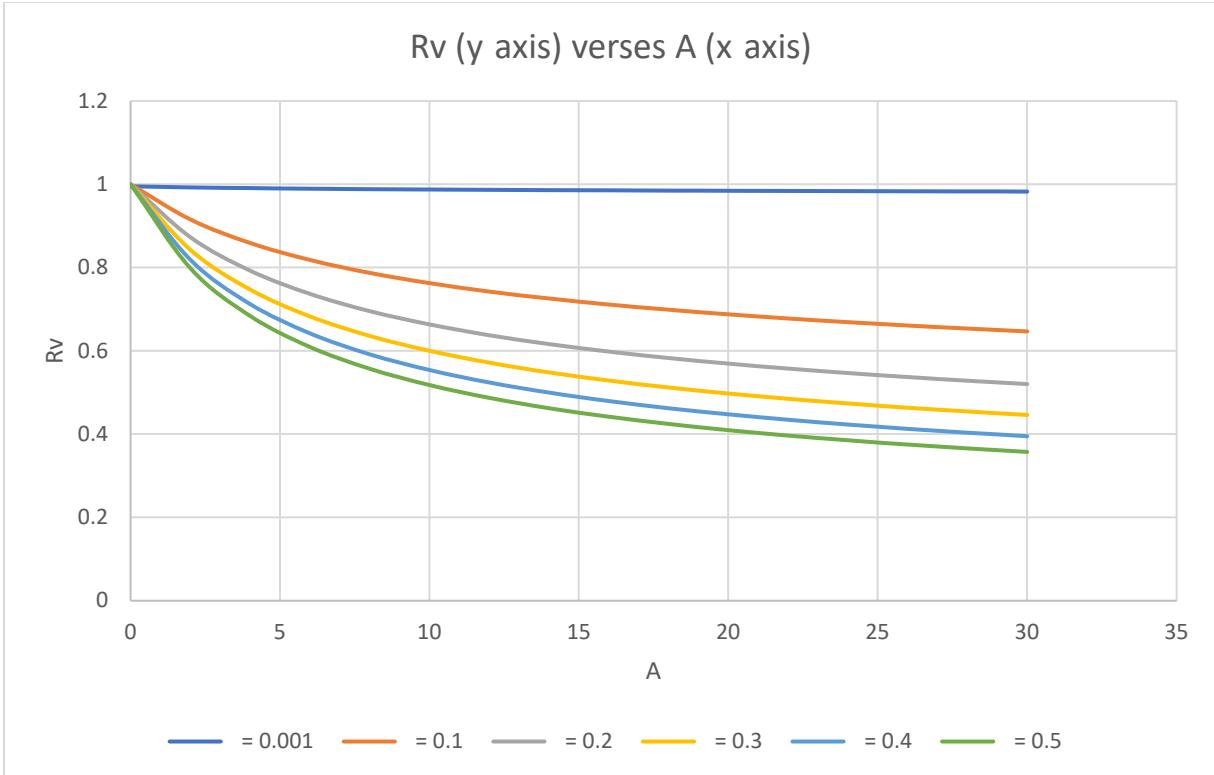
$$\begin{aligned} Rv &= -A \sum_{n=1}^{\infty} \frac{\frac{2}{\Gamma}(1+\Gamma)^2 \frac{\sin \alpha_n b}{(\pi n)^2} + \frac{2C2}{\Gamma p b^2} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} \sin \alpha_n b \\ &\quad + \frac{2A}{\Gamma}(1+\Gamma)^2 \frac{\sin \alpha_n b}{(\pi n)^2} \sin \alpha_n b \frac{1}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} \\ &\quad + \frac{2}{\Gamma}(1+\Gamma)^2 \frac{(\pi n)^3}{(1+\Gamma)^3} \frac{\sin \alpha_n b}{(\pi n)^2} \sin \alpha_n b \frac{1}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} \dots \dots \dots 68 \end{aligned}$$

Finally, the shear reduction at  $x = b^*$  becomes

$$Rv = \frac{1}{\Gamma} \sum_{n=1}^{\infty} \frac{2 \frac{\pi n}{(1+\Gamma)} - A \frac{C2}{0.5 p b^2}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} \sin \alpha_n b \sin \alpha_n b^* \dots \dots \dots \dots \dots \dots 69$$

**Rv Table 2 for different t/b b=b\***

A	= 0.001	= 0.1	= 0.2	= 0.3	= 0.4	= 0.5
0	0.9949295	0.9999387	0.9999635	0.9999715	0.9999752	0.9999772
2	0.9924688	0.9153313	0.8729995	0.8426343	0.8185008	0.7981535
4	0.9906877	0.8588505	0.7926728	0.7472143	0.7122938	0.68369
6	0.9893228	0.8181177	0.7368287	0.6826747	0.6420409	0.6094085
8	0.9882319	0.787103	0.6954463	0.6357757	0.5917738	0.5569488
10	0.9873312	0.7625172	0.6633309	0.5999157	0.5537789	0.5176781
12	0.9865682	0.7424127	0.6375201	0.5714337	0.5238727	0.4870007
14	0.9859085	0.7255623	0.6162	0.5481348	0.4995886	0.4622441
16	0.9853283	0.7111541	0.5981976	0.5286231	0.4793775	0.4417476
18	0.9848108	0.6986291	0.5827204	0.5119674	0.462217	0.4244241
20	0.9843438	0.6875899	0.5692132	0.4975229	0.4474048	0.4095321
22	0.983918	0.6777457	0.5572753	0.4848283	0.434442	0.3965477
24	0.9835265	0.668879	0.5466101	0.4735448	0.4229646	0.3850901
26	0.9831639	0.6608234	0.5369933	0.463418	0.4127001	0.374876
28	0.9828259	0.6534498	0.5282518	0.4542525	0.4034407	0.3656896
30	0.982509	0.6466561	0.52025	0.4458963	0.3950249	0.3573639



Again, when  $I$  is very large or infinite, we get the saw tooth shear and we have equilibrium. The stress is

Again, when  $I$  is very large or infinite we get the step stress  $-p$  and  $pb/t$  necessary stress for equilibrium on the lagging and columns. At  $x = 0$  the stress is

Or

$$\frac{\sigma_y}{p} \text{ (at } y = 0 \text{ and } x = 0) = \frac{sft}{p} = \frac{A}{\Gamma} \sum_{n=1}^{\infty} \frac{2 \frac{(1+\Gamma)}{\pi n} + \frac{\pi n}{(1+\Gamma)} \frac{C2}{0.5pb^2}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} \sin \alpha_n b - 1 \dots \dots 72$$

Where  $ft$  is the tension in the center of lagging. When  $A$  is large, we rewrite from the shear:

Or when the solid is rigid the stress is zero. The stress becomes tension for some  $A$  but when  $A$  is large there is zero stress. Thus  $sft/p$  has a maximum positive tension value see table below. As when  $A = 0$  its water.

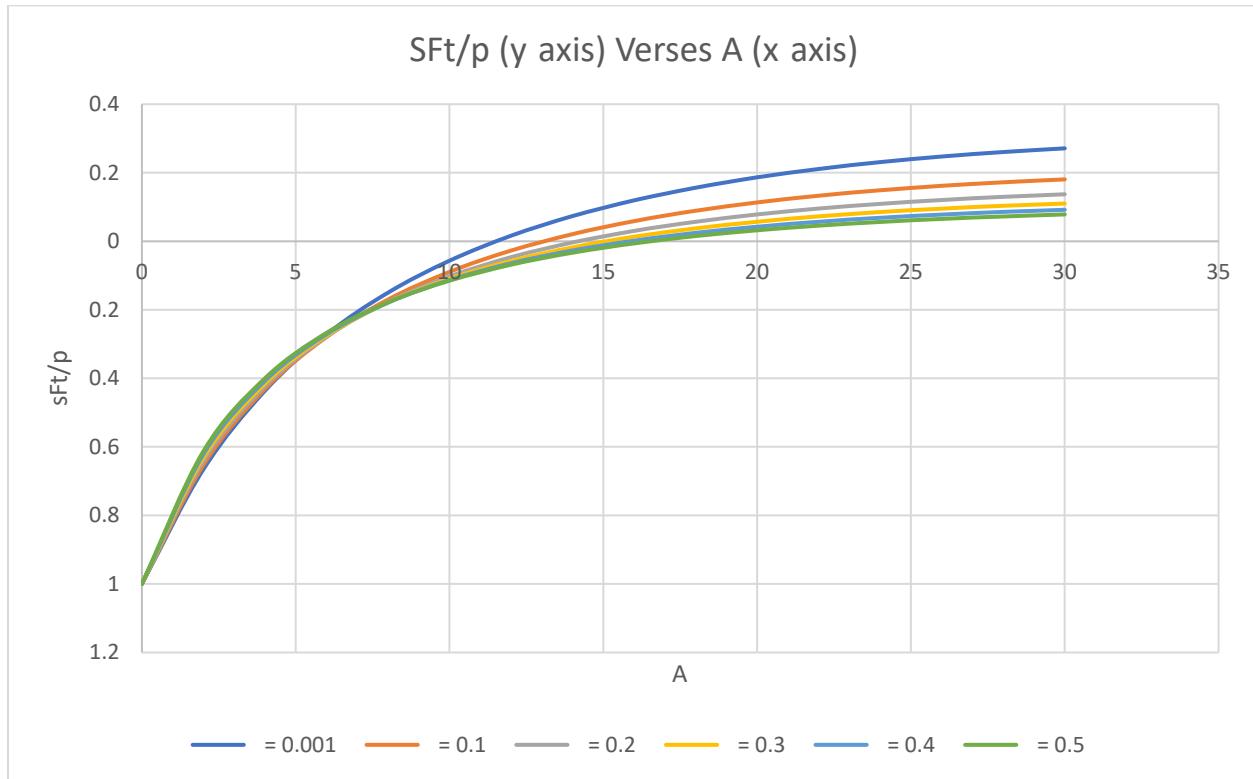
t/b	Amax	sft/p max	Rm	Rv
0.001	47.2	0.303748	0.108182	0.980359
0.1	47.85	0.206469	0.092543	0.602219
0.2	48.6	0.159955	0.082332	0.467557
0.3	49.35	0.131517	0.074818	0.390102
0.4	50.05	0.111985	0.068959	0.338085
0.5	50.85	0.097733	0.064001	0.300147

For the condition where  $sft/p$  is zero for soils see table below.

t/b	Amax	Stress	Rm	Rv
0.001	11.515	6.21E 05	0.399488	0.986743
0.1	13.07	4.6E 05	0.324267	0.733052
0.2	14.18	5.6E 05	0.279084	0.614458
0.3	15.08	9.59E 07	0.247479	0.537196
0.4	15.85	6.26E 05	0.223566	0.480777
0.5	16.53	9.06E 05	0.20456	0.436883

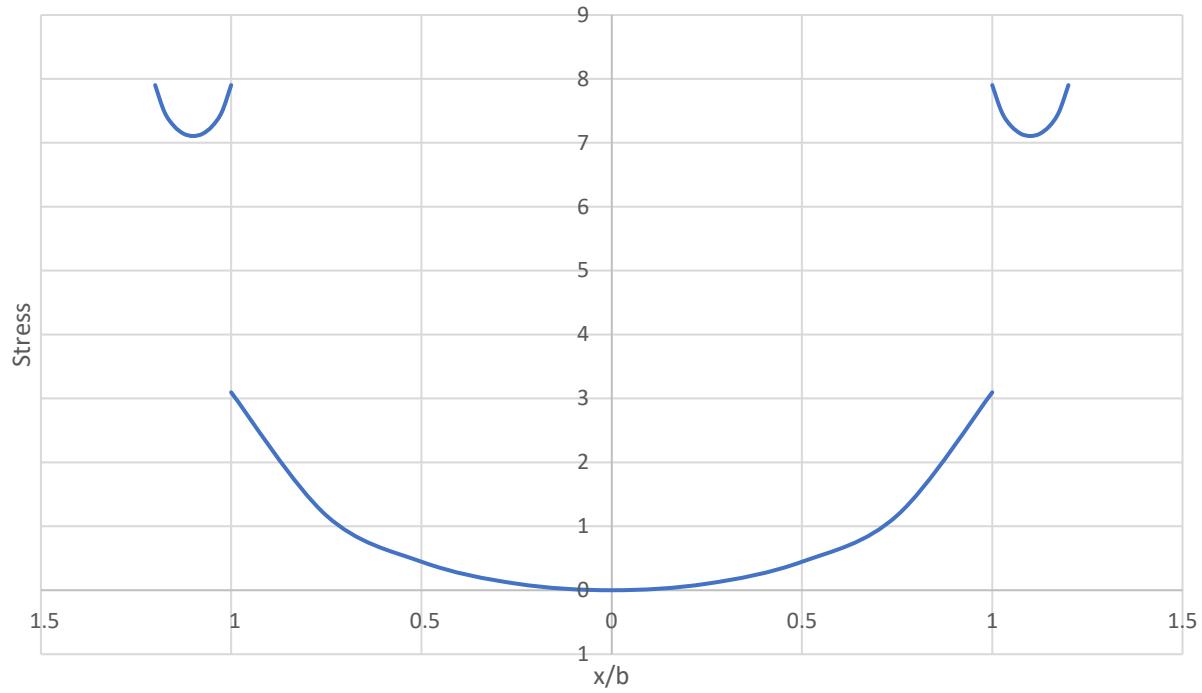
**sFt/p Table 3 for different t/b**

A	= 0.001	= 0.1	= 0.2	= 0.3	= 0.4	= 0.5
0	1	1	1	1	1	1
2	0.665	0.65291	0.64134	0.63083	0.62123	0.61234
4	0.43619	0.43123	0.42323	0.41506	0.40722	0.39978
6	0.27194	0.27935	0.27852	0.27542	0.27154	0.26739
8	0.14978	0.1702	0.17685	0.17896	0.1791	0.17825
10	0.05652	0.08904	0.10249	0.10929	0.11299	0.11501
12	0.016105	0.02714	0.04651	0.05732	0.06403	0.06845
14	0.073516	0.020962	0.00343	0.01761	0.02683	0.03324
16	0.119423	0.058891	0.030265	0.013281	0.001987	0.00606
18	0.156446	0.089122	0.056953	0.037639	0.024634	0.015253
20	0.186489	0.113411	0.078286	0.057045	0.042633	0.032159
22	0.210965	0.133033	0.095453	0.072622	0.057054	0.04569
24	0.230946	0.148938	0.109327	0.085189	0.068676	0.056587
26	0.247257	0.161844	0.120564	0.095357	0.078074	0.0654
28	0.260544	0.172309	0.129665	0.103591	0.085685	0.072541
30	0.271321	0.180764	0.137019	0.110248	0.091845	0.078328

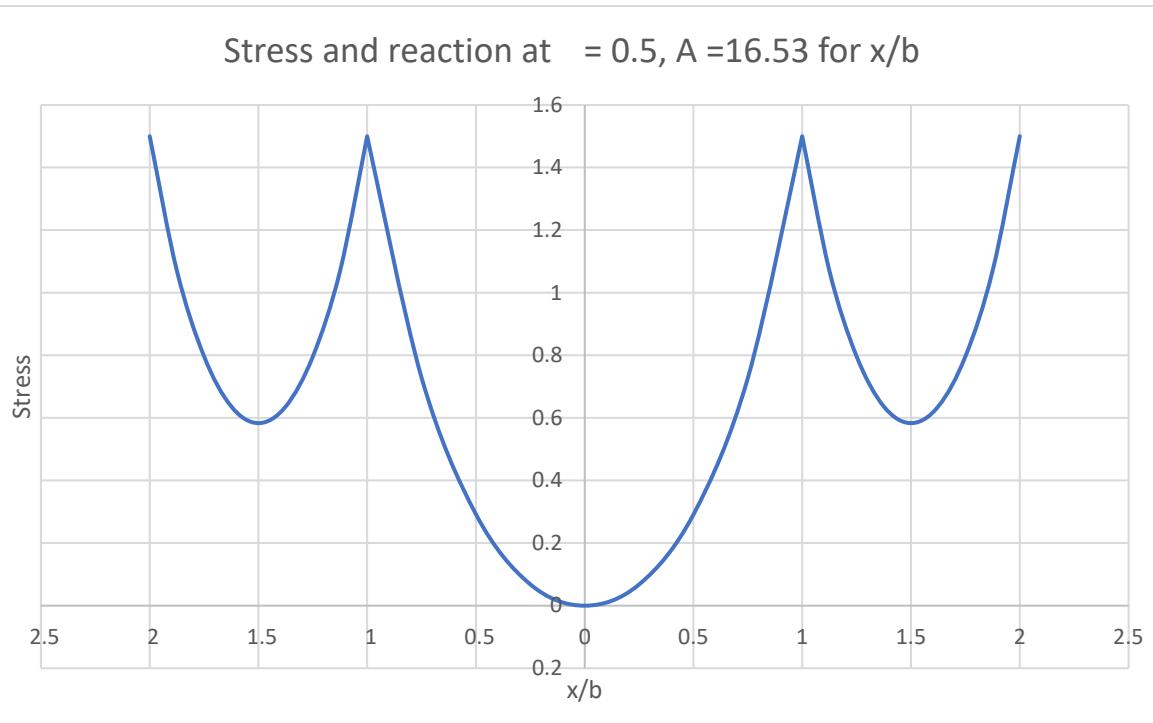


In the following graph the pressure on the lagging is shown positive to see how the distribution is for various  $x/b$  and the reaction is shown.

Stress and reaction at  $\gamma = 0.1$ ,  $A = 13.07$  for  $x/b$



Stress and reaction at  $\gamma = 0.5$ ,  $A = 16.53$  for  $x/b$



We seek to find the deflection when starting from  $n = 1$  we have

$$v_n = -\frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^5} + \frac{2C2}{t} \frac{\sin \alpha_n b}{\alpha_n^3}}{EI + \frac{s}{2\beta \alpha_n^3}}$$

$$\sum_{n=1}^{\infty} v_n \cos \alpha_n x = -\sum_{n=1}^{\infty} \frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^5} + \frac{2C2}{t} \frac{\sin \alpha_n b}{\alpha_n^3}}{EI + \frac{s}{2\beta \alpha_n^3}} \cos \alpha_n x$$

$$\frac{EI v(x)}{\frac{5pb^4}{24}} = -\frac{EI}{\frac{5pb^4}{24}} \sum_{n=1}^{\infty} \frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^5} + \frac{2C2}{t} \frac{\sin \alpha_n b}{\alpha_n^3}}{EI + \frac{s}{2\beta \alpha_n^3}} \cos \alpha_n x$$

$$\frac{EI v(x)}{\frac{5pb^4}{24}} = -\frac{1}{\frac{5pb}{24}} \sum_{n=1}^{\infty} \frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^2} + \frac{2C2}{t} \sin \alpha_n b}{b^3 \alpha_n^3 + \frac{sb^3}{2\beta EI}} \cos \alpha_n x$$

$$\frac{EI v(x)}{\frac{5pb^4}{24}} = -\frac{1}{\frac{5}{24}} \sum_{n=1}^{\infty} \frac{\frac{2}{tb} \frac{\sin \alpha_n b}{\alpha_n^2} + \frac{2C2}{tbp} \sin \alpha_n b}{b^3 \alpha_n^3 + A} \cos \alpha_n x$$

$$\frac{EI v(x)}{\frac{5pb^4}{24}} = -\frac{1}{\frac{5\Gamma}{24}} \sum_{n=1}^{\infty} \frac{2(1+\Gamma)^2 \frac{\sin \alpha_n b}{(\pi n)^2} + \frac{C2}{0.5pb^2} \sin \alpha_n b}{(\pi n)^3 + A} \cos \alpha_n x \dots \dots \dots 75$$

Also, the deflection should go more negative to account for the deflection of the column which is  $PL/AE$ . Where  $P$  is the reaction equal to  $-2pb - 2pt$ . Also, we have the requirement to have the relative deflection to be zero at  $x = \pm b$  to calculate the constant of integration C4. Rewrite the equations with these criteria: We have the deflection:

$$\frac{EI v(x)}{-\frac{5pb^4}{24}} = \frac{1}{\frac{5\Gamma}{24}} \sum_{n=1}^{\infty} \frac{2 \frac{(1+\Gamma)^2}{(\pi n)^2} + \frac{C2}{0.5pb^2}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} (\cos \alpha_n x - \cos \alpha_n b) \sin \alpha_n b \dots \dots \dots 76$$

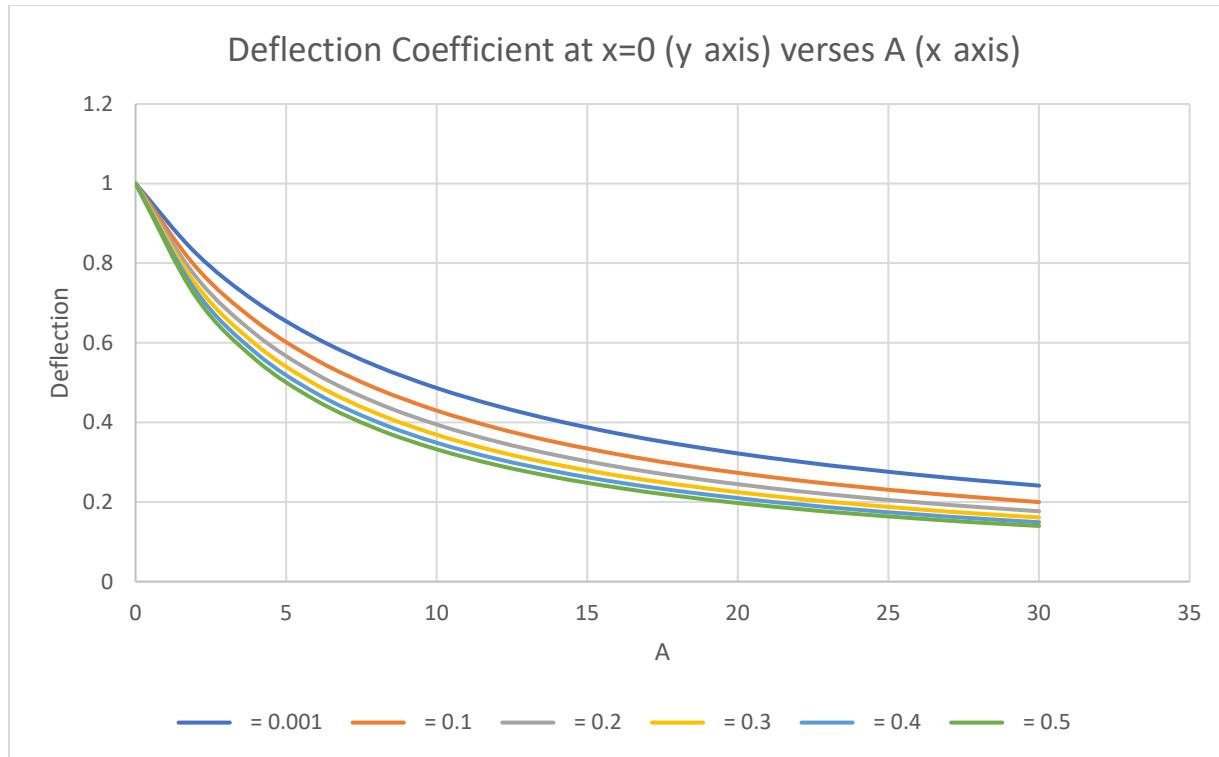
To verify the constant coefficient of the Fourier series we have:

$$\begin{aligned}
\frac{v_0}{2} &= \frac{1}{b+t} \int_0^{b+t} v(x) dx = -\frac{5pb^4}{24EI} \frac{1}{b+t} \sum_{n=1}^{\infty} \frac{2 \frac{(1+\Gamma)^2}{(\pi n)^2} + \frac{C2}{0.5pb^2}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} (- (b+t) \cos \alpha_n b) \sin \alpha_n b \\
&= -\frac{5pb^4}{24EI} \sum_{n=1}^{\infty} \frac{2 \frac{(1+\Gamma)^2}{(\pi n)^2} + \frac{C2}{0.5pb^2}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} (- \cos \alpha_n b) \sin \alpha_n b
\end{aligned}$$

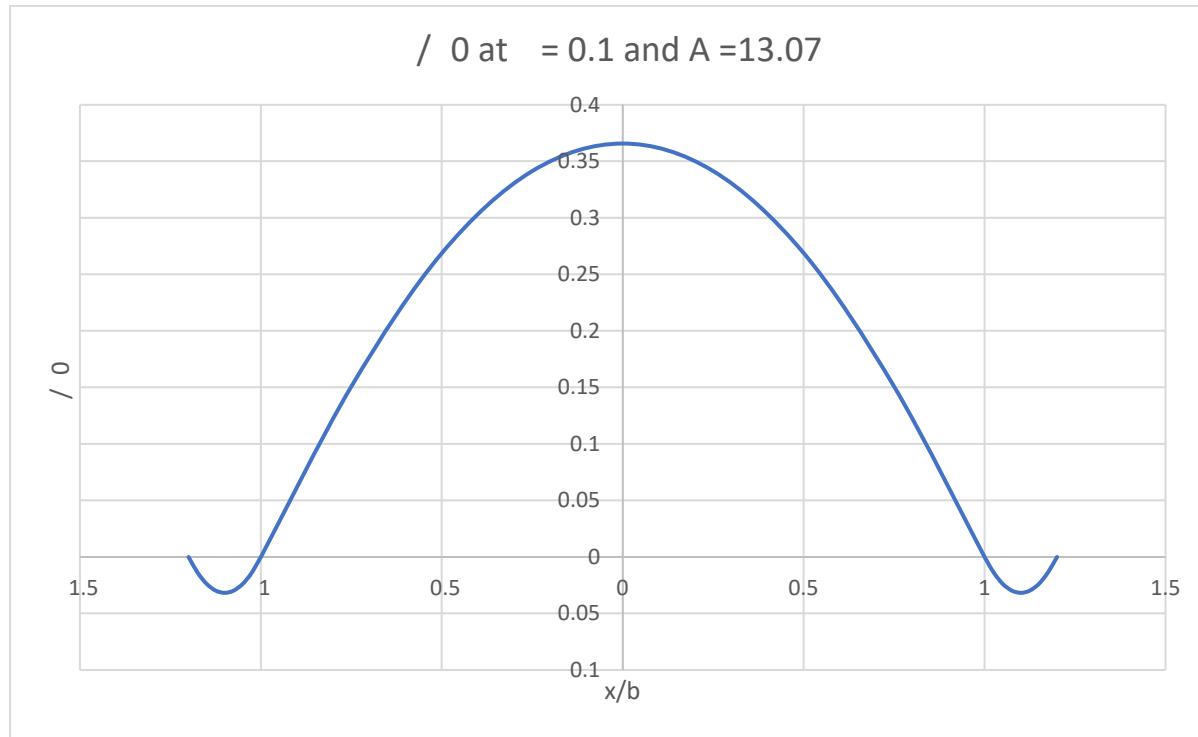
And it is verified and is defined not infinite.

**Deflection coefficient Table 4 for different A at x=0**

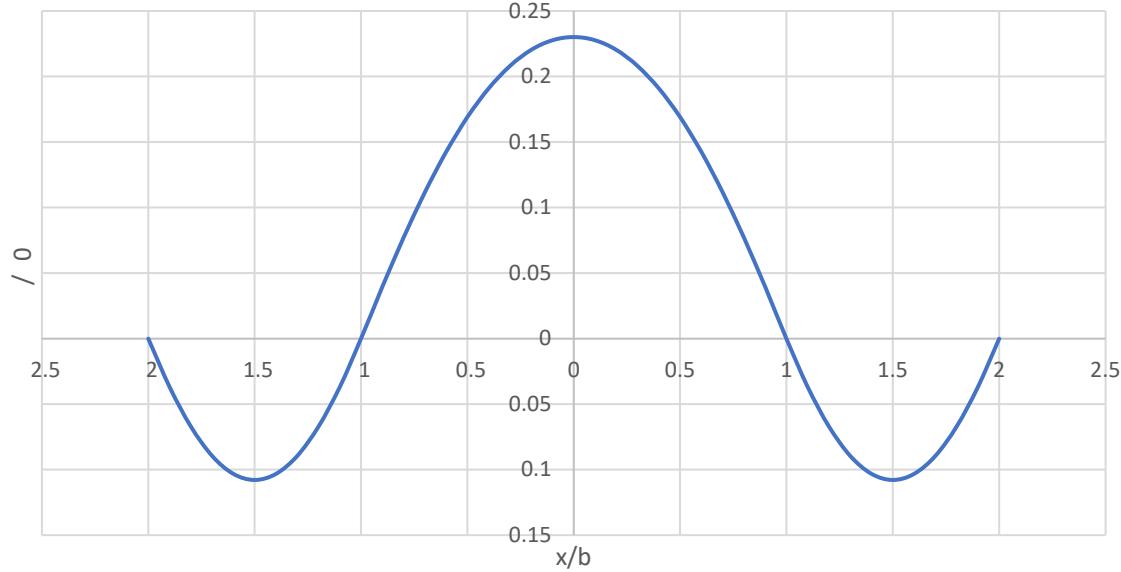
A	= 0.001	= 0.1	= 0.2	= 0.3	= 0.4	= 0.5
0	1	1	1	1	1	1
2	0.8252038	0.7908272	0.7662538	0.7467072	0.7302261	0.715774
4	0.7025856	0.653851	0.6207229	0.5952991	0.5744631	0.556631
6	0.6118157	0.5572023	0.5214101	0.4946146	0.4730689	0.454923
8	0.5419117	0.4853627	0.449325	0.4228341	0.4018235	0.384331
10	0.4864201	0.4298704	0.3946272	0.3690826	0.349032	0.332484
12	0.4413	0.3857187	0.351708	0.3273322	0.308355	0.2928
14	0.4038919	0.3497559	0.3171366	0.2939724	0.2760578	0.261457
16	0.372374	0.3198991	0.2886964	0.2667087	0.2497977	0.236081
18	0.3454565	0.2947164	0.2648917	0.244013	0.2280303	0.21512
20	0.3222005	0.2731911	0.2446764	0.2248286	0.2096965	0.197518
22	0.3019064	0.2545813	0.2272973	0.2084011	0.1940457	0.18253
24	0.284042	0.2383332	0.2121978	0.1941777	0.180531	0.169616
26	0.2681955	0.2240246	0.1989581	0.181744	0.1687446	0.158375
28	0.2540431	0.2113284	0.1872555	0.1707833	0.1583762	0.148504
30	0.241327	0.1999871	0.1768377	0.1610495	0.1491856	0.139768



In the following graph the deflection coefficient on the lagging is shown positive at the lagging to see how the distribution is for various  $x/b$ . We see the deflection at the column is practically zero for  $t/b = 0.1$ .



/ 0 at = 0.5 and A = 16.53



Note: in the above chart the deflection at the column has a much larger moment of inertia so when multiplying by  $-\frac{5pb^4}{24EI}$  the deflection is practically zero and match the previous article the compatibility of rotation is not required. The deflection at the center of lagging is shown at  $x = 0$ . However, if we define zero deflection at  $x = \pm(b + t)$  instead of  $\pm b$  and offset the chart, we see that the corners of the column deflect downward because of bearing. Allowing deflection at the column theoretically gives less arching since the more we have a ridged column, as zero deflection, the load will go to the stiffer column and reduce the load on the lagging or more arching. So, the presented solution in this article is more realistic. However, the deflection does not take into account the moment of inertia of the elastic media, so it is conservative. But for a media such as soils where tension is not allowed it is accurate and useful.

Finally, the rotation equation is:

$$\frac{EI v'(x)}{\frac{pb^3}{3}} = \frac{3}{\Gamma} \sum_{n=1}^{\infty} \frac{2(1+\Gamma) \frac{\sin \alpha_n b}{\pi n} + \frac{C2}{0.5pb^2} \frac{\pi n}{(1+\Gamma)} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} \sin \alpha_n x \dots \dots .77$$

Finn [3] shows that the presence of friction between soil and wall requires a minor change in the stress function, due to  $\varepsilon_x = 0$ , which leads to

Solving for  $D$  shows that the solution has the same form as in the frictionless case but  $1/(2\beta)$  is replaced by  $2\beta/[(3\beta - \rho)(\beta + \rho)]$ . Thus, may be used, if  $A$  is calculated as

The conditions here are slightly different from those addressed by Finn [3] since  $\varepsilon_x$  is not quite zero. Thus, is not strictly valid, however it acts as an upper bound on  $A$ , and so is useful. We can consider the lagging or the beam to be glued to the mass media such that tension will act upward on the beam in the plain stress problem. Thus, in plain stress we have:

For a rough surface we have:

For a smooth surface we have:

In plain strain we have:

For a rough surface we have:

For a smooth surface we have:

Once  $D$  is found the stresses at  $q = 0$  and  $h = \infty$  are:

$$D = -\frac{s\alpha_n v_n}{\beta d_n} = -\frac{\alpha_n p s b^4}{2\beta EI\Gamma} \left[ \frac{2(1+\Gamma)^2 \frac{\sin \alpha_n b}{(\pi n)^2} + \frac{C2}{0.5pb^2} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} \right]$$

$$D = -\frac{s\alpha_n v_n}{\beta d_n} = p \frac{sb^3}{\beta EI d_n \Gamma} \frac{\pi n}{1 + \Gamma} \left[ \frac{2(1 + \Gamma)^2 \frac{\sin \alpha_n b}{(\pi n)^2} + \frac{C2}{0.5pb^2} \sin \alpha_n b}{\frac{(\pi n)^3}{(1 + \Gamma)^3} + A} \right]$$

$$d_n = -2$$

$$\tau_{xy} = - \sum_{n=\varphi}^{\infty} \alpha_n Dye^{-\alpha_n y} \sin \alpha_n x$$

### Example 1:

We have a 4 ft x 4 ft opening to be install in a dam powerhouse for a mechanical duct. The wall is concrete 30 ft high and 3 ft thick. We can assume that qualify for an infinite height and can use the above solution with plain stress. Note at  $y = 0$  we have  $\sigma_y = \sigma_x/k_0$

	b =	2 ft	4 ft opening
	t =	1 ft	s = 3 ft
Concrete	150 pcf	at the powerhouse at the dam	
p =	13500 lbs/ft	30 ft height x 3 ft thick	
p =	1125 lbs/in	Pick t/b = 0.5	
p / inch =	1125 psi	v =	0.2
Ft =	87.63561 psi	We use an old code Ft = 1.6	3000
sFt/p/ =	0.389492 > 0.09773	F'c =	3000 psi
Use sft/p max		Allowable Shear = 0.02f'c =	60

For a rough surface we have:

$$A = \frac{4}{(3-\nu)(1+\nu)} \left[ \frac{sb^3 E_s}{2EI} \right]$$

$E_s = E$

Thus	IA =	14.28571	
A =	50.85	for sFt/p = 0.0977333	
thus I =	0.2809383	ft^4	
thus I =	5825.5373	in^4 = sh^3/12	
$h^3 =$	1941.8458		$h = 12.47589$ in
Sx =	933.8872	in^3 A =	449.1321 in^2
Mu = $1.3pb^2/2 =$	35100	Lbs ft	Ultimate Moment
Mu reduced =	2965.827	k ft at x = 0.615b	Rm = 0.084496
ft = M/Sx	38.10944	< Ft =	87.63561 psi
Rv =	0.025824	at x/b = 0.3787	Max shear
1.3fv =	3.03	< Fv =	60 psi
Ec = Es = E = 57000 3000 =			3122019 psi
$O = 5*p*b^4/24EI =$		0.00428	inch
/ O =	0.0861896	Thus =	0.00037 inch
Thus, the relative deflection at the center is = 0.00037 inch			

For a smooth surface we have:

$$A = \frac{sb^3 E_s}{2EI}$$

$E_s = E$

Thus	IA =	12	
A =	50.85	for sFt/p = 0.0977333	
thus I =	0.2359882	ft^4	
thus I =	4893.4513	in^4 = sh^3/12	
$h^3 =$	1631.1504		$h = 11.7715$ in
Sx =	831.4075	in^3 A =	423.774 in^2
Mu = $1.3pb^2/2 =$	35100	Lbs ft	Ultimate Moment
Mu reduced =	2965.827	k ft at x=0.615b	Rm = 0.084496
ft = M/Sx	42.807	< Ft =	87.63561 psi
Rv =	0.025824	at x/b = 0.3787	Max shear
1.3fv =	3.21	< Fv =	60 psi
Ec = Es = E = 57000 3000 =			3122019 psi
$O = 5*p*b^4/24EI =$		0.00509	inch
/ O =	0.0861896	Thus =	0.000439 inch
Thus, the relative deflection at the center is = 0.000439 inch			

The deflection does not take into account there is 30 ft above with high moment of inertia. So, this deflection is not actual.

Now,  $R_m$  and  $R_v$  was found for  $A = 50.85$  from the tables. So, we can see that the opening can be installed. The infinite height used as an assumption should be reduced by 12.476 inch which does not change the solution.

Example 1A:

A 15 ft cut permanent shoring retaining soils and it is desirable to use recycled plastic for lagging. Reference: [https://american-plasticlumber.com/wp-content/uploads/2016/03/apl\\_web\\_str\\_ti\\_hdpe\\_tech\\_data.pdf](https://american-plasticlumber.com/wp-content/uploads/2016/03/apl_web_str_ti_hdpe_tech_data.pdf)

The piles are 8 ft O.C. and the flange is 6-inch-wide on the steel pile. Thus, we have:

$$A = \frac{sb^3 E_s}{2EI} \quad \text{Smooth surface}$$

b =	45	in	8 ft spacing	6 inch flange
t =	3	in	s =	3.50 in 4x4 plastic lagging
Ep =	221260	psi		
=	0.3			
1/B = Es / (1 ^2) =	1221	psi	Es = 1111 psi	
I =	12.51	in^4		
A =	70.37			
=	0.067			
For = 0.067 Amax =	12.62	sFt/p =	3.97E 05	
Thus, cannot use A = 70.37 it produces tension in soil				
The allowable reduction is:	Rm =	0.344581	at x/b = 0	
	Rv =	0.573092	at x/b =	0.922222
15 ft Shoring Ka = 0.5, = 120 pcf p = 0.9 ksf				
p at s = 3.5 in or 0.2917 ft = 0.2625 ksf per s				
Vmax =	0.564138	kip		
Mmax =	0.635994	ft kip		
Sx =	7.145833	in^3		
M/Sx =	1068.024	psi < 2114 psi OK		
Assume shear as wood				
Area of member = 12.25 in^2				
= 1.5 Vmax / Area = 69.07806 psi < 95 psi OK				

**Thus, 4x4 Plastic Lagging Is OK**

### Closed form solution #2 fixed end span lagging with $h = \infty$ :

The equations remain the same as the previous solution only the constant C<sub>2</sub> is different. In the previous solution C<sub>2</sub> could not be define at  $x = b$  in a Fourier series because it was a step function and C<sub>2</sub> was done carefully. In the fixed solution C<sub>2</sub> can be found by setting the rotations to be zero at  $x = \pm b$  and the rotation at that point is not a step and can be found by setting rotation equation 71 to be zero at  $x = \pm b$ . Thus:

Or

The positive moment at  $x = 0$  is

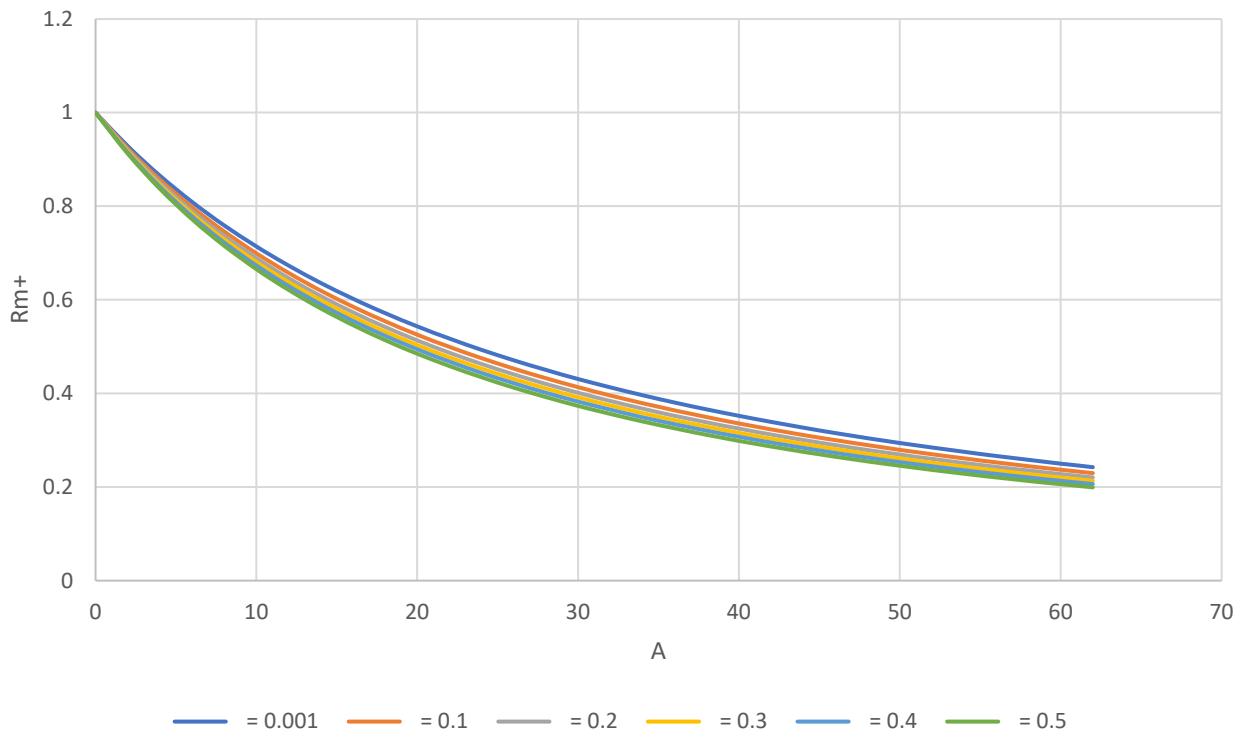
Note: the maximum positive moment may not be at  $x = 0$  depending on  $A$ .

$$\frac{M +}{pb^2} = Rm+ = 3 \left[ \frac{1}{\Gamma} \sum_{n=1}^{\infty} \frac{4 \sin \alpha_n b - 2A \left( \frac{C2}{0.5pb^2} \right) \frac{(1+\Gamma)}{\pi n} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} + \frac{C2}{0.5pb^2} \right] \dots \dots \dots 91$$

**Rm+ Table 5 for different t/b**

A	= 0.001	= 0.1	= 0.2	= 0.3	= 0.4	= 0.5
0	0.99999	0.999988	0.999985	0.999983	0.99998	0.999977
2	0.928426	0.92366	0.92021	0.917441	0.91492	0.912341
4	0.865258	0.856911	0.850908	0.846102	0.841721	0.837238
6	0.809117	0.798075	0.790179	0.783867	0.778106	0.772207
8	0.758917	0.745851	0.736554	0.72913	0.722343	0.71539
10	0.713784	0.699208	0.68888	0.680639	0.673093	0.665358
12	0.673006	0.657316	0.646241	0.637407	0.629304	0.620994
14	0.635999	0.619503	0.607899	0.598643	0.590137	0.581412
16	0.602278	0.585218	0.573251	0.563705	0.554918	0.545901
18	0.571437	0.554002	0.541804	0.532071	0.523095	0.513883
20	0.543133	0.525474	0.513147	0.503306	0.494215	0.484884
22	0.517077	0.499312	0.486936	0.47705	0.467901	0.458511
24	0.493021	0.475244	0.462881	0.452999	0.443837	0.434435
26	0.470752	0.453037	0.440737	0.430897	0.421758	0.412382
28	0.450086	0.432492	0.420292	0.410524	0.401437	0.392116
30	0.430862	0.413437	0.401367	0.391694	0.382681	0.373439
32	0.412942	0.395721	0.383804	0.374245	0.365323	0.356179
34	0.396203	0.379214	0.367469	0.358037	0.34922	0.340188
36	0.380537	0.363803	0.352242	0.342948	0.334246	0.325338
38	0.36585	0.349387	0.33802	0.328871	0.320293	0.311518
40	0.352058	0.335877	0.324711	0.315713	0.307265	0.298628
42	0.339084	0.323195	0.312234	0.303391	0.295077	0.286584
44	0.326863	0.311271	0.300518	0.291832	0.283656	0.27531
46	0.315334	0.300042	0.289498	0.280972	0.272935	0.264737
48	0.304443	0.289454	0.279119	0.270751	0.262854	0.254807
50	0.294142	0.279455	0.269328	0.261119	0.253363	0.245466
52	0.284387	0.27	0.260081	0.252029	0.244413	0.236666
54	0.275139	0.26105	0.251335	0.243439	0.235963	0.228365
56	0.266361	0.252567	0.243053	0.235312	0.227974	0.220524
58	0.258021	0.244518	0.235203	0.227613	0.220412	0.213108
60	0.250089	0.236873	0.227752	0.220312	0.213246	0.206086
62	0.242538	0.229603	0.220674	0.21338	0.206447	0.19943

Rm+ Table for different  $t/b$



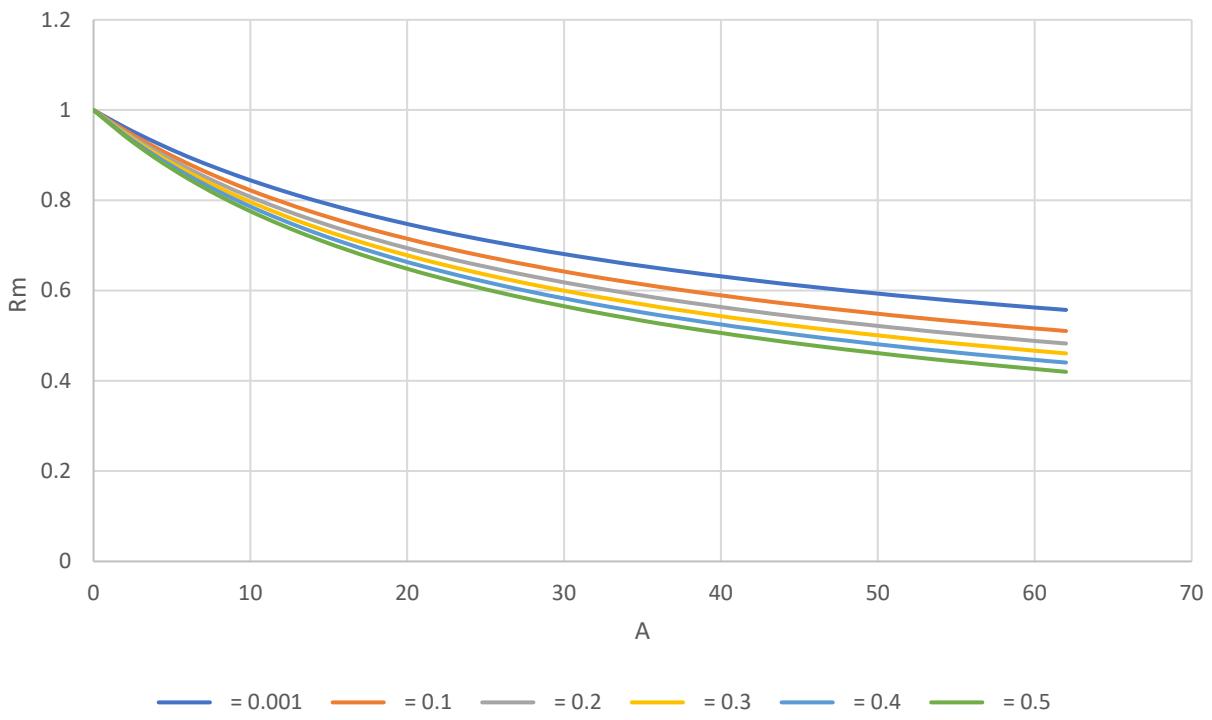
The negative moment at  $x = \pm b$  is

$$\frac{M -}{-pb^2} = Rm - = -\frac{3}{2} \left[ \frac{1}{\Gamma} \sum_{n=1}^{\infty} \frac{4 \sin \alpha_n b - 2A \left( \frac{C2}{0.5pb^2} \right) \frac{(1+\Gamma)}{\pi n} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + A} \cos \alpha_n b + \frac{C2}{0.5pb^2} \right] . 92$$

Rm- Table 6 for different t/b

A	= 0.001	= 0.1	= 0.2	= 0.3	= 0.4	= 0.5
0	1.000005	1.000006	1.000007	1.000009	1.00001	1.000011
2	0.96165	0.955499	0.951356	0.948115	0.945185	0.942193
4	0.927536	0.91628	0.90877	0.902914	0.897613	0.892197
6	0.896979	0.881437	0.871157	0.863159	0.85591	0.848502
8	0.869433	0.850255	0.837671	0.827898	0.819026	0.809958
10	0.844461	0.822169	0.807649	0.796387	0.786148	0.77568
12	0.821704	0.796725	0.780562	0.768039	0.756635	0.744975
14	0.80087	0.773553	0.755987	0.742385	0.729978	0.717294
16	0.781714	0.752351	0.733575	0.719045	0.705767	0.692195
18	0.764034	0.732866	0.713042	0.697705	0.683665	0.66932
20	0.747656	0.71489	0.694152	0.678108	0.663398	0.648373
22	0.732435	0.698245	0.676705	0.66004	0.644734	0.62911
24	0.718246	0.68278	0.660534	0.643318	0.627481	0.611327
26	0.704982	0.668369	0.645497	0.627791	0.611478	0.59485
28	0.69255	0.654901	0.631472	0.613328	0.596585	0.579535
30	0.680869	0.642281	0.618354	0.599816	0.582684	0.565255
32	0.66987	0.630427	0.606054	0.587159	0.569673	0.551904
34	0.659489	0.619265	0.594491	0.575274	0.557464	0.539388
36	0.649672	0.608734	0.583598	0.564086	0.545981	0.527627
38	0.640373	0.598777	0.573313	0.553532	0.535155	0.516551
40	0.631547	0.589346	0.563584	0.543557	0.524929	0.506097
42	0.623156	0.580396	0.554363	0.534109	0.51525	0.496212
44	0.615167	0.57189	0.545608	0.525145	0.506071	0.486846
46	0.607549	0.563791	0.537283	0.516626	0.497353	0.477958
48	0.600275	0.556069	0.529354	0.508517	0.489058	0.469508
50	0.593319	0.548697	0.52179	0.500786	0.481155	0.461464
52	0.58666	0.541648	0.514566	0.493406	0.473613	0.453794
54	0.580276	0.5349	0.507656	0.48635	0.466406	0.446471
56	0.574151	0.528433	0.501038	0.479596	0.45951	0.439471
58	0.568266	0.522227	0.494694	0.473123	0.452904	0.43277
60	0.562607	0.516266	0.488604	0.466912	0.446569	0.426348
62	0.557159	0.510534	0.482752	0.460947	0.440486	0.420187

Rm Table for different t/b

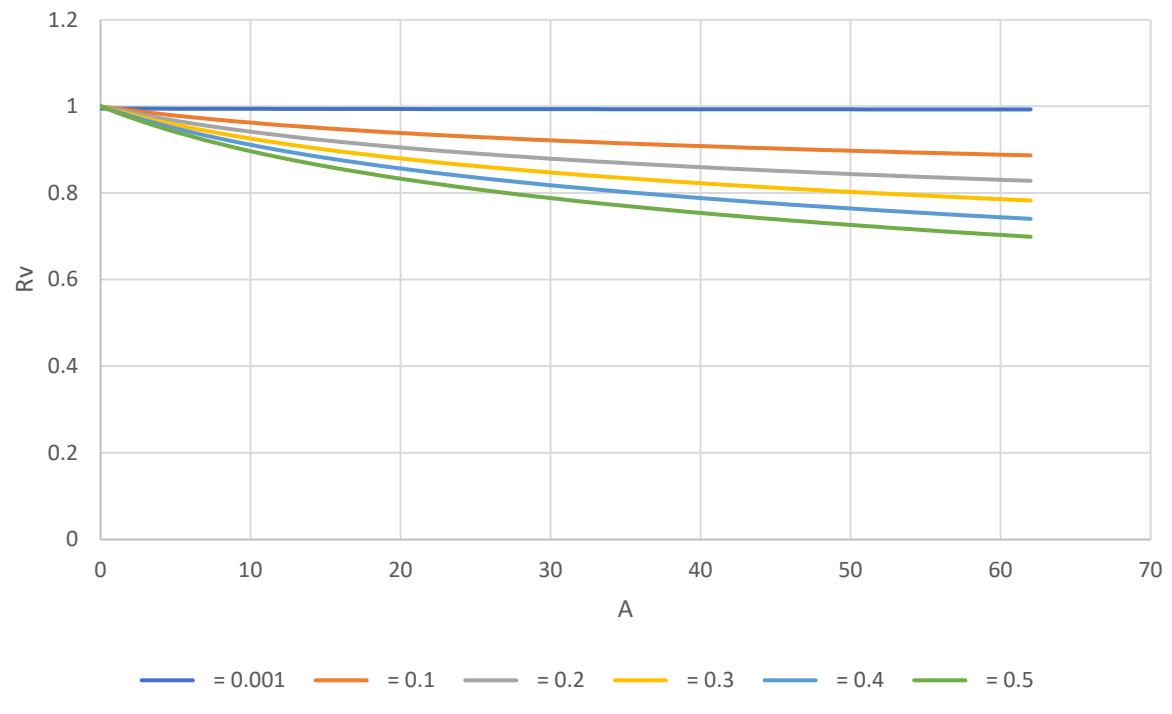


The shear at  $x = \pm b^*$  is

## Rv Table 7 for different t/b b=b\*

A	= 0.001	= 0.1	= 0.2	= 0.3	= 0.4	= 0.5
0	0.99493	0.999939	0.999964	0.999971	0.999975	0.999977
2	0.994785	0.990737	0.985426	0.981395	0.977764	0.974056
4	0.994654	0.982545	0.972578	0.965036	0.958237	0.951295
6	0.994535	0.975192	0.961116	0.950485	0.940887	0.93109
8	0.994425	0.968541	0.950808	0.937429	0.925329	0.912982
10	0.994325	0.962485	0.941468	0.925622	0.911266	0.896621
12	0.994232	0.956938	0.932953	0.914873	0.898462	0.881731
14	0.994145	0.95183	0.925142	0.905025	0.886731	0.868092
16	0.994064	0.947103	0.917941	0.895953	0.875922	0.855528
18	0.993987	0.94271	0.911271	0.887556	0.865912	0.843896
20	0.993916	0.93861	0.905065	0.879746	0.856598	0.833077
22	0.993848	0.93477	0.899269	0.872454	0.847897	0.822972
24	0.993783	0.931161	0.893836	0.86562	0.839738	0.8135
26	0.993722	0.927759	0.888726	0.859192	0.832059	0.804591
28	0.993664	0.924543	0.883907	0.853129	0.824812	0.796186
30	0.993608	0.921494	0.879348	0.847392	0.81795	0.788234
32	0.993555	0.918598	0.875024	0.84195	0.811437	0.78069
34	0.993503	0.915839	0.870913	0.836774	0.805241	0.773519
36	0.993454	0.913207	0.866997	0.831841	0.799331	0.766685
38	0.993407	0.91069	0.863258	0.82713	0.793684	0.76016
40	0.993361	0.908279	0.859682	0.822621	0.788277	0.753919
42	0.993317	0.905965	0.856255	0.818298	0.783091	0.74794
44	0.993274	0.903742	0.852965	0.814146	0.778109	0.742201
46	0.993233	0.901602	0.849802	0.810152	0.773315	0.736685
48	0.993193	0.899539	0.846756	0.806304	0.768695	0.731377
50	0.993154	0.897548	0.84382	0.802593	0.764237	0.726261
52	0.993116	0.895624	0.840985	0.799007	0.759931	0.721325
54	0.993079	0.893763	0.838245	0.79554	0.755765	0.716557
56	0.993043	0.89196	0.835593	0.792182	0.751731	0.711947
58	0.993008	0.890212	0.833025	0.788927	0.747821	0.707484
60	0.992974	0.888516	0.830533	0.78577	0.744028	0.70316
62	0.992941	0.886868	0.828115	0.782702	0.740343	0.698968

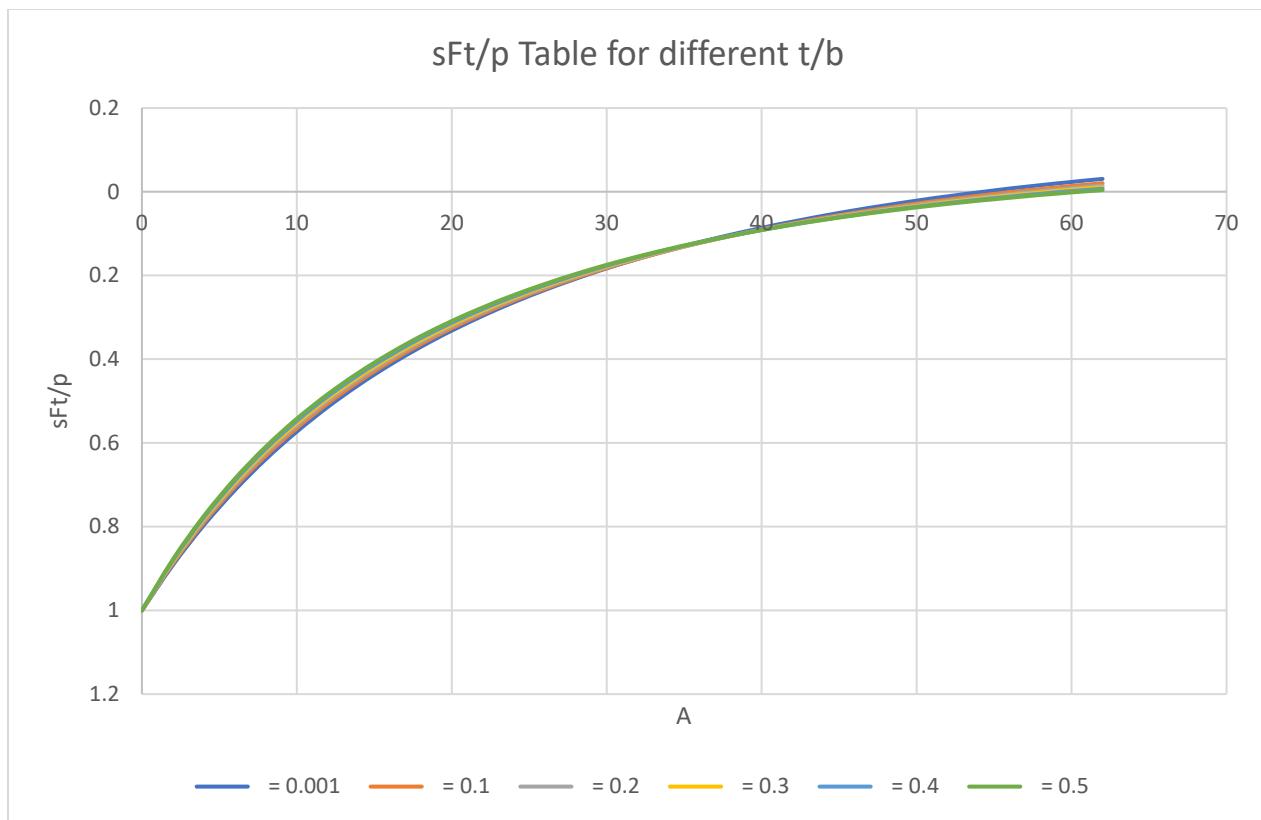
Rv Table for different t/b



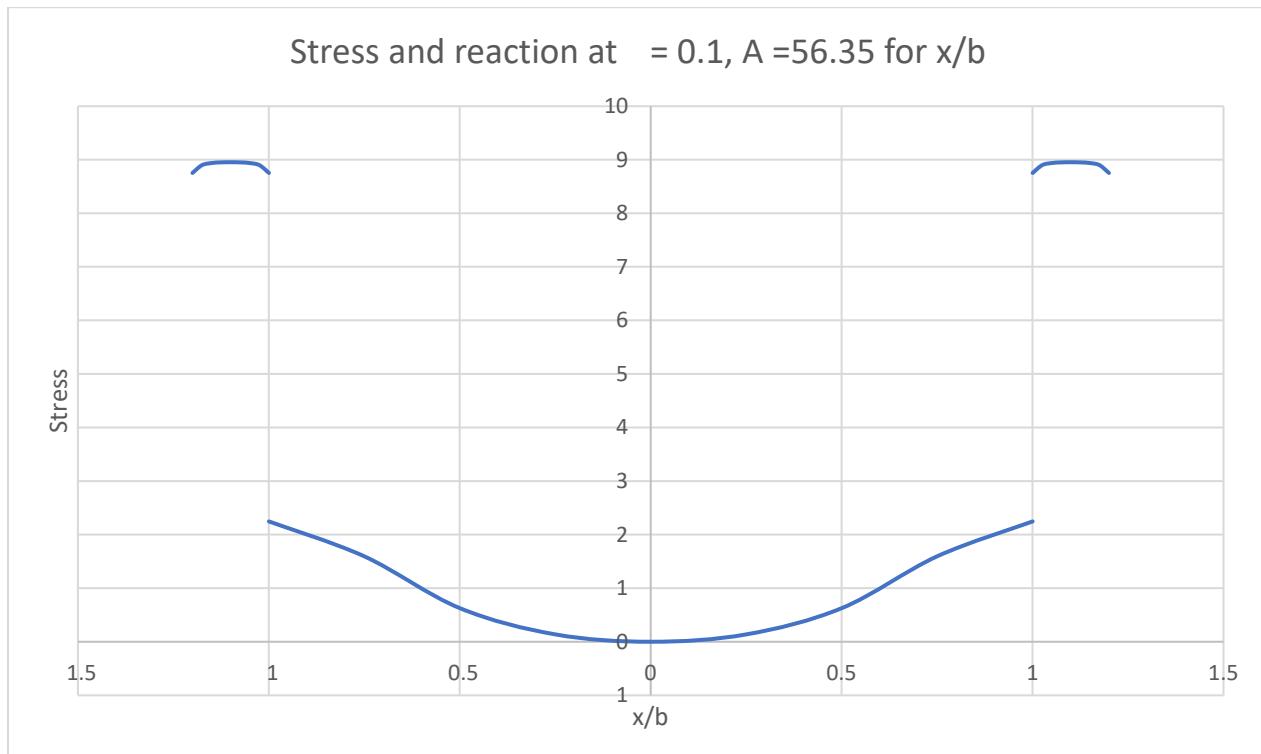
$$\frac{\sigma_y}{p} \text{ (at } y = 0 \text{ and } x = 0) = \frac{sft}{p} = \frac{A}{\Gamma} \sum_{n=1}^{\infty} \frac{2 \frac{(1 + \Gamma)}{\pi n} + \frac{\pi n}{(1 + \Gamma)} \frac{C2}{0.5pb^2}}{\frac{(\pi n)^3}{(1 + \Gamma)^3} + A} \sin \alpha_n b - 1 \dots 94$$

**sFt/p Table 8 for different t/b**

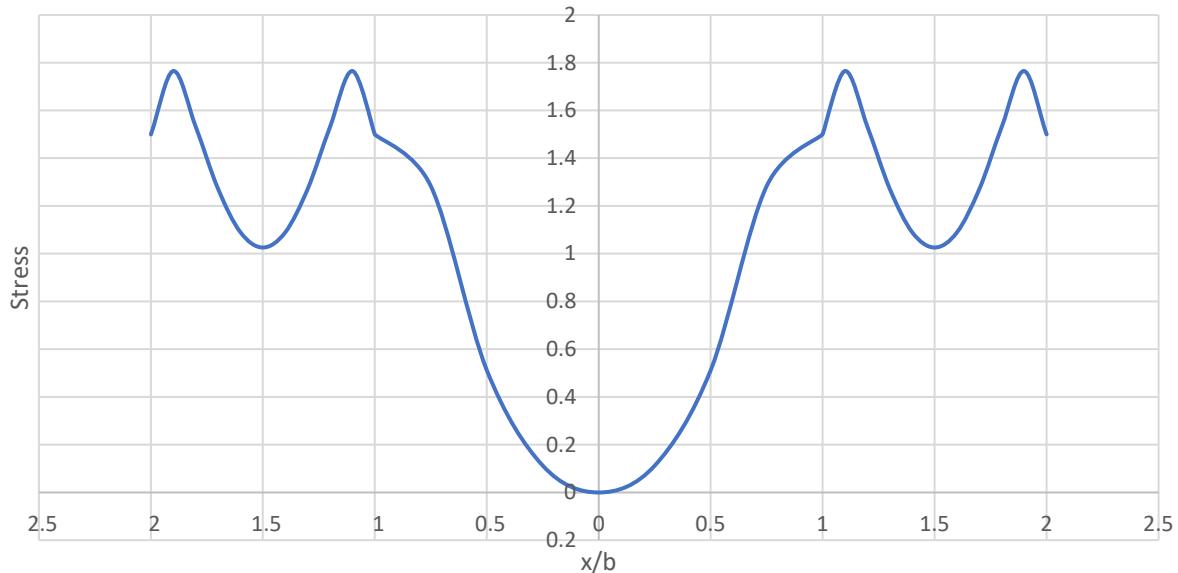
A	= 0.001	= 0.1	= 0.2	= 0.3	= 0.4	= 0.5
0	1	1	1	1	1	1
2	0.89135	0.88758	0.8847	0.88234	0.88017	0.87795
4	0.79633	0.79016	0.78545	0.78158	0.77801	0.77435
6	0.71269	0.70512	0.6993	0.69451	0.69009	0.68555
8	0.63866	0.63038	0.62399	0.61871	0.61382	0.60879
10	0.5728	0.56433	0.55773	0.55226	0.54718	0.54195
12	0.51394	0.50565	0.49912	0.49366	0.48859	0.48336
14	0.46113	0.45328	0.447	0.44173	0.4368	0.43171
16	0.41357	0.40636	0.40046	0.39547	0.39079	0.38595
18	0.37061	0.36415	0.35873	0.3541	0.34974	0.34522
20	0.33167	0.32606	0.32119	0.31697	0.31298	0.30883
22	0.29629	0.29159	0.28729	0.28352	0.27993	0.27619
24	0.26406	0.2603	0.25661	0.2533	0.25013	0.24681
26	0.23464	0.23182	0.22875	0.22592	0.22318	0.2203
28	0.20772	0.20586	0.2034	0.20104	0.19874	0.19631
30	0.18305	0.18213	0.18028	0.1784	0.17653	0.17454
32	0.1604	0.1604	0.15915	0.15774	0.1563	0.15474
34	0.13957	0.14048	0.13981	0.13885	0.13784	0.13671
36	0.12039	0.12217	0.12207	0.12156	0.12096	0.12024
38	0.1027	0.10533	0.10577	0.1057	0.1055	0.10518
40	0.08637	0.08981	0.09078	0.09113	0.09132	0.09139
42	0.07128	0.07551	0.07698	0.07772	0.07829	0.07874
44	0.05732	0.06229	0.06425	0.06538	0.06631	0.06713
46	0.04439	0.05008	0.0525	0.054	0.05528	0.05644
48	0.03241	0.03878	0.04165	0.04351	0.04511	0.04661
50	0.0213	0.02832	0.03161	0.03381	0.03574	0.03755
52	0.01098	0.01863	0.02232	0.02484	0.02708	0.02919
54	0.00141	0.00964	0.01372	0.01655	0.01908	0.02148
56	0.007489	0.00131	0.00574	0.00887	0.01168	0.01436
58	0.015757	0.006429	0.001649	0.00176	0.00484	0.00778
60	0.023445	0.013613	0.008506	0.004829	0.001499	0.00169
62	0.030595	0.020286	0.014869	0.010938	0.007365	0.003943



In the following graph the pressure on the lagging is shown positive to see how the distribution is for various  $x/b$  and the reaction is shown.



Stress and reaction at  $t/b = 0.5$ ,  $A = 60.6$  for  $x/b$



Thus  $sft/p$  has a maximum positive tension value see table below.

$t/b$	$A_{max}$	$sft/p$ max	$R_{m+}$	$R_m$	$R_v$
0.001	140.5	0.114742	0.098186	0.432431	0.991999
0.1	141.25	0.098316	0.091545	0.379861	0.84313
0.2	141.25	0.088702	0.087597	0.350644	0.764456
0.3	141	0.081035	0.084239	0.32706	0.701593
0.4	140.5	0.073923	0.08117	0.305358	0.644224
0.5	141	0.067442	0.077476	0.28453	0.592369

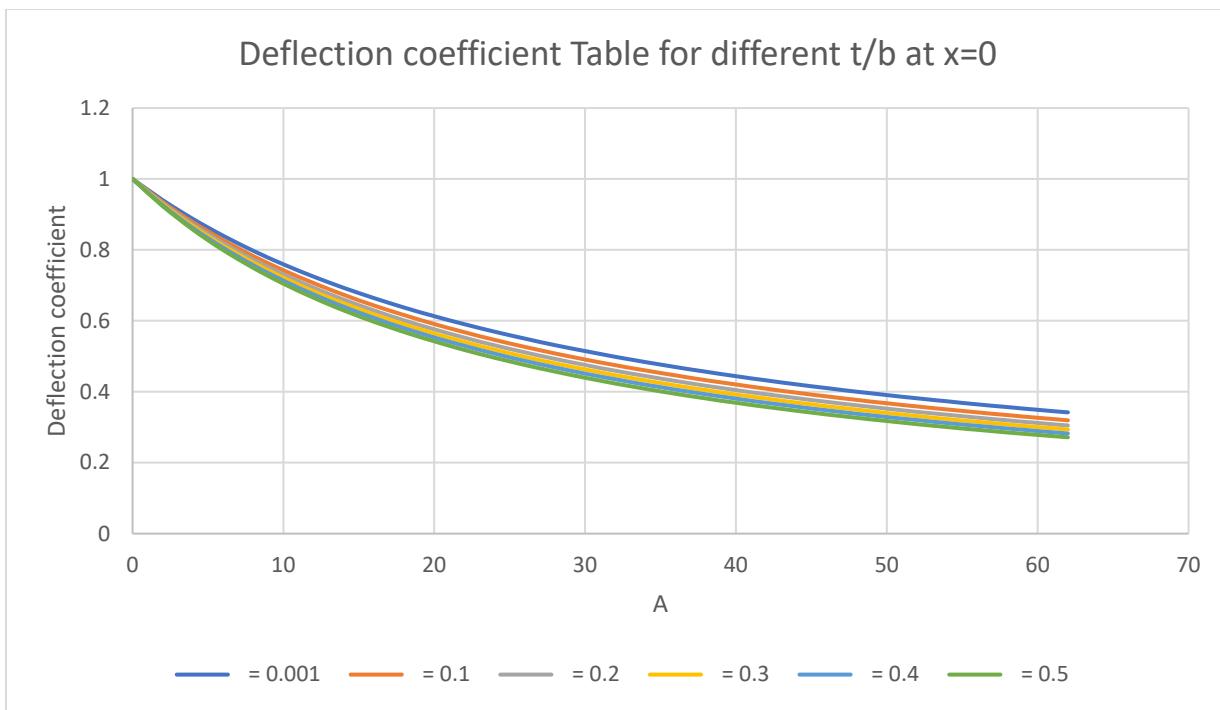
For the condition where  $sft/p$  is zero for soils see table below.

$t/b$	$A_{max}$	Stress	$R_{m+}$	$R_m$	$R_v$
0.001	54.3	3.1E 05	0.273793	0.579342	0.993073
0.1	56.35	8.8E 05	0.251128	0.527328	0.89165
0.2	57.55	3.35E 05	0.236933	0.496099	0.833596
0.3	58.5	6.5E 05	0.225752	0.471547	0.788129
0.4	59.5	3.9E 05	0.215002	0.448129	0.744966
0.5	60.6	4.77E 05	0.204052	0.424473	0.701889

Now the deflection

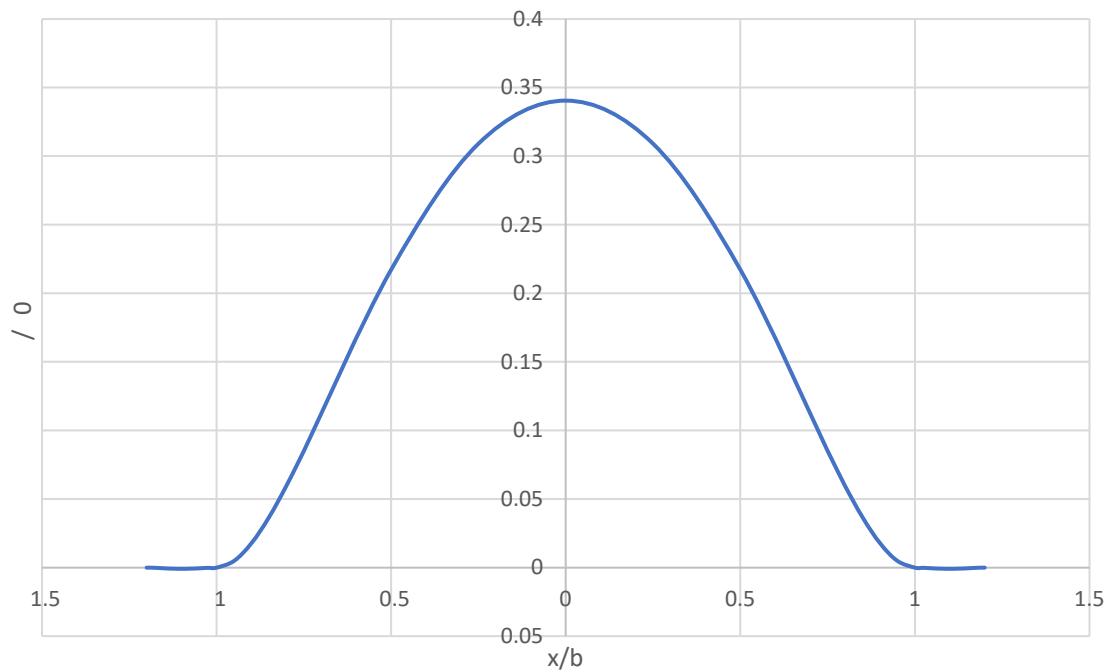
## Deflection coefficient Table 9 for different x=0

A	= 0.001	= 0.1	= 0.2	= 0.3	= 0.4	= 0.5
0	0.99998	0.999975	0.999971	0.999966	0.99996	0.999954
2	0.940169	0.93502	0.931359	0.928444	0.925795	0.923088
4	0.887196	0.878028	0.871562	0.866427	0.861758	0.856983
6	0.83995	0.82762	0.818985	0.812141	0.80591	0.799535
8	0.79755	0.782717	0.772395	0.764226	0.756778	0.749154
10	0.759287	0.742465	0.730825	0.721623	0.71322	0.704616
12	0.724582	0.706176	0.693505	0.683496	0.674342	0.664965
14	0.692961	0.673294	0.659816	0.649176	0.639428	0.629441
16	0.664031	0.643359	0.629253	0.618121	0.607904	0.597436
18	0.637461	0.615992	0.601401	0.589886	0.5793	0.568454
20	0.612974	0.590877	0.575913	0.564104	0.553229	0.542088
22	0.590334	0.567747	0.552503	0.54047	0.52937	0.518002
24	0.569339	0.546376	0.530925	0.518726	0.507454	0.495913
26	0.549817	0.526569	0.510973	0.498655	0.487254	0.475586
28	0.531617	0.508162	0.492471	0.48007	0.468576	0.456818
30	0.514609	0.491012	0.475265	0.462814	0.451255	0.439439
32	0.498679	0.474993	0.459224	0.446748	0.435149	0.4233
34	0.483729	0.459997	0.444234	0.431755	0.420136	0.408275
36	0.46967	0.44593	0.430196	0.417729	0.406107	0.394254
38	0.456425	0.432708	0.41702	0.404582	0.39297	0.381138
40	0.443925	0.420256	0.404631	0.392232	0.380642	0.368845
42	0.432109	0.40851	0.392959	0.38061	0.369052	0.3573
44	0.420922	0.397411	0.381945	0.369653	0.358136	0.346438
46	0.410315	0.386907	0.371534	0.359305	0.347836	0.336199
48	0.400244	0.376951	0.361678	0.349519	0.338102	0.326533
50	0.390669	0.367502	0.352335	0.340248	0.328889	0.317393
52	0.381555	0.358522	0.343464	0.331454	0.320157	0.308738
54	0.372868	0.349977	0.335031	0.323101	0.311868	0.30053
56	0.36458	0.341835	0.327005	0.315157	0.303991	0.292736
58	0.356664	0.33407	0.319357	0.307591	0.296495	0.285327
60	0.349094	0.326655	0.312061	0.300379	0.289354	0.278273
62	0.341849	0.319567	0.305092	0.293496	0.282543	0.271552

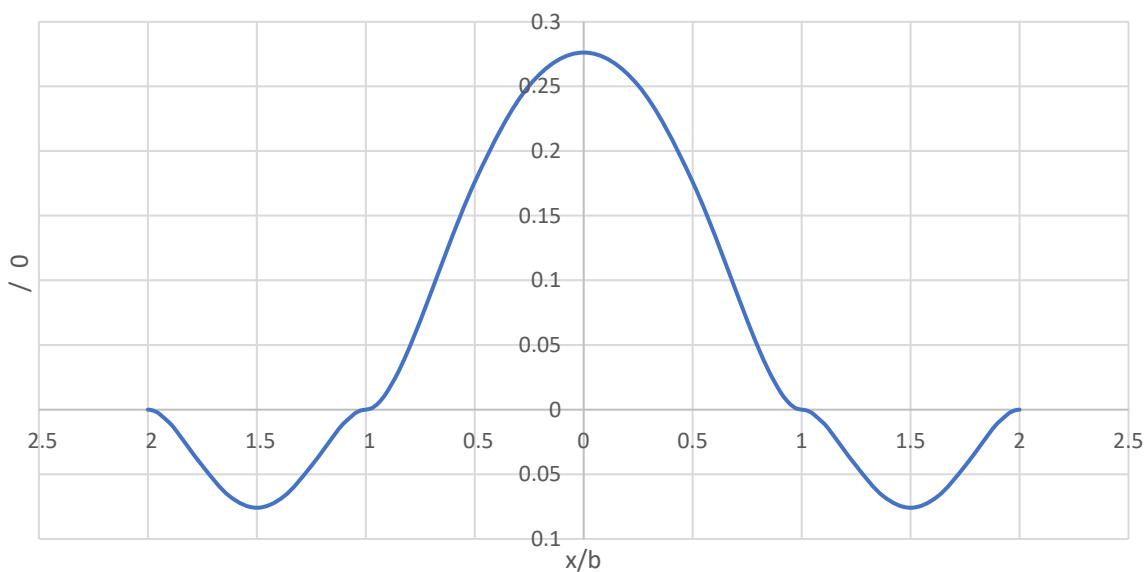


In the following graph the deflection coefficient on the lagging is shown positive at the lagging to see how the distribution is for various  $x/b$ . We see the deflection at the column is practically zero for  $t/b = 0.1$ .

$/_0$  at  $= 0.1, A = 56.35$  for  $x/b$



$/_0$  at  $= 0.5, A = 60.6$  for  $x/b$



The deflection does not take into account the moment of inertia of the elastic media, so it is conservative. But for a media such as soils where tension is not allowed it is accurate and useful.

Once  $D$  is found the stresses at  $q = 0$  and  $h = \infty$  are the same as before only C2 is different:

$$\sigma_x = k_0 \sum_{n=\varphi}^{\infty} D \cos \alpha_n x \left( 1 + \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} - 2k_0 \sum_{n=\varphi}^{\infty} D e^{-\alpha_n y} \cos \alpha_n x - k_0 p \dots \dots \dots 99$$

### Example 2:

We redo example 1 with fixed end condition:

Example in article the strain at y=0 is zero

	b =	2	ft		4 ft opening
	t =	1	ft	s =	3 ft
Concrete	150 pcf		at the powerhouse at the dam		
p =	13500	lbs/ft	30 ft height x 3 ft thick		
p =	1125	lbs/in	Pick t/b = 0.5		
p/inch =	1125	psi	v =	0.2	
			We use ACI 318 11 code use Ft =		
Ft =	87.63561	psi	1.6 3000		
sFt/p/ =	0.3895	>0.067442	F'c =	3000	psi

$$M_n = 5\lambda \sqrt{f_c'} S_m \quad (22-2)$$

$$V_n = 4/3 \lambda \sqrt{f_c} b_w h \quad (22-9)$$

$$A = \frac{4}{(3-\nu)(1+\nu)} \left[ \frac{sb^3 E_s}{2EI} \right] \quad Es = E$$

Thus	IA =	14.28571	
	for sFt/p =		
A =	141	0.077803	0.067442
thus I =	0.101317	ft^4	
thus I =	2100.912	in^4	= sh^3/12
h^3 =	700.304		h = 8.880325 in
Sx =	473.161	in^3	A = 319.6917 in^2
1.3pb^2/3			
Mu =	= 23400	Lbs ft	Ultimate Moment
Mu reduced =	6042.493	k ft	Rm = 0.28453
ft = M/Sx	168.8558	< Ft =	273.8613 psi
Rv =	0.070038		At x = 0.63b
1.3fv =	11.53453	> Fv =	73.02967 psi

So, the tension sFt/p will act upward on the beam with 0.067442 at the center. The beam is ok in compression also.

If we don't allow tension in concrete			
sFt/p = 0.30883			
Use compression at A = 20			Nominal
Thus	IA =	14.28571	
A =	20		
thus I =	0.714286	ft^4	
thus I =	14811.43	in^4	= sh^3/12
h^3 =	4937.143		h = 17.0278 in
Sx =	1739.676	in^3	A = 613.0008 in^2
1.3pb^2/3			
Mu =	= 23400	Lbs ft	Ultimate Moment
Mu reduced =	15171.93	k ft	Rm = 0.648373
ft = M/Sx	104.6535	< Ft =	273.8613 psi OK
Rv =	0.290508		At x = 0.291b
1.3fv =	24.95145	< Fv =	73.02967 psi OK

So, Rm- and Rv was found for A = 20 from the tables. So, we can see that the opening can be installed. The infinite height used as an assumption should be reduced by 17.0278 inch which does not change the solution.

### Closed form solution #3 simple span beam with height $h$ :

When starting with equation 24 this process can be repeated where  $s/2\beta$  is skn

The load on the lagging and reaction at  $y = 0$  is

The reaction is

Thus

We note  $w_n$  is Fourier cosine for the load  $q$  on the beam at  $y = h$  thus

We find the constant term in equation 102 and 103 is for all  $x$  thus they cannot produce arching and they cancel each other. Thus, they are dropped, and the shear is

or

$$\int \sigma_y = - \sum_{n=1}^{\infty} \left( s \frac{k_n}{\beta} v_n \right) \sin \alpha_n x - \sum_{n=1}^{\infty} \left( \frac{2p \sin \alpha_n b}{t} + \frac{W_n}{\alpha_n} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^2} \right) \sin \alpha_n x$$

Thus

$$v_n = - \frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^5} + \frac{W_n}{\alpha_n^4} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^5} + \frac{2C2}{t} \frac{\sin \alpha_n b}{\alpha_n^3}}{EI + \frac{sk_n}{\beta \alpha_n^3}} \quad \dots \dots \dots \quad 106$$

Finding  $v_0$  set the equation to

$$EIv_n = - \frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^5} + \frac{W_n}{\alpha_n^4} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^5} + \frac{2C2}{t} \frac{\sin \alpha_n b}{\alpha_n^3}}{1 + \frac{sk_n}{\beta EI \alpha_n^3}}$$

$$EIv_n = - \frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^2} + \frac{W_n}{\alpha_n} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^2} + \frac{2C2}{t} \sin \alpha_n b}{\alpha_n^3 + \frac{sk_n}{\beta EI}}$$

At  $n = 0$  is the same as saying  $h = 0$  and we have no medium.  $kn = 0$   $Wn = -wn$ . and  $v_0 = C4 = \infty$ . Again, making the argument  $v_0$  and  $C4$  will be calculated when setting the deflection to zero at  $\pm b$ .

$$\iint \sigma_y = \sum_{n=1}^{\infty} \left( \frac{sk_n v_n}{\beta \alpha_n} \right) \cos \alpha_n x + \sum_{n=1}^{\infty} \left( \frac{2p \sin \alpha_n b}{t \alpha_n^3} + \frac{W_n}{\alpha_n^2} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^3} \right) \cos \alpha_n x + C2$$

$$\begin{aligned} \iint \sigma_y = & - \sum_{n=1}^{\infty} \left( \frac{sk_n}{\beta EI \alpha_n} \right) \frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^2} + \frac{W_n}{\alpha_n} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^2} + \frac{2C2}{t} \sin \alpha_n b}{\alpha_n^3 + \frac{sk_n}{\beta EI}} \cos \alpha_n x \\ & + \sum_{n=1}^{\infty} \left( \frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^3} + \frac{W_n}{\alpha_n^2} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^3} \right) \cos \alpha_n x + C2 \end{aligned}$$

$$\begin{aligned} \iint \sigma_y = & - \sum_{n=1}^{\infty} \left( \frac{sk_n}{\beta EI} \right) \frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^3} + \frac{W_n}{\alpha_n^2} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^3} + \frac{2C2}{t \alpha_n} \sin \alpha_n b}{\alpha_n^3 + \frac{sk_n}{\beta EI}} \cos \alpha_n x \\ & + \sum_{n=1}^{\infty} \left( \frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^3} + \frac{W_n}{\alpha_n^2} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^3} \right) \cos \alpha_n x + C2 \end{aligned}$$

$$\begin{aligned} \int \int \sigma_y = & \sum_{n=1}^{\infty} \frac{\left[ \frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^3} + \frac{W_n}{\alpha_n^2} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^3} \right] \alpha_n^3 - \left( \frac{sk_n}{\beta EI \alpha_n} \right) \frac{2C2}{t} \sin \alpha_n b}{\alpha_n^3 + \frac{sk_n}{\beta EI}} \cos \alpha_n x + \\ & + C2 \end{aligned}$$

$$\int \sigma_y = \sum_{n=1}^{\infty} \frac{\frac{2p}{t} \sin \alpha_n b + \alpha_n W_n + 2q \frac{b}{t(b+t)} \sin \alpha_n b - \left( \frac{sk_n}{\beta EI \alpha_n} \right) \frac{2C2}{t} \sin \alpha_n b}{\alpha_n^3 + \frac{sk_n}{\beta EI}} \cos \alpha_n x$$

+ C2 ..... 107

At  $x = b$  the moment is zero

$$\sum_{n=1}^{\infty} \frac{\frac{2p}{t} \sin \alpha_n b + \alpha_n W_n + 2q \frac{b}{t(b+t)} \sin \alpha_n b}{\alpha_n^3 + \frac{sk_n}{\beta EI}} \cos \alpha_n b = -C2 + \frac{\left(\frac{sk_n}{\beta EI \alpha_n}\right) \frac{2C2}{t}}{\alpha_n^3 + \frac{sk_n}{\beta EI}} \sin \alpha_n b$$

$$C_2 = \frac{\sum_{n=1}^{\infty} \frac{\frac{2p}{t} \sin \alpha_n b + \alpha_n W_n + 2q \frac{b}{t(b+t)} \sin \alpha_n b}{\alpha_n^3 + \frac{sk_n}{\beta EI}} \cos \alpha_n b}{-1 + \frac{\left(\frac{sk_n}{\beta EI \alpha_n}\right)^2 t}{\alpha_n^3 + \frac{sk_n}{\beta EI}} \sin \alpha_n b}$$

$$C_2 = \frac{b^3 \sum_{n=1}^{\infty} \frac{\frac{2p}{\Gamma b} \sin \alpha_n b + \frac{\pi n}{(1+\Gamma)b} W_n + 2q \frac{1}{\Gamma(1+\Gamma)b} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + \frac{sk_n b^3}{\beta EI}} \cos \alpha_n b}{-1 + \frac{\left(\frac{sk_n b^3}{\beta EI \alpha_n}\right) \frac{2}{t}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + \frac{sk_n b^3}{\beta EI}} \sin \alpha_n b}$$

And the moment is

$$\int \sigma_y = \sum_{n=1}^{\infty} \frac{\frac{2p}{t} \sin \alpha_n b + \alpha_n W_n + 2q \frac{b}{t(b+t)} \sin \alpha_n b - \left( \frac{sk_n}{\beta EI \alpha_n} \right) \frac{2C2}{t} \sin \alpha_n b}{\alpha_n^3 + \frac{sk_n}{\beta EI}} \cos \alpha_n x + C2$$

And the shear is,

And the stress is,

Thus

Again, when  $I$  is infinite or  $E_s$  is zero as in water, we have C2 with  $W_n = w_n$ ,

$$\begin{aligned}
C_2 &= - \sum_{n=1}^{\infty} \left( \frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^3} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n} + \frac{2q}{(b+t)} \frac{\sin \alpha_n b}{\alpha_n} \right) \cos \alpha_n b \\
&= - \sum_{n=1}^{\infty} \left( \frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^3} + \frac{2q \sin \alpha_n b}{t \alpha_n^3} \right) \cos \alpha_n b = -(p+q) \frac{b^2}{2} \quad \text{in } -b \leq x \leq b
\end{aligned}$$

We will set first  $q$  to zero and calculate  $Rm$   $Rv$  and the stress ratio. Now  $A$  can be zero for infinite moment of inertia, but  $E_s$  cannot be zero because of Hooke's law does not work for water in the deflection. We find there is live load stress distribution for  $A = 0$  because of the simple span requirements and  $I$  do not enter the stress in the right side of equation 22 and in this case the stress is when  $A$  is zero for small  $s$ . When we do the fixed end solution, which is more realistic, the stress at  $A = 0$  is a different distribution. There is a region for  $t/b$  where the thickness of the column causes more arching meaning a thinner column cause more arching for some  $t/b$ . This means a thinner column for some  $t/b$  sucks up more load because it becomes a hard spot with deflection close to zero. This is where the deflection at the column makes a difference. When  $t/b$  is large a thicker column causes more arching or sucks up more load.

Thus, at  $q = 0$

and  $x = 0$  we have

Note: the maximum moment may not be at  $x = 0$  depending on  $A$

$$Rm1 = \frac{\iint \sigma_y}{0.5pb^2} = \frac{1}{\Gamma} \sum_{n=1}^{\infty} \frac{4 \sin \alpha_n b - 4k_n A \left( \frac{C2}{0.5pb^2} \right) \frac{(1+\Gamma)}{\pi n} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} + \frac{C2}{0.5pb^2} \dots \dots \dots 113$$

Table 10 Rm 1 verses h/b for  $\gamma = 0.1$

h/b	A = 0	A = 5	A = 10	A = 15	A = 20	A = 25	A = 30
1E 13	1	1	1	1	1	1	1
0.1	1	0.996449	0.992929	0.989439	0.985978	0.982547	0.979145
0.2	1	0.975475	0.952305	0.930378	0.909593	0.88986	0.871099
0.3	1	0.933979	0.876881	0.826976	0.782958	0.74382	0.708774
0.4	1	0.876778	0.78172	0.70609	0.644434	0.593167	0.549838
0.5	1	0.813006	0.685413	0.592778	0.522453	0.467235	0.42272
0.6	1	0.752675	0.602241	0.501212	0.428762	0.37432	0.331949
0.7	1	0.702067	0.537632	0.433687	0.362221	0.310186	0.270685
0.8	1	0.66298	0.49062	0.386386	0.316823	0.267269	0.230292
0.9	1	0.634414	0.457733	0.354168	0.286449	0.238923	0.203869
1	1	0.614298	0.435278	0.332568	0.26633	0.220305	0.186624
1.5	1	0.578338	0.396517	0.296022	0.232726	0.189489	0.158269
2	1	0.574166	0.392131	0.291944	0.229011	0.186103	0.155168

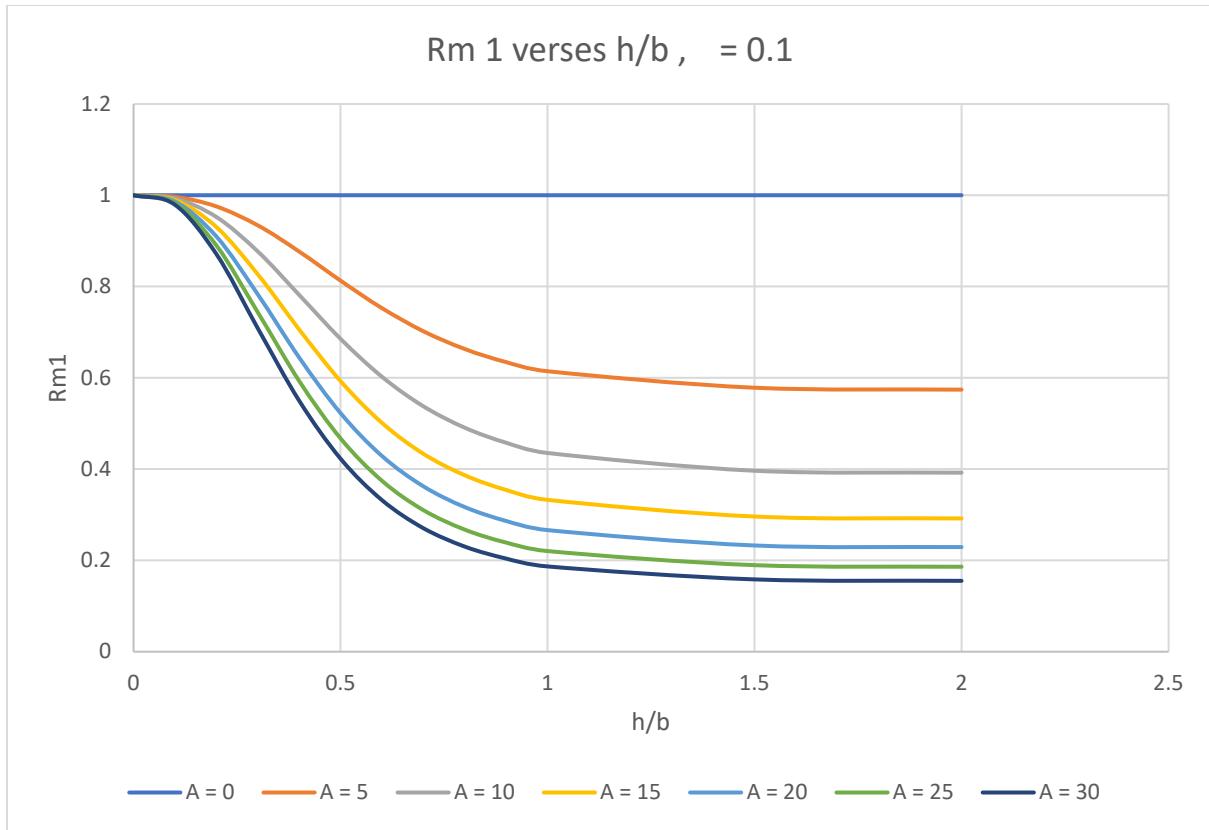
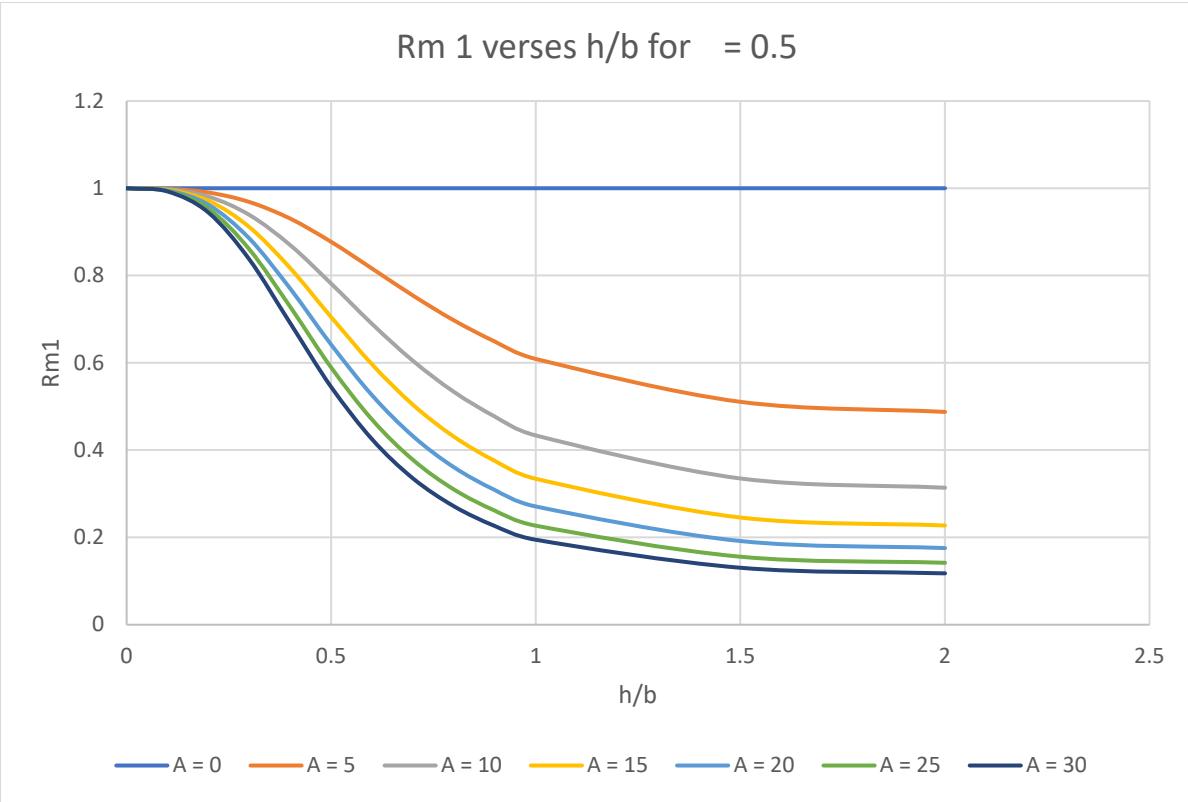


Table 11 Rm 1 verses  $h/b$  for  $= 0.5$

$h/b$	$A = 0$	$A = 5$	$A = 10$	$A = 15$	$A = 20$	$A = 25$	$A = 30$
1E 13	1	1	1	1	1	1	1
0.1	1	0.998757	0.997516	0.996279	0.995045	0.993814	0.992585
0.2	1	0.990256	0.980702	0.971331	0.962138	0.953119	0.944268
0.3	1	0.96847	0.93889	0.911082	0.884891	0.860179	0.836823
0.4	1	0.930241	0.869694	0.816639	0.76976	0.728033	0.690648
0.5	1	0.877276	0.781566	0.704822	0.641904	0.589377	0.544858
0.6	1	0.815961	0.689069	0.596299	0.525528	0.469766	0.4247
0.7	1	0.753995	0.604342	0.503795	0.431645	0.377386	0.335122
0.8	1	0.697228	0.533319	0.430758	0.360647	0.309761	0.271193
0.9	1	0.648664	0.476936	0.375435	0.308569	0.261302	0.226186
1	1	0.609013	0.433588	0.334387	0.270825	0.226756	0.194493
1.5	1	0.510505	0.335085	0.245555	0.191601	0.155743	0.130316
2	1	0.48759	0.313862	0.227177	0.175613	0.141649	0.117729



Thus, at  $q = 0$  and  $x = b^*$  we have

Table 12 Rv 1 verses h/b for  $\gamma = 0.1$  at  $b = b^*$

h/b	A = 0	A = 5	A = 10	A = 15	A = 20	A = 25	A = 30
1E 13	0.999755	0.999755	0.999755	0.999755	0.999755	0.999755	0.999755
0.1	0.999755	0.980388	0.96121	0.942218	0.92341	0.904783	0.886335
0.2	0.999755	0.941074	0.88597	0.83414	0.785314	0.73925	0.695732
0.3	0.999755	0.908958	0.831256	0.764081	0.705492	0.653995	0.608422
0.4	0.999755	0.886445	0.799749	0.731367	0.676118	0.630602	0.592494
0.5	0.999755	0.871046	0.783078	0.719088	0.670398	0.632068	0.601079
0.6	0.999755	0.860401	0.774357	0.715577	0.67263	0.639708	0.613543
0.7	0.999755	0.852953	0.76961	0.715185	0.676407	0.647082	0.623927
0.8	0.999755	0.847742	0.766876	0.715675	0.679734	0.652726	0.631427
0.9	0.999755	0.844131	0.765223	0.71628	0.682225	0.656703	0.636564
1	0.999755	0.84166	0.764187	0.716785	0.683972	0.659406	0.64
1.5	0.999755	0.837346	0.762532	0.717777	0.687024	0.664008	0.645767
2	0.999755	0.836851	0.762352	0.717896	0.68737	0.664521	0.646405

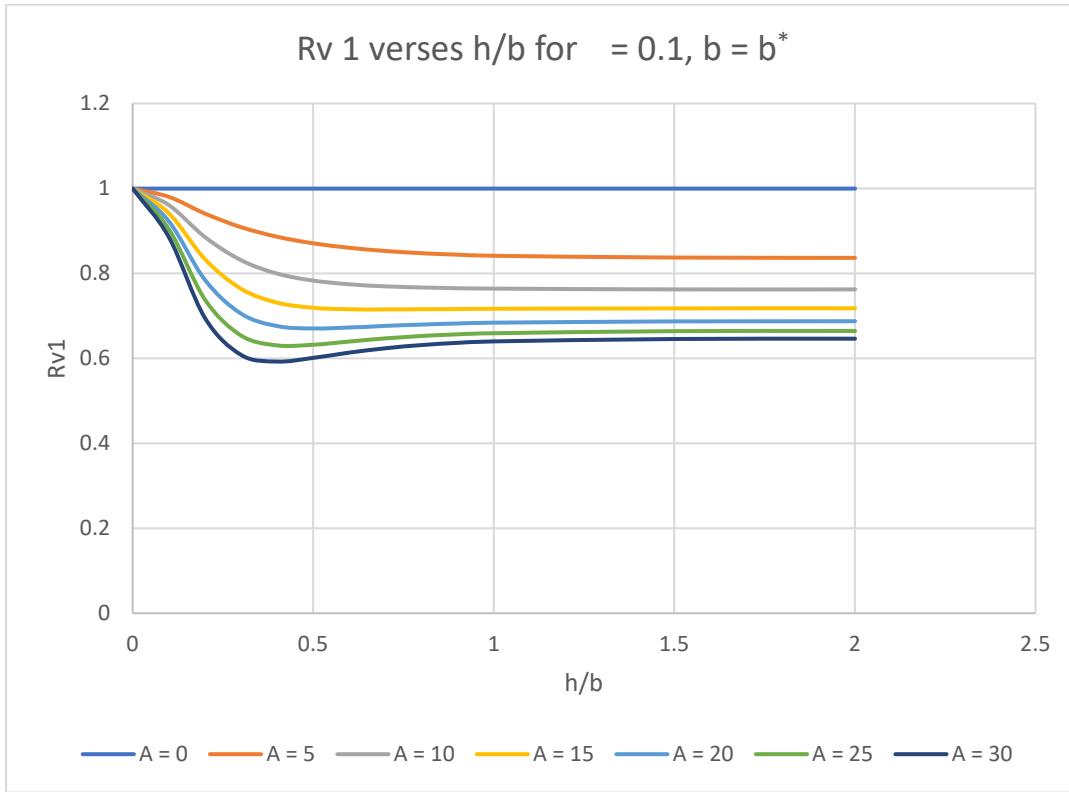
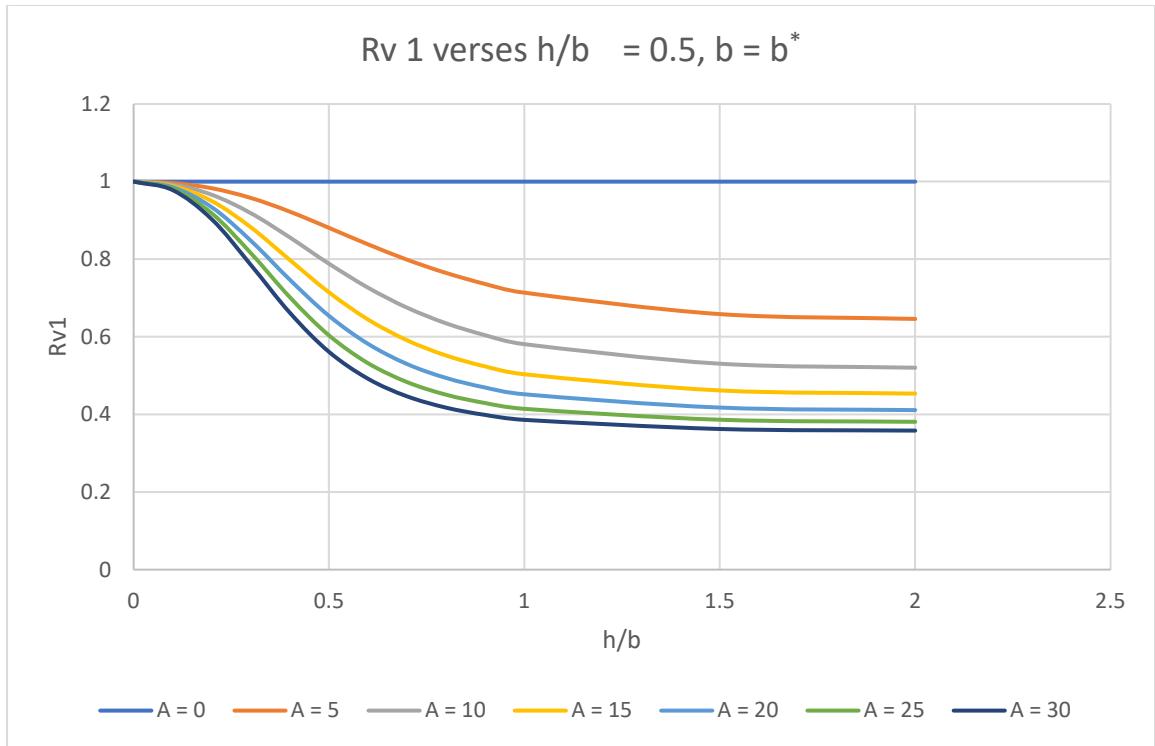


Table 13 Rv 1 verses h/b for  $\gamma = 0.5$   $b = b^*$

$h/b$	$A = 0$	$A = 5$	$A = 10$	$A = 15$	$A = 20$	$A = 25$	$A = 30$
1E 13	0.999909	0.999909	0.999909	0.999909	0.999909	0.999909	0.999909
0.1	0.999909	0.996247	0.992597	0.988959	0.985331	0.981716	0.978111
0.2	0.999909	0.982858	0.966192	0.949899	0.933968	0.918387	0.903146
0.3	0.999909	0.957919	0.918768	0.882191	0.847954	0.815848	0.785689
0.4	0.999909	0.922535	0.855827	0.797763	0.746801	0.70174	0.661634
0.5	0.999909	0.880926	0.788425	0.714489	0.654066	0.60378	0.561294
0.6	0.999909	0.838325	0.726542	0.644524	0.581719	0.532042	0.491733
0.7	0.999909	0.798776	0.675075	0.590965	0.529844	0.483278	0.446521
0.8	0.999909	0.764494	0.634697	0.551796	0.493866	0.450851	0.417483
0.9	0.999909	0.736173	0.60403	0.523665	0.469065	0.429206	0.398606
1	0.999909	0.713553	0.581133	0.503545	0.451866	0.414553	0.386082
1.5	0.999909	0.658659	0.530801	0.461854	0.417677	0.386358	0.362637
2	0.999909	0.646061	0.520186	0.453484	0.411046	0.381037	0.358316



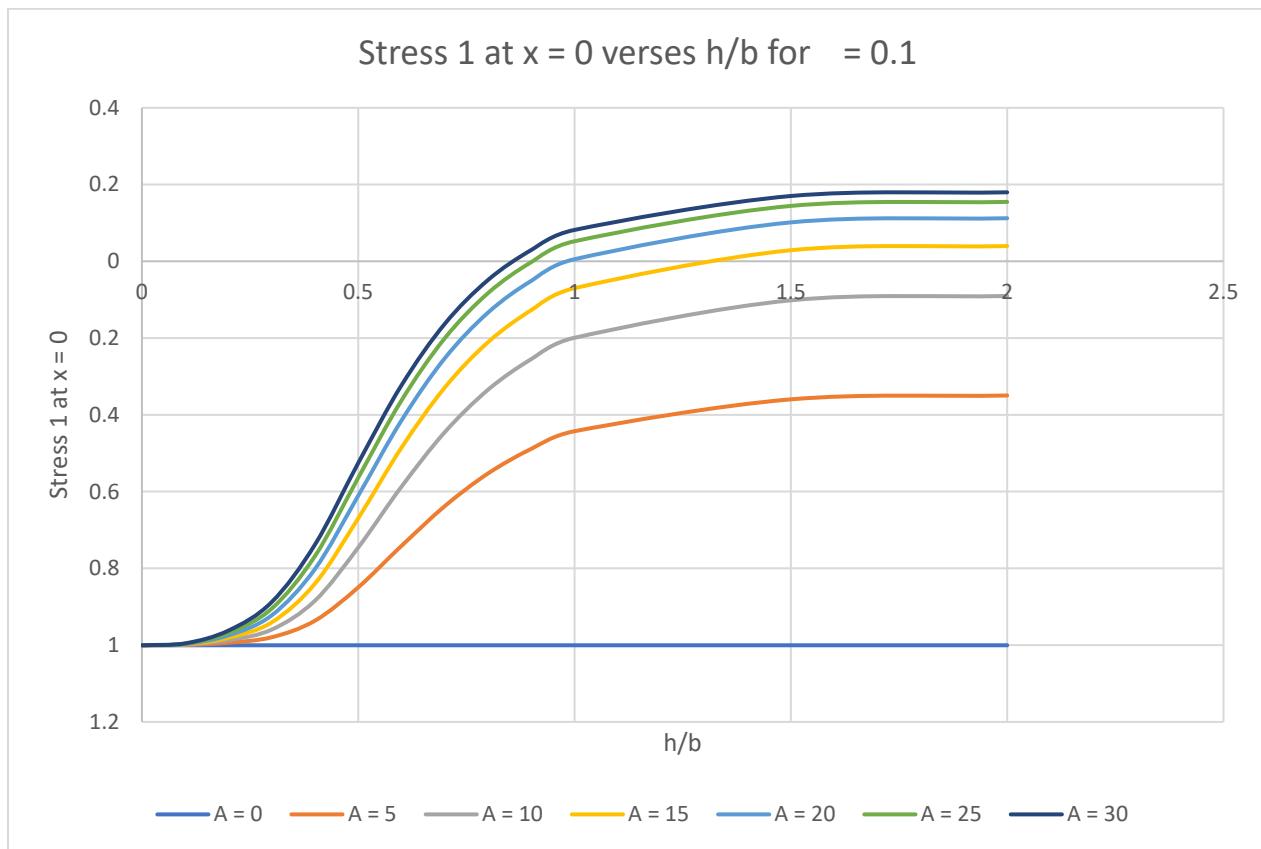
$$\frac{\sigma_y}{p} \text{ at } x = 0 = \frac{sft1}{p} = \frac{1}{\Gamma} \sum_{n=1}^{\infty} 2k_n A \frac{2 \frac{(1+\Gamma)}{\pi n} + \frac{\pi n}{(1+\Gamma)} \frac{C2}{0.5pb^2} \sin \alpha_n b - 1}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \dots \dots \dots 115$$

At the reaction:

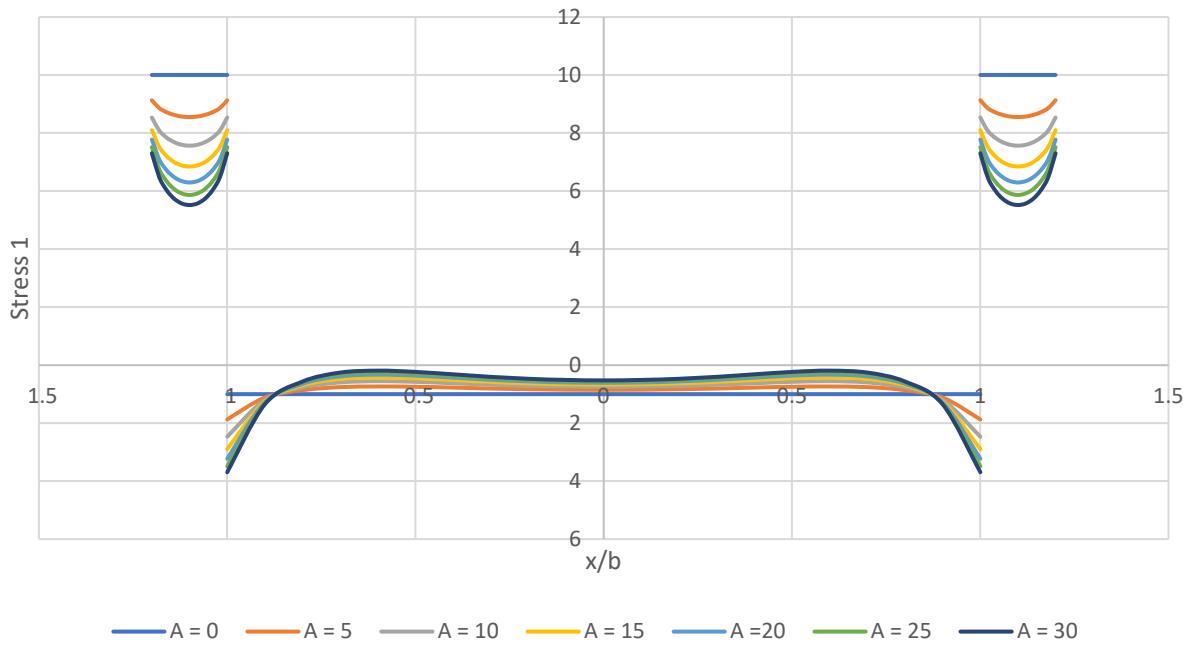
$$\begin{aligned} \text{at } x = b + t &= \frac{sft1}{p} \\ &= \frac{1}{\Gamma} \sum_{n=1}^{\infty} 2k_n A \frac{2 \frac{(1+\Gamma)}{\pi n} + \frac{\pi n}{(1+\Gamma)} \frac{C2}{0.5pb^2} \sin \alpha_n b \cos \pi n + \frac{1}{\Gamma}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \dots \dots \dots 116 \end{aligned}$$

Table 14 Stress 1 at  $x = 0$  verses  $h/b$  for  $\epsilon = 0.2$   $\epsilon = 0.1$

$h/b$	$A = 0$	$A = 5$	$A = 10$	$A = 15$	$A = 20$	$A = 25$	$A = 30$
1E 13	1	1	1	1	1	1	1
0.1	1	0.99917	0.99834	0.99751	0.99668	0.99585	0.99503
0.2	1	0.99344	0.98697	0.98057	0.97425	0.968	0.96183
0.3	1	0.97921	0.95937	0.94038	0.92217	0.90467	0.88783
0.4	1	0.93614	0.88374	0.83945	0.80117	0.76749	0.73745
0.5	1	0.84945	0.74559	0.66926	0.61054	0.56377	0.52549
0.6	1	0.74107	0.58666	0.48519	0.41409	0.36193	0.32231
0.7	1	0.63765	0.44506	0.32863	0.25254	0.20015	0.16275
0.8	1	0.55263	0.33472	0.21065	0.13354	0.08296	0.04859
0.9	1	0.4885	0.25472	0.12709	0.05055	0.00208	0.029568
1	1	0.44258	0.19903	0.06985	0.005723	0.052364	0.081927
1.5	1	0.35941	0.10136	0.028745	0.101572	0.144419	0.169988
2	1	0.34969	0.09022	0.039845	0.112276	0.154645	0.179733



Stress 1 and reaction,  $\Gamma = 0.1$ ,  $h/b = 0.5$



Stress 1 and reaction at  $\Gamma = 0.1$ ,  $h/b = 1$

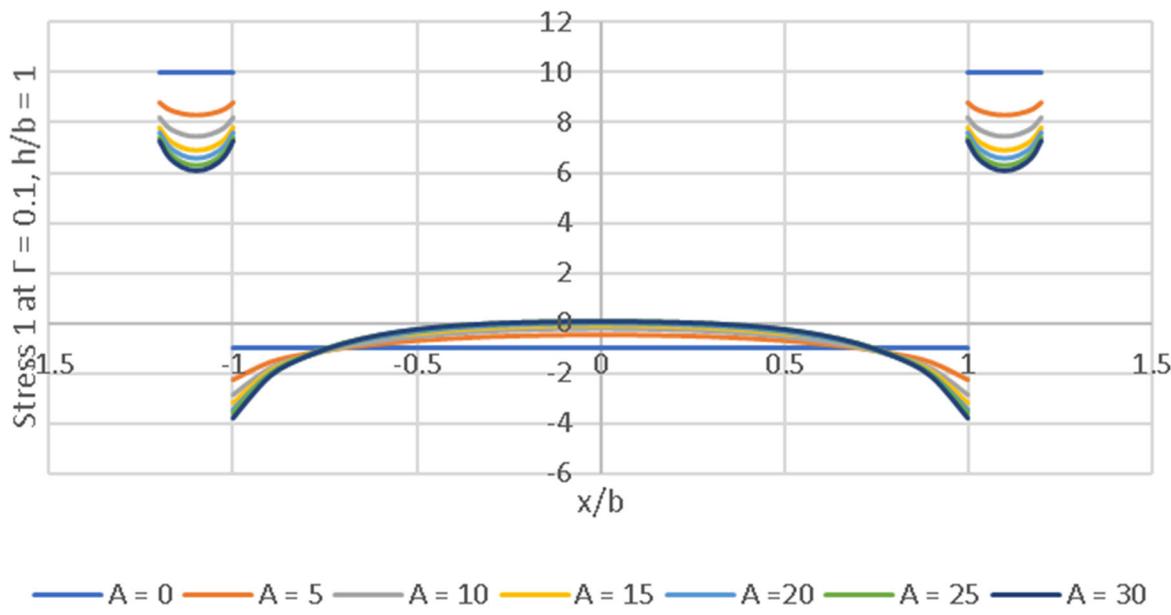
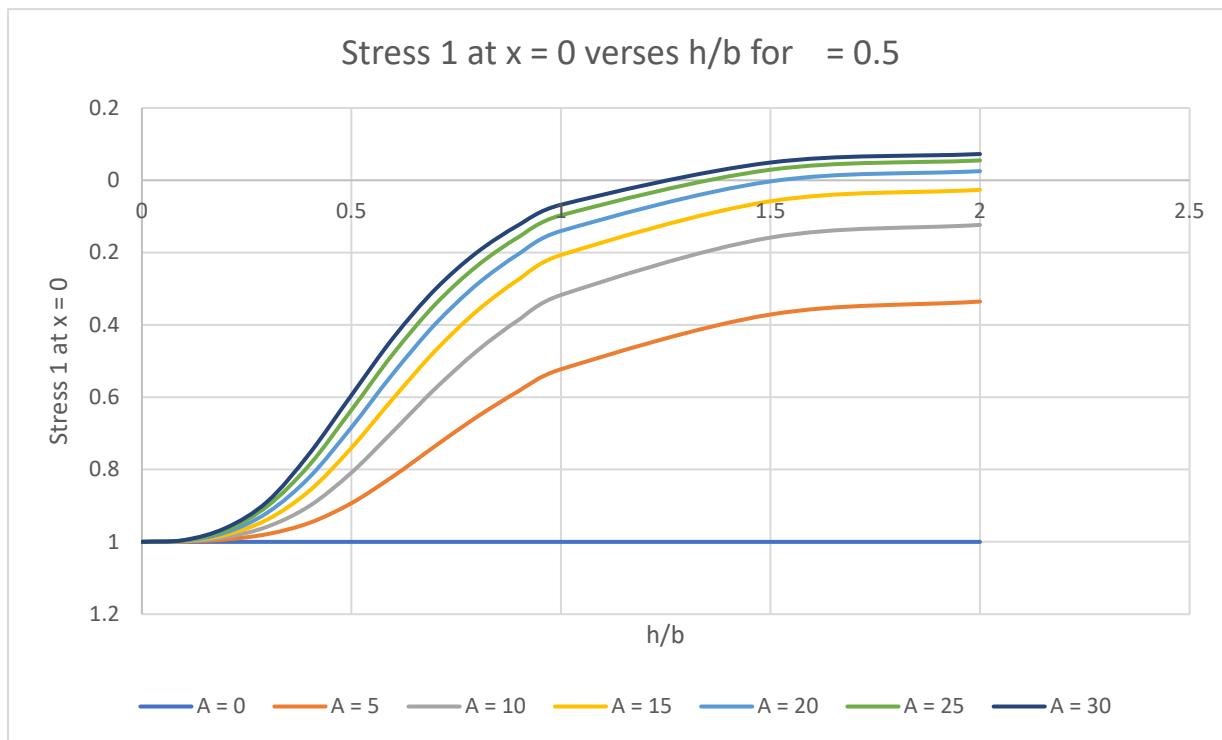
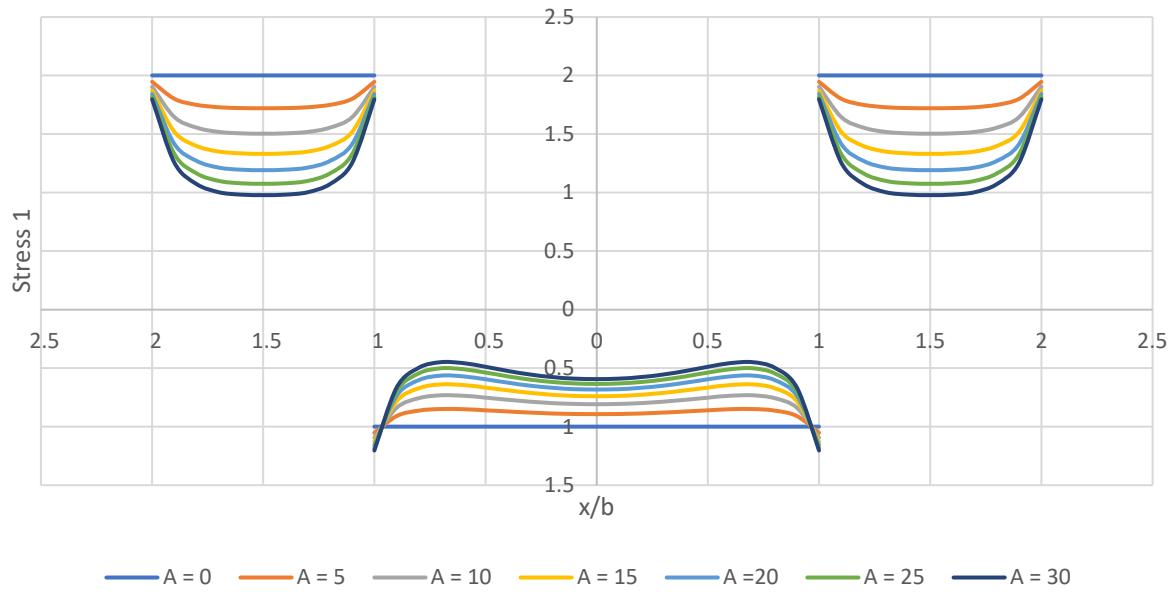


Table 15 Stress 1 at  $x = 0$  verses  $h/b$  for  $\gamma = 0.5$

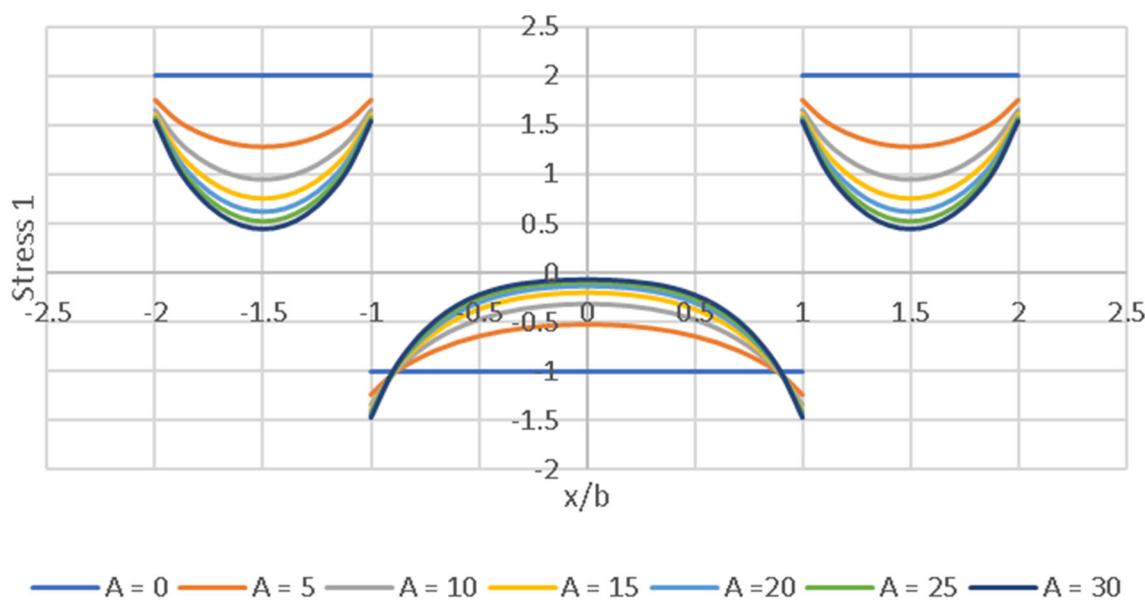
$h/b$	$A = 0$	$A = 5$	$A = 10$	$A = 15$	$A = 20$	$A = 25$	$A = 30$
1E 13	1	1	1	1	1	1	1
0.1	1	0.99917	0.99834	0.99751	0.99668	0.99585	0.99502
0.2	1	0.99339	0.98686	0.98042	0.97406	0.96778	0.96158
0.3	1	0.97832	0.95759	0.93775	0.91873	0.90048	0.88783
0.4	1	0.94726	0.90048	0.85861	0.82085	0.78658	0.75529
0.5	1	0.89286	0.80838	0.73988	0.68311	0.63518	0.59412
0.6	1	0.81831	0.69346	0.6025	0.53332	0.47898	0.43517
0.7	1	0.73496	0.57634	0.47163	0.39787	0.34344	0.30184
0.8	1	0.65401	0.47164	0.36096	0.28779	0.23655	0.19913
0.9	1	0.58233	0.3852	0.27345	0.20325	0.15616	0.12309
1	1	0.5226	0.3171	0.20672	0.14012	0.09698	0.0677
1.5	1	0.37107	0.15826	0.05782	0.00298	0.029318	0.049101
2	1	0.3354	0.12349	0.0264	0.025341	0.055049	0.07268



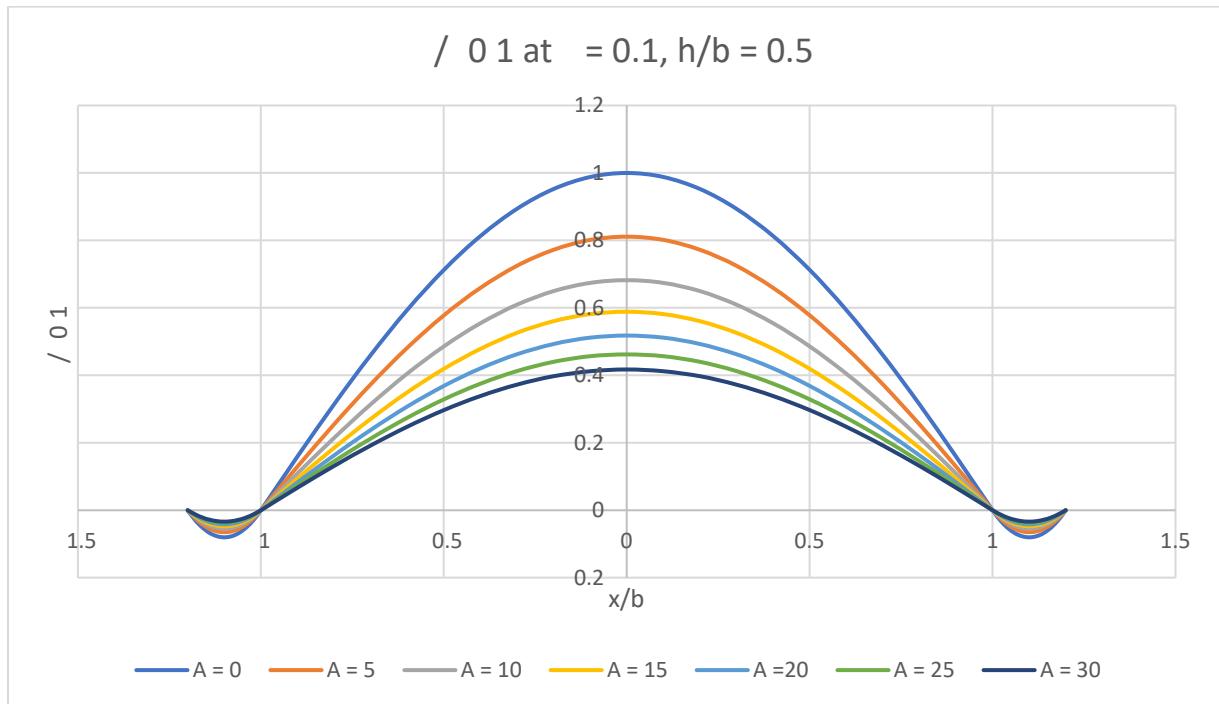
Stress 1 and reaction at  $\Gamma = 0.5$ ,  $h/b = 0.5$



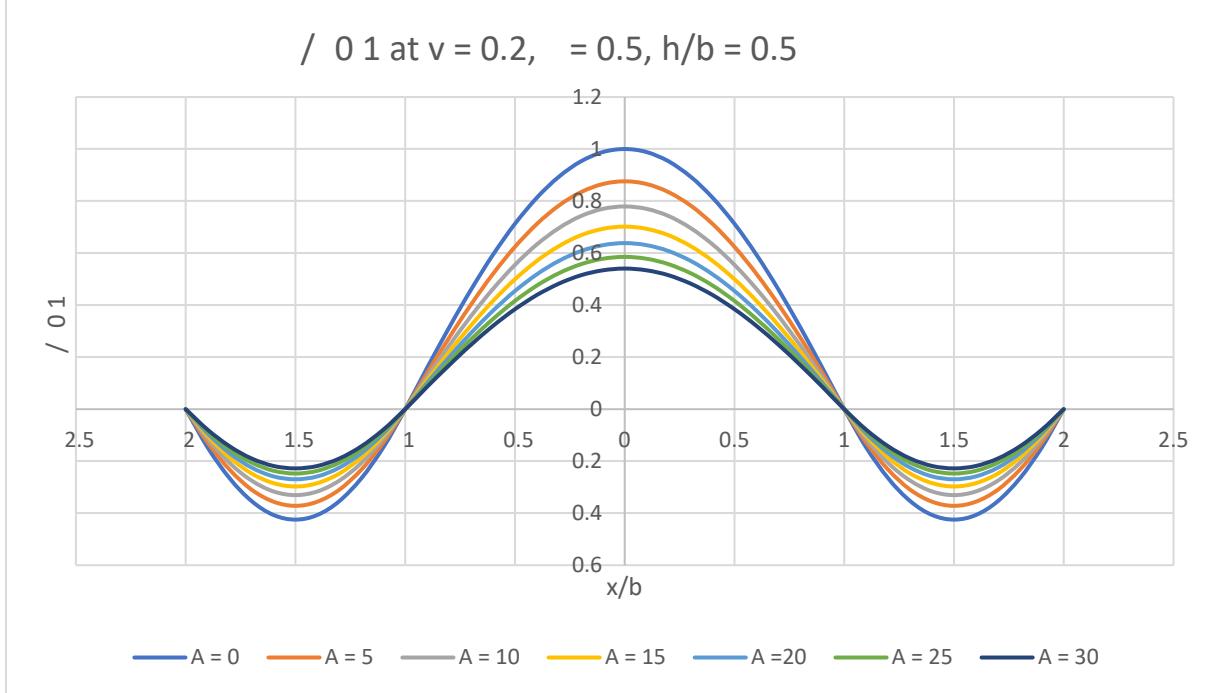
Stress 1 and reaction at  $\Gamma = 0.5$ ,  $h/b = 1$



$$EIv_n = -\frac{\frac{2p}{t} \frac{\sin \alpha_n b}{\alpha_n^2} + \frac{2C2}{t} \sin \alpha_n b}{\alpha_n^3 + \frac{sk_n}{\beta EI}}$$



The deflection does not take into account the moment of inertia of the elastic media, so it is conservative. But for a media such as soils where tension is not allowed it is accurate and useful.



Once  $D$  is found the stresses with  $q = 0$  are:

$$D = -\frac{s\alpha_n v_n}{\beta d_n} = p \left( \frac{sb^3}{\Gamma \beta EI d_n} \right) \frac{\frac{2(1+\Gamma)}{\pi n} + \frac{\pi n}{(1+\Gamma)} \frac{C2}{0.5pb^2}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \sin \alpha_n b$$

$$D = -\frac{s\alpha_n v_n}{\beta d_n} = p \left( \frac{2k_n A}{\Gamma d_n k_n} \right) \frac{\frac{2(1+\Gamma)}{\pi n} + \frac{\pi n}{(1+\Gamma)} \frac{C2}{0.5pb^2}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \sin \alpha_n b$$

$$a_n = -[c_n(1 + 2\alpha_n h) + 2\alpha_n^2 h^2]e^{-2\alpha_n h}$$

$$b_n = [2c_n - 1 + 2\alpha_n h]e^{-2\alpha_n h}$$

$$c_n = 1 + \frac{2\alpha_n^2 h^2 e^{-2\alpha_n h}}{(1 - 2\alpha_n h)e^{-2\alpha_n h} - 1}$$

$$d_n = a_n - b_n - c_n - 1$$

$$k_n = -\frac{a_n + c_n}{d_n}$$

## Rewrite for the excel sheet

$$a_{n1} = a_n e^{\alpha_n y} = -[c_n(1 + 2\alpha_n h) + 2\alpha_n^2 h^2] e^{-\alpha_n h}$$

$$b_{n1} = b_n e^{\alpha_n y} = [2c_n - 1 + 2\alpha_n h] e^{-\alpha_n h}$$

Setting  $p$  to zero we have:

and  $x = 0$  we have, note: the maximum moment may not be at  $x = 0$  depending on  $A$

Table 16 Rm2 verses h/b for  $\gamma = 0.1$

$h/b$	$A = 0$	$A = 5$	$A = 10$	$A = 15$	$A = 20$	$A = 25$	$A = 30$
1E 13	1	1	1	1	1	1	1
0.1	0.999966	0.996415	0.9928954	0.9894054	0.985945	0.982514	0.9791119
0.2	0.998738	0.974258	0.9511304	0.9292434	0.908496	0.888799	0.8700707
0.3	0.995767	0.930134	0.8733674	0.8237493	0.779981	0.741062	0.706209
0.4	0.991541	0.869724	0.7757292	0.7009294	0.639936	0.589208	0.5463235
0.5	0.985873	0.80228	0.676959	0.5859352	0.516802	0.462495	0.4186948
0.6	0.978705	0.737807	0.5912057	0.4926927	0.422002	0.368846	0.3274474
0.7	0.970476	0.682775	0.5238924	0.4233886	0.354236	0.303845	0.2655587
0.8	0.961877	0.639248	0.474148	0.3742339	0.307499	0.25992	0.2243827
0.9	0.953547	0.60645	0.4386096	0.34016	0.275736	0.230483	0.1970753
1	0.945923	0.582458	0.4136744	0.3167775	0.254243	0.210758	0.1789087
1.5	0.921602	0.533563	0.3662073	0.2736838	0.215392	0.175559	0.1467868
2	0.912914	0.524343	0.3582264	0.2667938	0.209354	0.170187	0.1419458

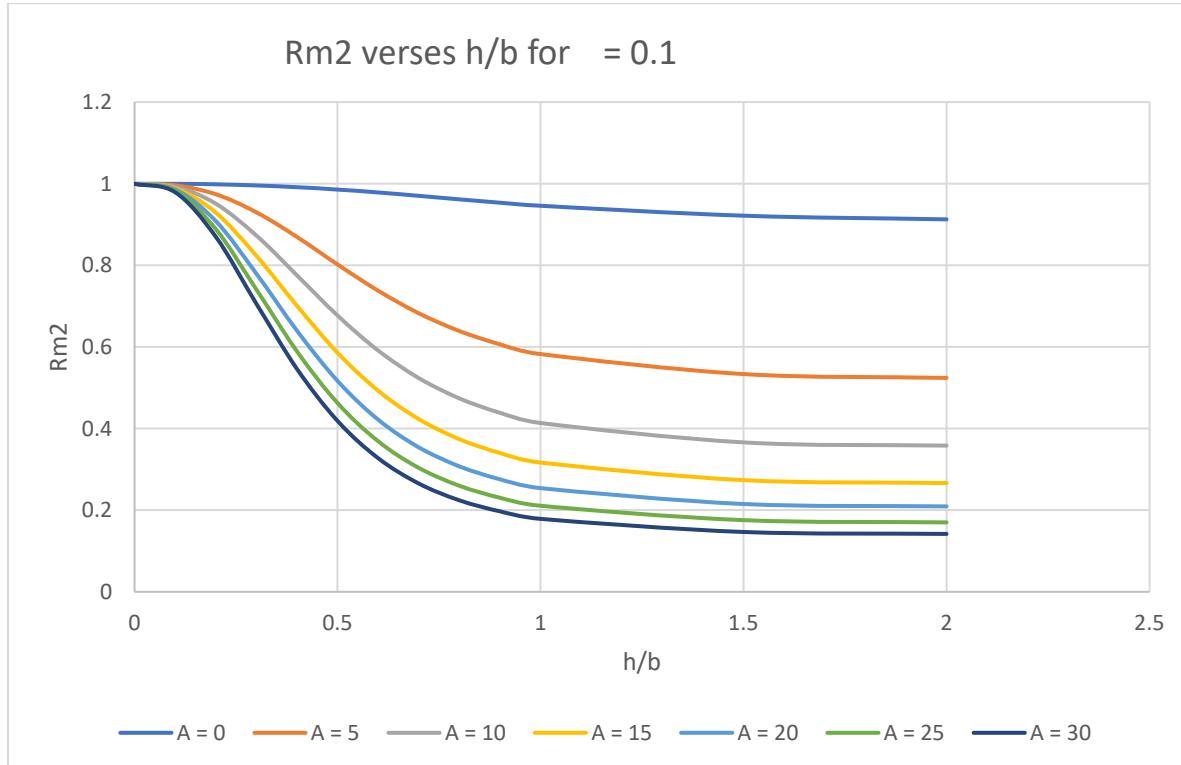
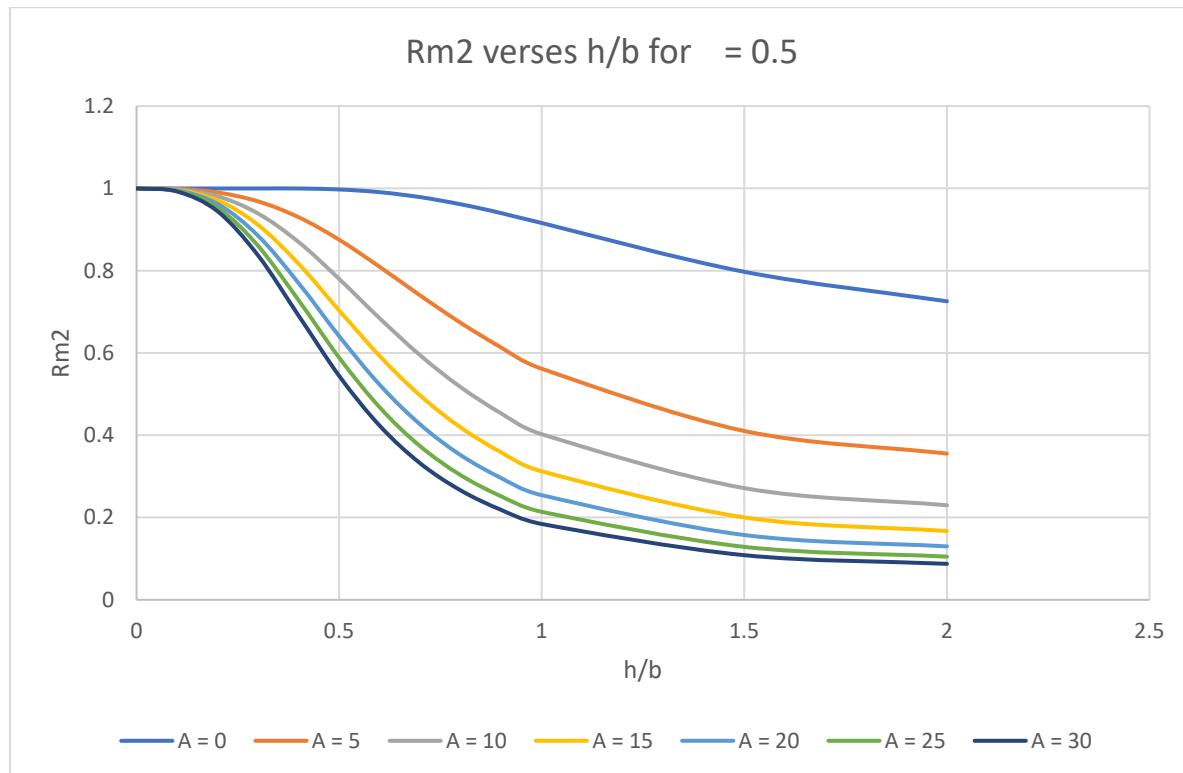


Table 17 Rm2 verses h/b for  $\gamma = 0.5$

$h/b$	$A = 0$	$A = 5$	$A = 10$	$A = 15$	$A = 20$	$A = 25$	$A = 30$
1E 13	1	1	1	1	1	1	1
0.1	1	0.998757	0.9975162	0.9962789	0.995045	0.993814	0.9925855
0.2	1.000001	0.990257	0.9807024	0.9713314	0.962139	0.953119	0.944268
0.3	1.000039	0.96851	0.9389301	0.9111224	0.884931	0.860219	0.8368634
0.4	0.999751	0.930068	0.8695831	0.8165785	0.76974	0.728046	0.6906872
0.5	0.997393	0.875328	0.7801027	0.7037239	0.641087	0.588779	0.5444328
0.6	0.990813	0.809423	0.6842776	0.5927215	0.522828	0.467719	0.4231498
0.7	0.978815	0.739858	0.5943384	0.4964561	0.426134	0.373185	0.3318897
0.8	0.96151	0.67314	0.5168041	0.4188233	0.351728	0.302942	0.2658966
0.9	0.939967	0.613217	0.4532308	0.3584788	0.295922	0.251598	0.2185886
1	0.915694	0.561643	0.4024858	0.3122782	0.25433	0.214044	0.1844648
1.5	0.797388	0.410392	0.2714711	0.2004163	0.157488	0.128878	0.1085304
2	0.725583	0.355456	0.2298526	0.1671051	0.129728	0.10507	0.0876748



Thus, at  $q = 0$  and  $x = b^*$  we have

$$\begin{aligned}
 Rv2 &= \frac{\int \sigma_y}{-qb} \\
 &= \sum_{n=1}^{\infty} \left( \frac{\frac{W_n}{q} \frac{(\pi n)^2}{(1+\Gamma)^2} + \frac{2}{\Gamma} \frac{\pi n}{(1+\Gamma)^2} \sin \alpha_n b - \frac{1}{\Gamma} 2k_n A \frac{C2}{0.5qb^2} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \right) \sin \alpha_n b^* \dots \dots \dots .128
 \end{aligned}$$

Table 18 Rv 2 verses h/b for  $= 0.1$ ,  $b = b^*$

h/b	A = 0	A = 5	A = 10	A = 15	A = 20	A = 25	A = 30
1E 13	0.999755	0.999755	0.9997549	0.9997549	0.999755	0.999755	0.9997549
0.1	0.985215	0.965852	0.9466779	0.9276897	0.908885	0.890262	0.8718173
0.2	0.966849	0.908325	0.85337	0.8016819	0.752991	0.707057	0.6636643
0.3	0.951827	0.861758	0.7846966	0.7180917	0.660015	0.608982	0.5638328
0.4	0.941455	0.829834	0.7444883	0.6772215	0.622917	0.578217	0.5408259
0.5	0.934249	0.80844	0.7225693	0.6601956	0.612811	0.575573	0.545519
0.6	0.92902	0.793924	0.7106718	0.6539237	0.612559	0.580927	0.5558527
0.7	0.925027	0.783905	0.703978	0.6519228	0.614939	0.587052	0.5650969
0.8	0.92186	0.776931	0.7000296	0.6514768	0.617498	0.592041	0.5720263
0.9	0.919299	0.77207	0.6976028	0.6515437	0.619588	0.595708	0.5769168
1	0.917218	0.768686	0.6960651	0.6517443	0.621145	0.598296	0.5802903
1.5	0.911498	0.762202	0.693493	0.6524328	0.624248	0.603175	0.586491
2	0.909658	0.761059	0.6931225	0.6525952	0.624776	0.60396	0.5874608

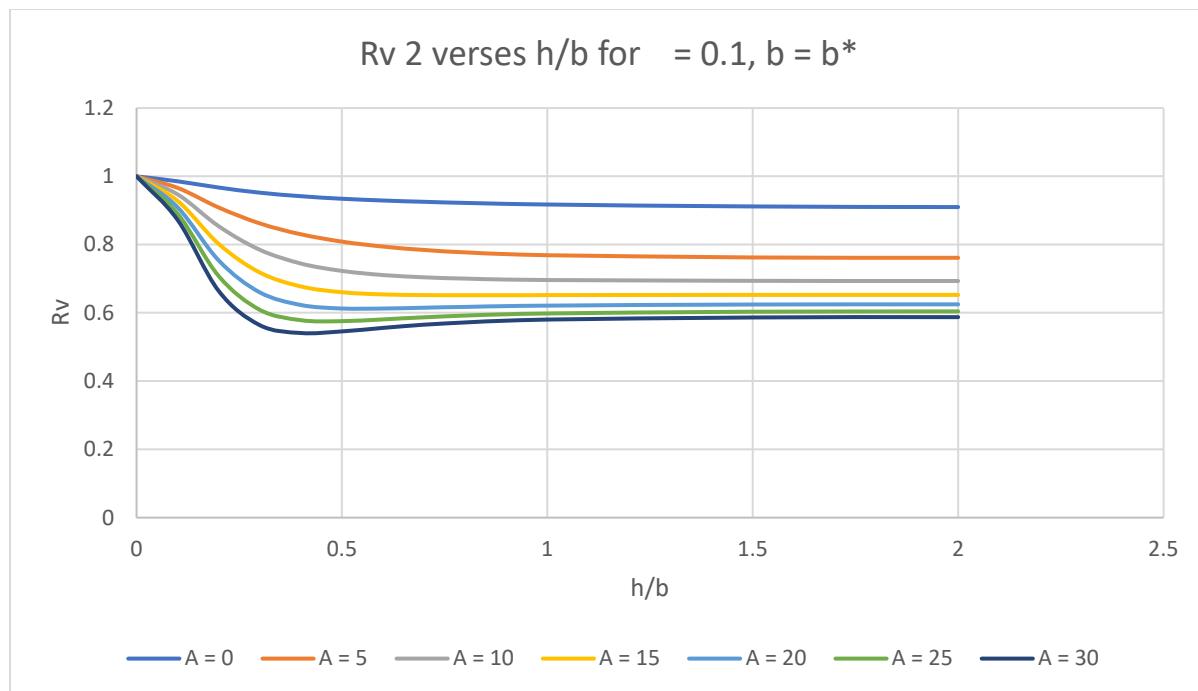
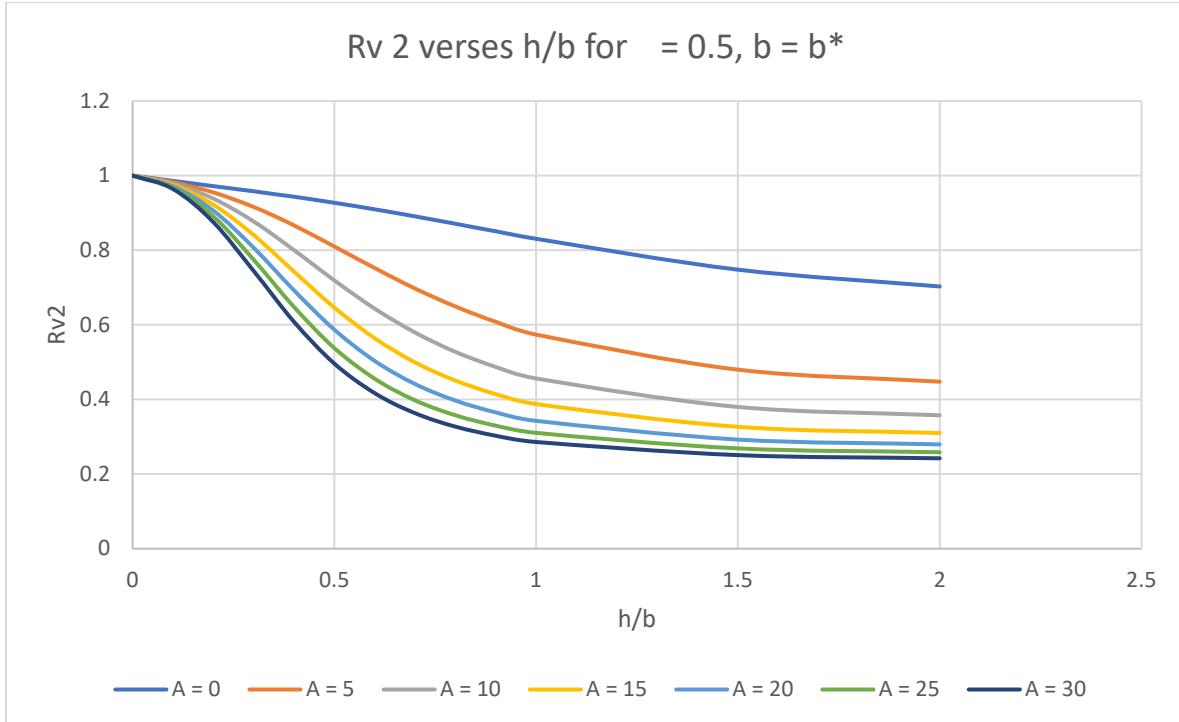


Table 19 Rv 2 verses h/b for  $\gamma = 0.5$ ,  $b = b^*$

$h/b$	A = 0	A = 5	A = 10	A = 15	A = 20	A = 25	A = 30
1E 13	0.999909	0.999909	0.9999088	0.9999088	0.999909	0.999909	0.9999088
0.1	0.985886	0.982226	0.9785781	0.9749416	0.971317	0.967703	0.9641004
0.2	0.971833	0.954817	0.9381846	0.9219251	0.906027	0.890479	0.8752706
0.3	0.957787	0.91597	0.8769891	0.8405777	0.806502	0.774554	0.7445505
0.4	0.943192	0.866426	0.8002856	0.7427567	0.692299	0.647717	0.6080664
0.5	0.927158	0.809895	0.71888	0.646254	0.587005	0.537784	0.4962732
0.6	0.90952	0.751955	0.6432791	0.563794	0.50313	0.455309	0.4166402
0.7	0.890587	0.697399	0.5791212	0.4990945	0.441241	0.397398	0.362976
0.8	0.870787	0.649045	0.5275126	0.4504011	0.396893	0.357444	0.3270621
0.9	0.850617	0.607917	0.4871676	0.414316	0.365238	0.329719	0.3026859
1	0.830625	0.57386	0.4560401	0.387619	0.342476	0.310195	0.2857984
1.5	0.747966	0.479726	0.3798913	0.326481	0.292546	0.268689	0.250766
2	0.702596	0.447731	0.3573919	0.3097253	0.279535	0.258282	0.2422599



And the stress at  $x = 0$  we have

And the reaction at  $x = b + t$  we have

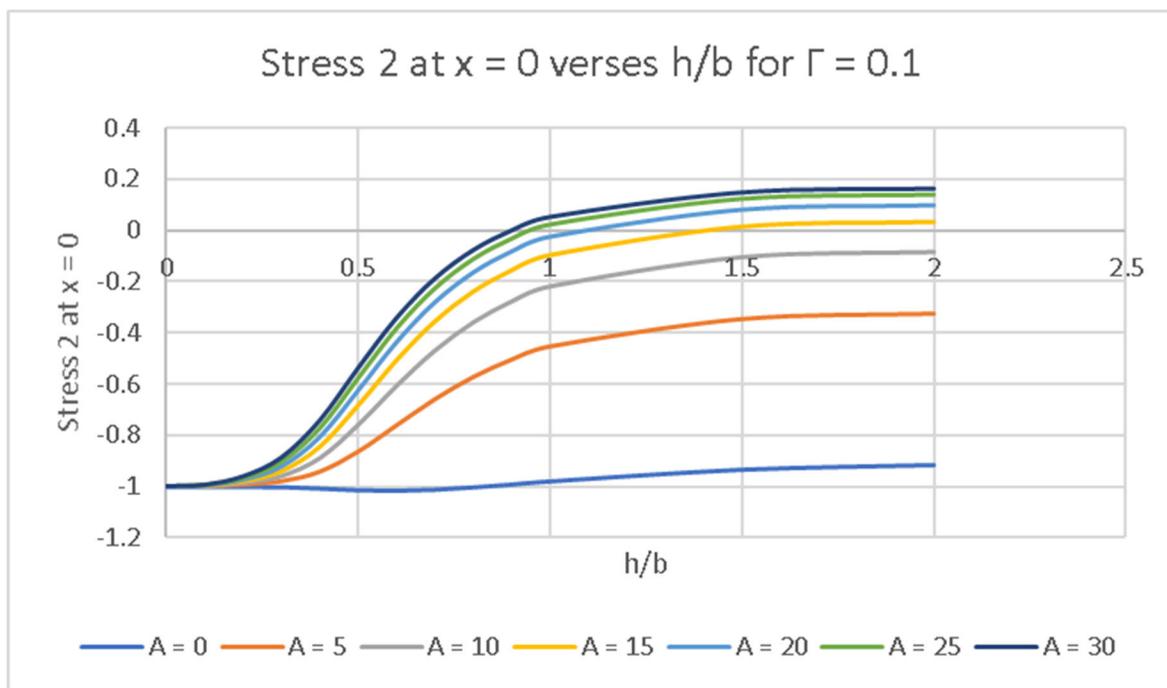
Please note the stress with  $\omega_n$  is not defined at  $\pm b$  because it reverses from compression to tension for the reaction. So, for the shear and stress depending on how high  $n$  is taken for conversion calculations, the stress can at  $x = \pm 0.999b$  and not  $b$ . Also, this happens for  $h/b$  is small. So, for example when  $h/b = .001$  with nominal  $A$  the live load stress on the beam should match the dead

load stress on the beam at  $x/b = 0.99$ . Attached on my web page is an excel spreadsheet. The best is to have a general program for the solution where  $n$  can be taken to a very high number. The spreadsheet goes to  $n = 5000$  so it can be posted on my webpage for a smaller file size. It is recommended to extend to at least  $n = 20000$ .

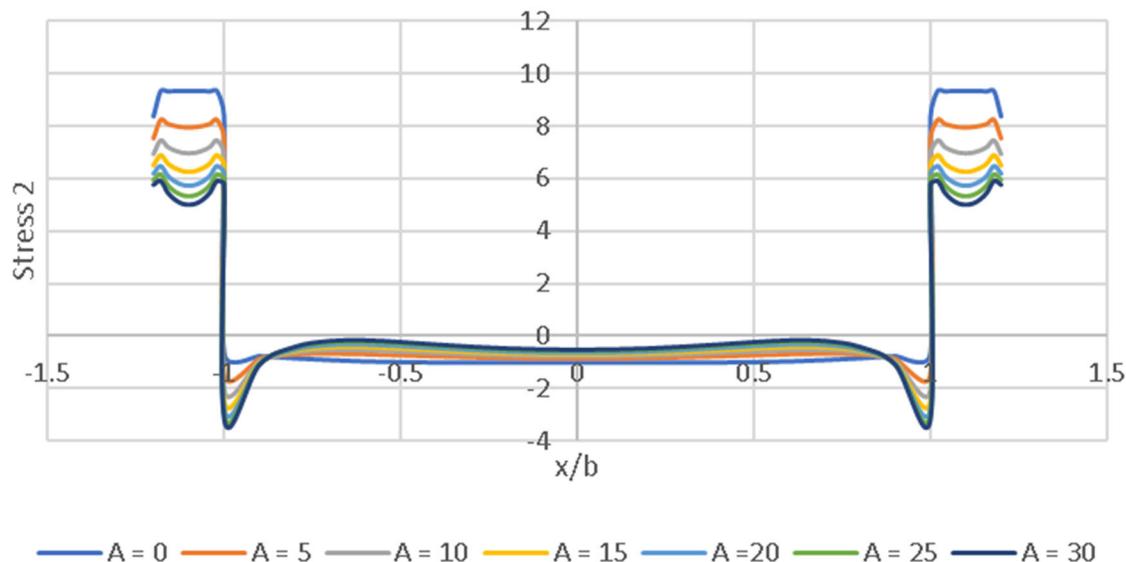
Again, we assume the beam is glued to the mass media and it can experience a vertical stress and a tension stress.

Table 20 Stress 2 at  $x = 0$  verses  $h/b$  for  $\Gamma = 0.1$

$h/b$	$A = 0$	$A = 5$	$A = 10$	$A = 15$	$A = 20$	$A = 25$	$A = 30$
1E 13	0.99932	0.99932	0.99932	0.99932	0.99932	0.99932	0.999321
0.1	0.99938	0.99855	0.99772	0.996889	0.99606	0.99523	0.994408
0.2	0.99934	0.99279	0.98631	0.97991	0.97359	0.96734	0.961172
0.3	1.00008	0.97922	0.95932	0.94026	0.92199	0.90444	0.887544
0.4	1.00565	0.94167	0.88908	0.844545	0.80599	0.77203	0.741693
0.5	1.01221	0.86273	0.75919	0.682763	0.62371	0.57649	0.537687
0.6	1.01401	0.7596	0.60708	0.506251	0.43513	0.38258	0.342362
0.7	1.00978	0.65747	0.46918	0.354556	0.27903	0.22653	0.188635
0.8	1.00104	0.57045	0.35958	0.238664	0.16285	0.11256	0.077931
0.9	0.98992	0.50218	0.27816	0.155028	0.08053	0.03282	0.001195
1	0.97814	0.45104	0.21972	0.09628	0.02347	0.021962	0.0511994
1.5	0.93391	0.34371	0.10557	0.0148063	0.082425	0.122414	0.1464632
2	0.91639	0.32277	0.08579	0.0330934	0.099373	0.138209	0.1612637



Stress 2 and reaction at  $\Gamma = 0.1$ ,  $h/b = 0.5$



Stress 2 and reaction at  $\Gamma = 0.1$ ,  $h/b = 1$

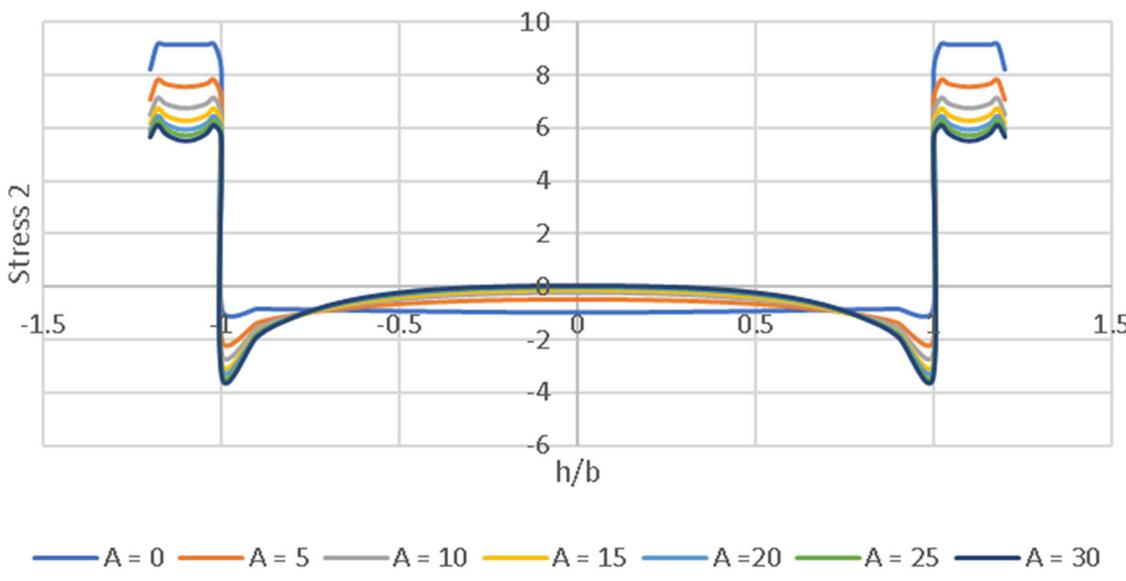
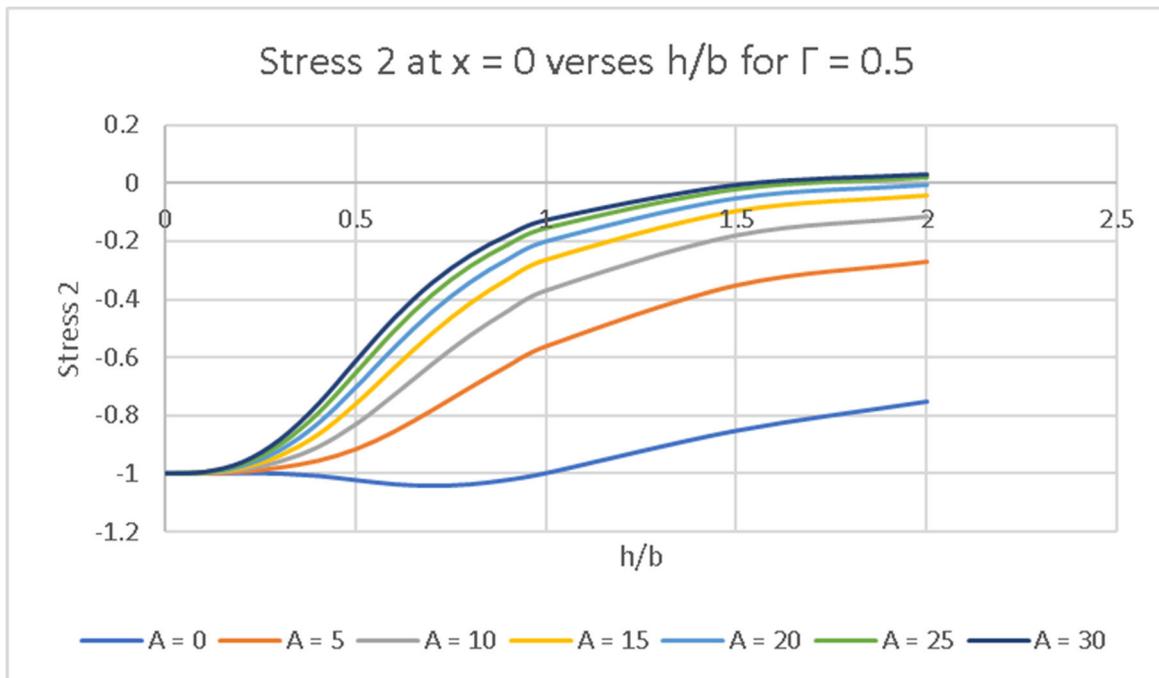
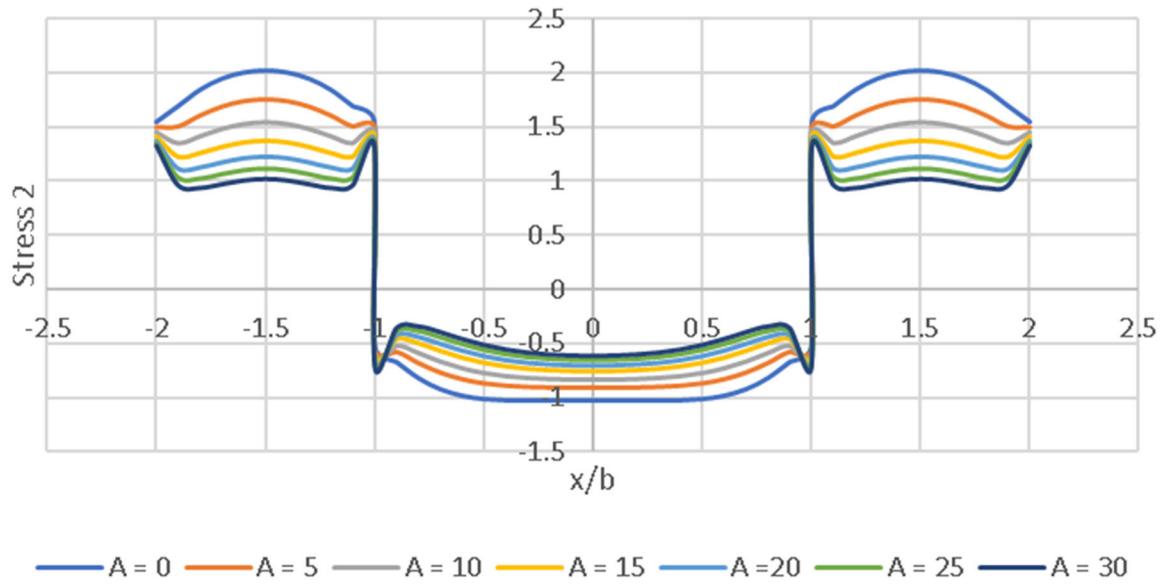


Table 21 Stress 2 at  $x = 0$  verses  $h/b$  for  $\Gamma = 0.5$

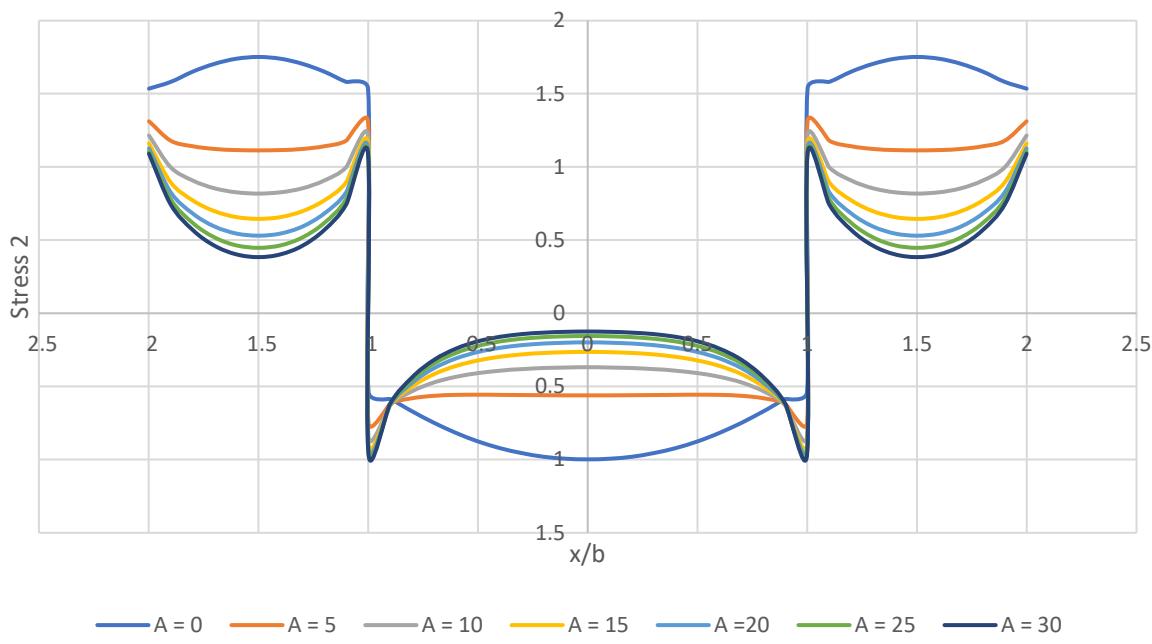
$h/b$	$A = 0$	$A = 5$	$A = 10$	$A = 15$	$A = 20$	$A = 25$	$A = 30$
1E 13	0.99989	0.99989	0.99989	0.99989	0.99989	0.99989	0.99989
0.1	0.99993	0.999094	0.998263	0.997433	0.996604	0.99578	0.994951
0.2	0.99988	0.99327	0.986744	0.980302	0.973943	0.96767	0.961469
0.3	1.00049	0.978746	0.957953	0.938047	0.918969	0.90066	0.883083
0.4	1.00831	0.954998	0.907667	0.865283	0.827044	0.79232	0.760609
0.5	1.0234	0.914948	0.829169	0.759415	0.70143	0.65236	0.610211
0.6	1.03734	0.854884	0.728687	0.63611	0.565232	0.50918	0.463698
0.7	1.04297	0.780886	0.622444	0.516696	0.44133	0.38504	0.341485
0.8	1.03763	0.702642	0.523781	0.413591	0.339515	0.28668	0.247348
0.9	1.02208	0.627844	0.438962	0.329914	0.25993	0.21183	0.177128
1	0.99879	0.560712	0.369056	0.263956	0.198934	0.15556	0.125089
1.5	0.85212	0.351264	0.179459	0.096764	0.0504	0.02211	0.003957
2	0.75185	0.270081	0.115318	0.04363	0.004828	0.017936	0.0318659



Stress 2 and reaction at  $\Gamma = 0.5$ ,  $h/b = 0.5$

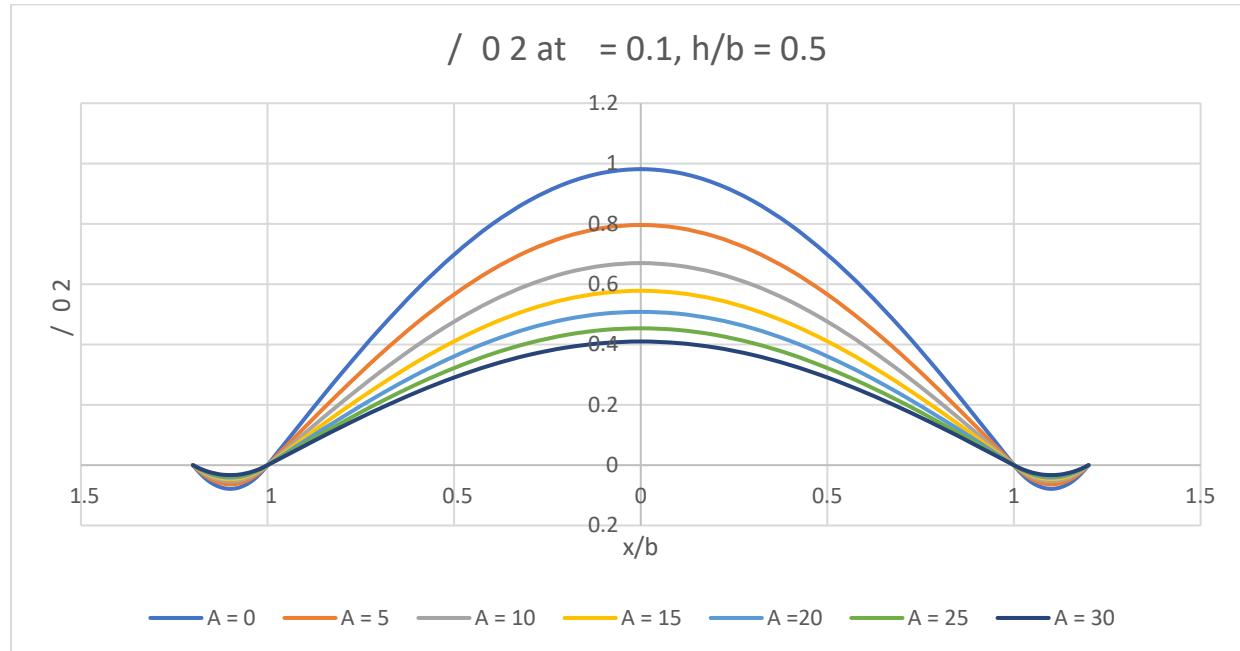


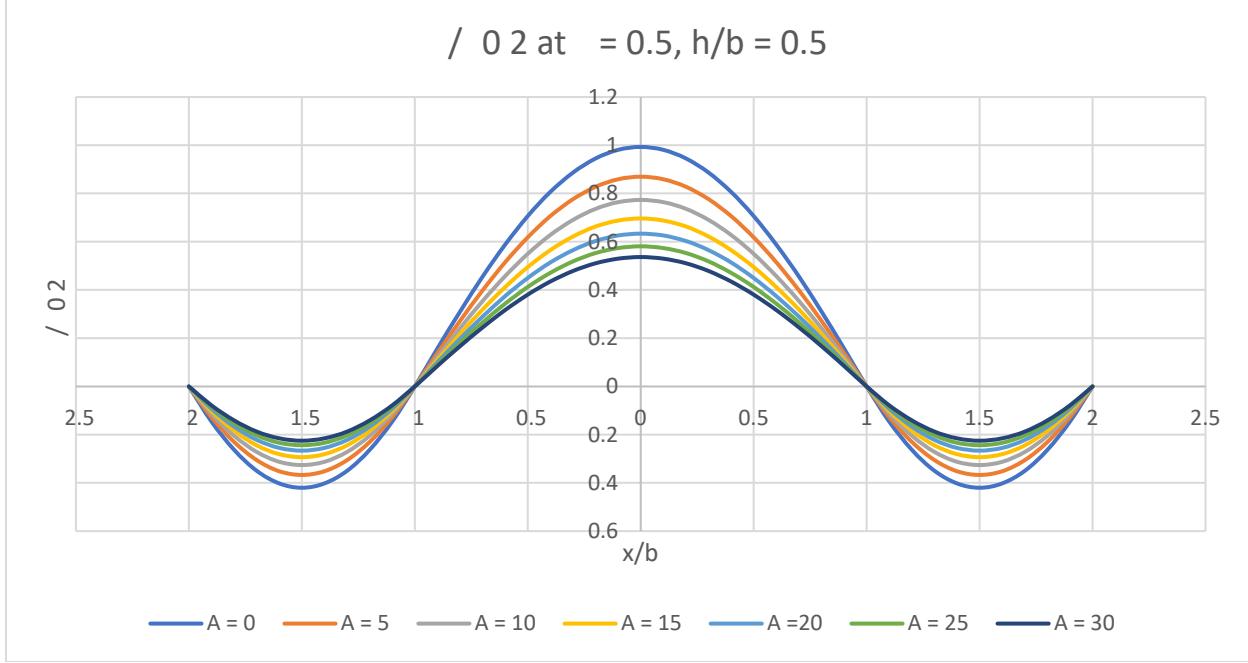
Stress 2 and reaction at  $\Gamma = 0.5$ ,  $h/b = 1$



$$EIv_n = -\frac{\frac{W_n}{\alpha_n} + 2q \frac{b}{t(b+t)} \frac{\sin \alpha_n b}{\alpha_n^2} + \frac{2C2}{t} \sin \alpha_n b}{\alpha_n^3 + \frac{sk_n}{\beta EI}}$$

The deflection does not take into account the moment of inertia of the elastic media, so it is conservative. But for a media such as soils where tension is not allowed it is accurate and useful.





We rewrite  $2\beta kn$  from before as

$$2\beta k_n = - \frac{a_n + c_n}{\frac{d_n}{2\beta}}$$

$$\frac{d_n}{2\beta} = \frac{a_n - b_n - c_n - 1}{2}$$

And we have

$$W_n = -\frac{(a_n + c_n)(w_{n1} - w_{n2} - w_{n3})w_n}{a_n - b_n - c_n - 1} + (w_{n1} + w_{n3})w_n$$

We note  $\rho$  does not enter the equations

The total reaction at the column is  $2(b + t)p + 2(b + t)q$ .

Once  $D$  is found the stresses for  $p = 0$  are

$$\begin{aligned} \sigma_y = & -\sum_{n=\varphi}^{\infty} \cos \alpha_n x \left[ \left( a_n D + w_{n1} w_n + (b_n D + w_{n2} w_n) \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right) e^{\alpha_n y} \right. \\ & \left. + \left( c_n D + w_{n3} w_n + D \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} + \frac{2q \sin \alpha_n b}{\Gamma} \frac{1}{n\pi} \right] \quad -b < x < b \dots \dots 133 \end{aligned}$$

$$\begin{aligned}\sigma_x = k_0 \sum_{n=0}^{\infty} \cos \alpha_n x & \left[ \left( a_n D + w_{n1} w_n + (b_n D + w_{n2} w_n) \frac{\pi n}{(1+\Gamma)b} y \right) e^{\alpha_n y} \right. \\ & + \left( c_n D + w_{n3} w_n + D \frac{\pi n}{(1+\Gamma)b} y \right) e^{-\alpha_n y} - \frac{2q \sin \alpha_n b}{\Gamma n \pi} \Big] \\ & + 2k_0 \sum_{n=0}^{\infty} \cos \alpha_n x [ (b_n D + w_{n2} w_n) e^{\alpha_n y} - D e^{-\alpha_n y}] \quad -b < x < b \dots \dots 134\end{aligned}$$

$$\tau_{xy} = \sum_{n=\varphi}^{\infty} \sin \alpha_n x \left[ \left( (a_n + b_n)D + w_{n1}w_n + w_{n2}w_n + (b_n D + w_{n2}w_n) \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right) e^{\alpha_n y} + \left( -(c_n - 1)D - w_{n3}w_n - D \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} \right] \quad -b < x < b \dots \dots 135$$

Where:

$$W1_n = (w_{n1} - w_{n2} - w_{n3})w_n$$

$$a_n = -[c_n(1 + 2\alpha_n h) + 2\alpha_n^2 h^2]e^{-2\alpha_n h}$$

$$b_n = [2c_n - 1 + 2\alpha_n h]e^{-2\alpha_n h}$$

$$c_n = 1 + \frac{2\alpha_n^2 h^2 e^{-2\alpha_n h}}{(1 - 2\alpha_n h)e^{-2\alpha_n h} - 1}$$

$$d_n = a_n - b_n - c_n - 1$$

$$k_n = -\frac{a_n + c_n}{d_n}$$

$$W_n = -\frac{(a_n + c_n)(w_{n1} - w_{n2} - w_{n3})w_n}{(a_n - b_n - c_n - 1)} + (w_{n1} + w_{n3})w_n$$

$$w_{n1} = -w_{n3}(1 + 2\alpha_n h)e^{-2\alpha_n h} - (1 + \alpha_n h)e^{-\alpha_n h}$$

$$w_{n2} = 2w_{n3}e^{-2\alpha_n h} + e^{-\alpha_n h}$$

$$w_{n3} = \frac{\alpha_n h e^{-\alpha_n h}}{(1 - 2\alpha_n h)e^{-2\alpha_n h} - 1}$$

Rewrite for the excel sheet

$$\sigma_y = -\sum_{n=\varphi}^{\infty} \cos \alpha_n x \left[ \left( a_{n1}D + w_{n11}w_n + (b_{n1}D + w_{n21}w_n) \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) e^{-\alpha_n(h-y)} + \left( c_n D + w_{n3}w_n + D \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} + \frac{2q}{\Gamma} \frac{\sin \alpha_n b}{n\pi} \right] \quad -b < x < b \dots \dots 137$$

$$\sigma_x = k_0 \sum_{n=\varphi}^{\infty} \cos \alpha_n x \left[ \left( a_{n1}D + w_{n11}w_n + (b_{n1}D + w_{n21}w_n) \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) e^{-\alpha_n(h-y)} + \left( c_n D + w_{n3}w_n + D \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} - \frac{2q}{\Gamma} \frac{\sin \alpha_n b}{n\pi} \right] + 2k_0 \sum_{n=\varphi}^{\infty} \cos \alpha_n x \left[ (b_{n1}D + w_{n21}w_n) e^{-\alpha_n(h-y)} - D e^{-\alpha_n y} \right] \quad -b < x < b \dots 138$$

$$\tau_{xy} = \sum_{n=\varphi}^{\infty} \sin \alpha_n x \left[ \left( a_{n1} + b_{n1} \right) D + w_{n11}w_n + w_{n21}w_n + (b_{n1}D + w_{n21}w_n) \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) e^{-\alpha_n(h-y)} + \left( -(c_n - 1)D - w_{n3}w_n - D \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} \right] \quad -b < x < b \dots \dots 139$$

$$a_{n1} = a_n e^{\alpha_n y} = -[c_n(1 + 2\alpha_n h) + 2\alpha_n^2 h^2] e^{-\alpha_n h}$$

$$b_{n1} = b_n e^{\alpha_n y} = [2c_n - 1 + 2\alpha_n h] e^{-\alpha_n h}$$

$$w_{n11} = w_{n1} e^{\alpha_n y} = -w_{n3}(1 + 2\alpha_n h)e^{-\alpha_n h} - (1 + \alpha_n h)$$

$$w_{n21} = w_{n2} e^{\alpha_n y} = 2w_{n3}e^{-\alpha_n h} + 1$$

### Example 3:

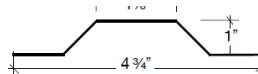
In example 1 we used a 30 ft wall to approximate an infinite wall. When redoing the calculation with  $h/b = 15$  and  $A = 50.85$   $R_m$  is identical to the example = 0.084496 at  $x = 0.615b$  and  $R_v = 0.025824$  verses 0.025824 at  $x = 0.3787b$  is also identical.

### Example 4:

An underground tunnel made from timber 30 ft deep 8 ft wide. The ribs are 12x12 actual 11.25" x 11.25" timber spaces at 5 ft on center. It is required to investigate the 5 ft lagging spanning from rib to rib. It is a simple span condition. We try the lagging is 4x12 actual 3.5" x 11.25"  $E_{s,eff} = 1111$  psi,  $1/\beta = E_s/(1-\nu^2) = 1221$  psi.  $t/b = 0.5$ .  $p = 120 * 30 * 11.25 / 12 = 3375$  lbs/ft.  $K_0 = 0.428571$  for  $\gamma = 0.3$ .  $x/p = 0.23174$  at  $x = 0$  and  $y = 0$  so we have compression everywhere and the solution is applicable for soils.  $A = 2.56303$  and  $h/b = 12$ . Thus  $R_m1 = 0.649475$  thus  $M_{max} = 3375 * 0.649475 * 5^2 / 8 = 6849.93$  lbs ft.  $S_x$  required for  $F_b = 900 * 1.1 = 990$  psi for flatwise condition for 4x12 timber is  $6849.93 * 12 / 990 = 83.03$  in<sup>3</sup> >  $11.25 * 3.5^2 / 6 = 22.97$  in<sup>3</sup> so it is no good. We try 10x12 lagging  $x/p = 0.41282$  at  $x = 0$  and  $y = 0$  so we have compression everywhere and the solution is applicable for soils.  $A = 0.13885$  and  $h/b = 12$ . Thus  $R_m1 = 0.972179$  thus  $M_{max} = 3375 * 0.972179 * 5^2 / 8 = 10253.45$  lbs ft.  $S_x$  required for  $F_b = 900 * 1.1 = 990$  psi for flatwise condition for 10x12 timber is  $10253.45 * 12 / 990 = 124.28$  in<sup>3</sup> <  $11.25 * 9.25^2 / 6 = 160.43$  in<sup>3</sup>. At  $x/b = 0.69167$   $R_v = 0.672171$  or  $V = 1.5 * 3375 * 0.672171 * 2.5 / (11.25 * 9.25) = 84.12$  psi < 95 psi so the shear is OK. Thus, 10x12 lagging is good. Finding the strength of the ribs is not part of this example.

### Example 5:

A semicircular culvert measures 15 ft to the above grade at the center of the semicircle. The radius is 10 ft so at the bottom it is 20 width. Soil weight is 120 pcf  $K_0 = 0.5$ . We assume 7 ft lateral for the horizontal pressure and 14.28 ft vertical pressure with  $15+3 = 18$  ft overburden. The culver has ribs made from bended pipe at 2.4 ft O.C. and the lagging is corrugated sheet metal. The 2.4 ft was selected to have everything in compression in the soil. We are to check the strength of the corrugated sheet metal. We assume the corrugated sheet metal is simply supported on the ribs.



SECTION PROPERTIES									
GAUGE	NOM. THICK (IN.)	WT. (PLF)	Fy (KSI)	TOP IN COMPRESSION			BOTTOM IN COMPRESSION		
				I <sub>x</sub> (in. <sup>4</sup> )	S <sub>x</sub> (in. <sup>3</sup> )	F <sub>b</sub> (KSI)	I <sub>x</sub> (in. <sup>4</sup> )	S <sub>x</sub> (in. <sup>3</sup> )	F <sub>b</sub> (KSI)
18	.049	0.97	33.0	.0471	.0949	19.8	.0471	.0853	19.8

So  $E_s \text{eff} = 1111 \text{ psi}$ ,  $1/\beta = E_s/(1-\nu^2) = 1221 \text{ psi}$  for  $\nu = 0.3$ .  $I = .0471$   $b = 1.2 \text{ ft}$   $E_s = 29000 \text{ ksi}$  thus  $A = 16.02$ ,  $t/b = 0.5$  and  $s = 4.75"$   $p = 120 * 18 = 2160 \text{ lbs/ft}$ .  $Rm1 = 0.210373$   $Rv1 = 0.441536$  thus  $\text{Max} = 2.160 * 0.210373 * 2.4^2 / 8 = 0.3272 \text{ kip ft}$ . The allowable section modulus =  $0.3272 * 12 / 19.8 = 0.918 \text{ in}^3$  per 12 inch strip for  $s = 4.75$  the available section modulus becomes  $0.0853 * 12 / 4.75 = 0.21 \text{ in}^3 > 0.918 \text{ in}^3$  so the flexure is OK. The shear  $V = 2.160 * 0.441536 * 2.4 / 2 = 1.1445 \text{ Kips}$ .

The area =  $0.97/490 = 0.002 \text{ ft}^2$  or  $0.285 \text{ in}^2$  so  $\text{fv} = 1.1445/0.285 = 4.01 \text{ ksi} < 0.4*33 = 13.2 \text{ ksi}$   
 OK. Checking the lateral pressure, we have at the bottom  $(15+7) * 120 * K_0 = 1320 \text{ lbs/ft} < 2160$ .  
 Thus, the vertical pressure controls the design of the corrugated sheet metal. Thus, the  
 corrugated sheet metal checks and is OK. The design of the ribs made out of pipes is not part of  
 the example.

#### Closed form solution #4 fixed span beam with height $h$ :

Setting the rotation to be zero at  $x = \pm b$  for  $q = 0$  we have.

$$0 = \sum_{n=1}^{\infty} \frac{2(1+\Gamma) \frac{\sin \alpha_n b}{\pi n} + \frac{C2}{0.5pb^2} \frac{\pi n}{(1+\Gamma)} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \sin \alpha_n b$$

$$0 = \sum_{n=1}^{\infty} \frac{2(1+\Gamma) \frac{\sin \alpha_n b}{\pi n} \sin \alpha_n b + \sum_{n=1}^{\infty} \frac{\frac{C2}{0.5pb^2} \frac{\pi n}{(1+\Gamma)} \sin \alpha_n b \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A}$$

And  $Rm1+$  is, note: the maximum positive moment may not be at  $x = 0$  depending on  $A$

$$Rm1+ = \frac{M +}{\frac{pb^2}{6}} = 3 \left[ \frac{1}{\Gamma} \sum_{n=1}^{\infty} \frac{4 \sin \alpha_n b - 4k_n A \left( \frac{C2}{0.5pb^2} \right) \frac{(1+\Gamma)}{\pi n} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} + \frac{C2}{0.5pb^2} \right] \dots \dots \dots 141$$

Table 22 Rm 1+ verses h/b for  $\gamma = 0.1$

h/b	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	0.999951	0.999951	0.999951	0.999951	0.99995096	0.999951	0.999951
0.1	0.999951	0.998383	0.996821	0.995263	0.99371005	0.992162	0.9906192
0.2	0.999951	0.988358	0.977034	0.965971	0.95515929	0.94459	0.9342565
0.3	0.999951	0.964717	0.93186	0.901151	0.8723885	0.845396	0.8200173
0.4	0.999951	0.928113	0.86552	0.81054	0.76189838	0.718586	0.6797946
0.5	0.999951	0.884514	0.79151	0.715142	0.651433	0.597565	0.5514872
0.6	0.999951	0.841114	0.722643	0.631228	0.55878815	0.50014	0.4518116
0.7	0.999951	0.803077	0.665842	0.565243	0.48869295	0.428736	0.3806792
0.8	0.999951	0.772618	0.622556	0.516785	0.43867458	0.378937	0.3319877
0.9	0.999951	0.749726	0.591226	0.48265	0.40415017	0.345104	0.2993234
1	0.999951	0.73327	0.569312	0.459225	0.38078797	0.322454	0.2776398
1.5	0.999951	0.70313	0.530431	0.418548	0.34084387	0.284176	0.2413245
2	0.999951	0.699571	0.525944	0.413925	0.33635337	0.279908	0.2373007

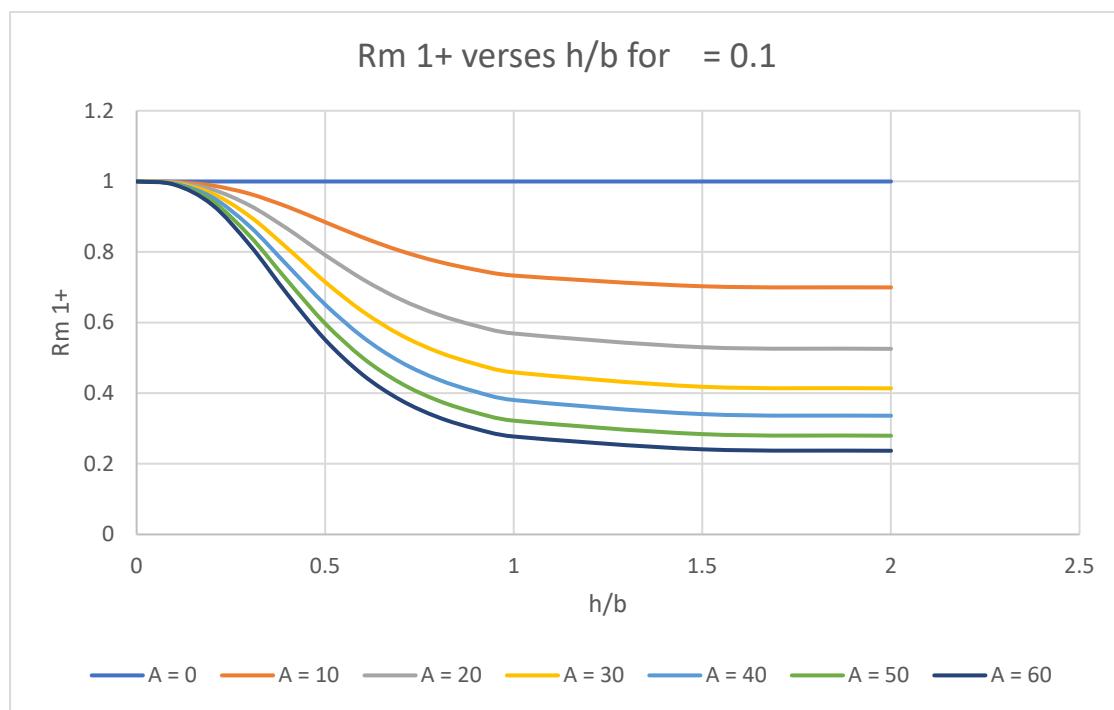
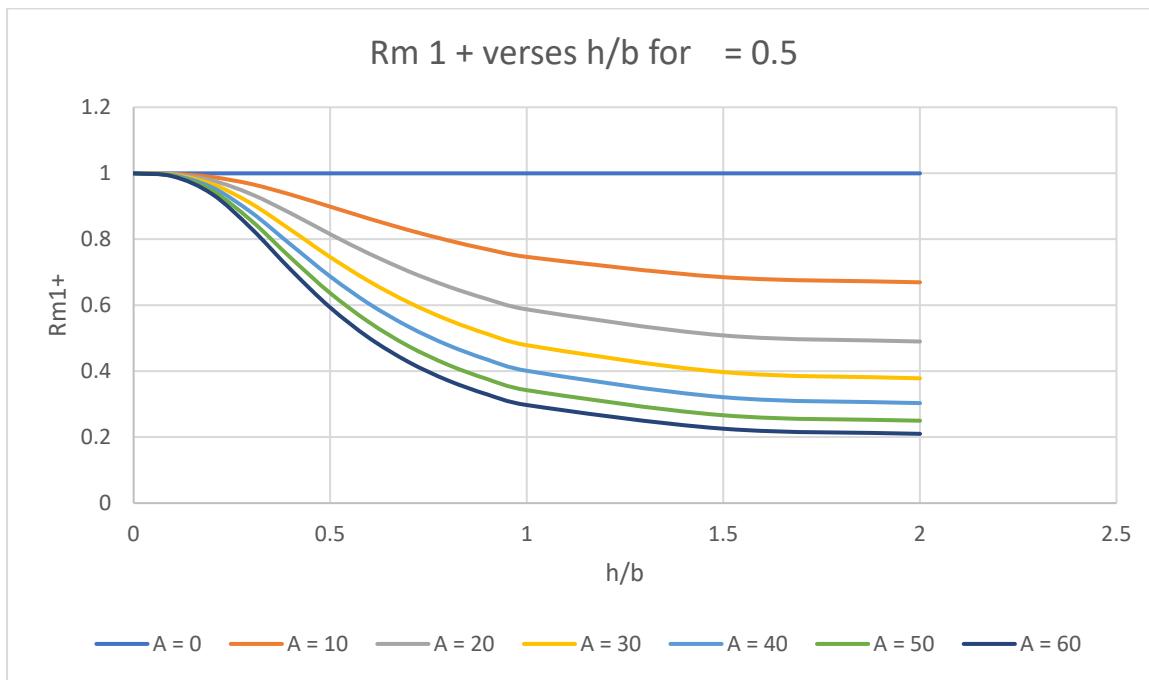


Table 23 Rm 1 + verses h/b for  $\gamma = 0.5$

h/b	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	0.999909	0.999909	0.999909	0.999909	0.99990882	0.999909	0.9999088
0.1	0.999909	0.998372	0.996839	0.995312	0.99378887	0.992271	0.9907579
0.2	0.999909	0.988781	0.977913	0.967294	0.95691684	0.946773	0.9368535
0.3	0.999909	0.967093	0.936445	0.90776	0.88085563	0.855572	0.8317669
0.4	0.999909	0.935255	0.878464	0.828206	0.78343392	0.74331	0.7071564
0.5	0.999909	0.898799	0.815667	0.746202	0.68735739	0.636924	0.5932582
0.6	0.999909	0.862153	0.755944	0.671778	0.60359578	0.547353	0.5002502
0.7	0.999909	0.827676	0.702616	0.608077	0.53436923	0.475479	0.4274808
0.8	0.999909	0.796564	0.656754	0.555309	0.47872612	0.419123	0.3715985
0.9	0.999909	0.769451	0.618469	0.512661	0.43487749	0.375611	0.3291725
1	0.999909	0.746567	0.587329	0.478896	0.40087229	0.342413	0.2972326
1.5	0.999909	0.684949	0.508518	0.39703	0.32099004	0.266306	0.2254128
2	0.999909	0.669306	0.48961	0.378128	0.30305216	0.249574	0.2098857



And Rm1- is

Table 24 Rm 1 verses h/b for = 0.1

h/b	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	1.000025	1.000025	1.000025	1.000025	1.00002452	1.000025	1.0000245
0.1	1.000025	0.999284	0.998544	0.997807	0.99707129	0.996337	0.9956053
0.2	1.000025	0.994063	0.988203	0.982443	0.97678025	0.971211	0.9657331
0.3	1.000025	0.98062	0.962237	0.944789	0.92819977	0.912402	0.8973358
0.4	1.000025	0.95914	0.922618	0.889749	0.85997236	0.832841	0.8079907
0.5	1.000025	0.933217	0.877679	0.830639	0.7901768	0.754921	0.7238644
0.6	1.000025	0.907266	0.835588	0.778258	0.73115992	0.691636	0.6578891
0.7	1.000025	0.884476	0.800808	0.737005	0.68645644	0.64522	0.610796
0.8	1.000025	0.866215	0.774303	0.706731	0.65460182	0.61292	0.5786579
0.9	1.000025	0.85249	0.755128	0.685429	0.63265586	0.591034	0.5571726
1	1.000025	0.842624	0.741723	0.670825	0.61782886	0.576414	0.5429501
1.5	1.000025	0.824557	0.717955	0.645497	0.59252415	0.551767	0.5192063
2	1.000025	0.822424	0.715213	0.642621	0.5896826	0.549023	0.5165806

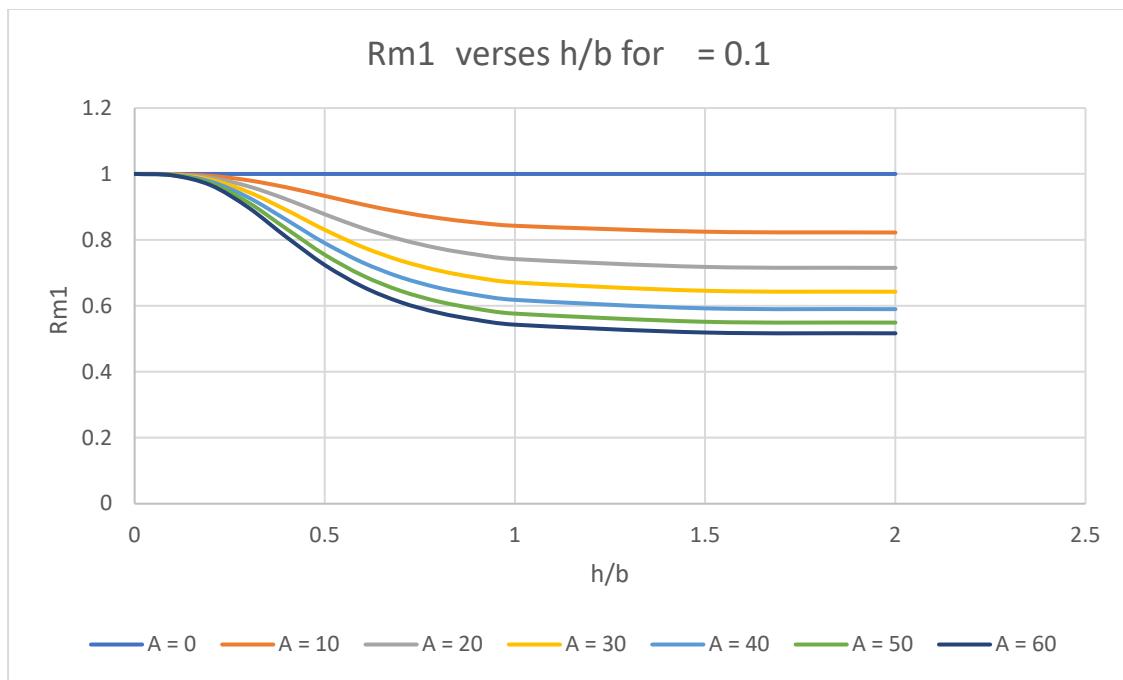
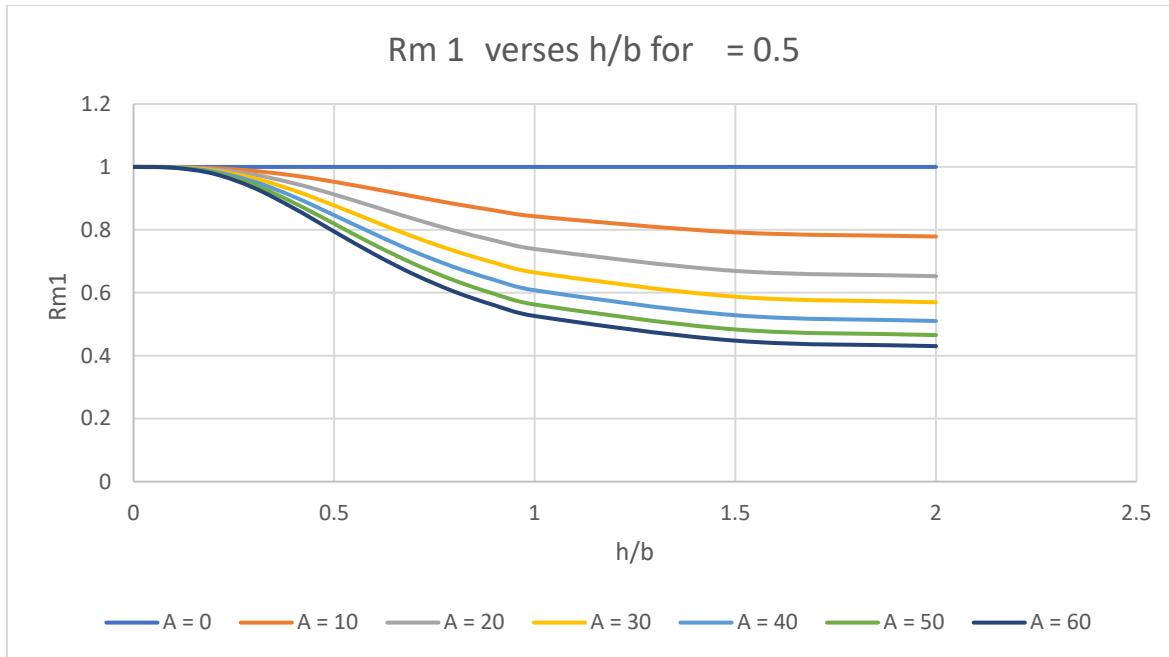


Table 25 Rm 1 verses h/b for  $\gamma = 0.5$

h/b	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	1.000046	1.000046	1.000046	1.000046	1.00004559	1.000046	1.0000456
0.1	1.000046	0.999581	0.999118	0.998657	0.99819622	0.997737	0.9972794
0.2	1.000046	0.996195	0.99242	0.988718	0.98508605	0.981523	0.978026
0.3	1.000046	0.987408	0.97545	0.96411	0.95333555	0.943078	0.9332962
0.4	1.000046	0.97257	0.947801	0.92531	0.90475491	0.885861	0.8684047
0.5	1.000046	0.952792	0.912461	0.877482	0.84673991	0.819417	0.7949
0.6	1.000046	0.929925	0.87337	0.826488	0.78677611	0.752547	0.7226189
0.7	1.000046	0.905967	0.834211	0.77721	0.73052288	0.691362	0.6578864
0.8	1.000046	0.88272	0.79783	0.732948	0.68135135	0.639071	0.6036074
0.9	1.000046	0.861518	0.765953	0.69532	0.64052962	0.596489	0.5601135
1	1.000046	0.84312	0.739251	0.664601	0.60785843	0.562945	0.5262956
1.5	1.000046	0.792188	0.66965	0.587817	0.52869721	0.483616	0.4478665
2	1.000046	0.779074	0.652694	0.569804	0.51063227	0.465894	0.4306407



*Rv1*, the stress and deflections are the same just C2 is different

Table 26 Rv 1 verses h/b for  $\gamma = 0.1$

h/b	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	0.999755	0.999755	0.999755	0.999755	0.99975487	0.999755	0.9997549
0.1	0.999755	1.002742	1.005721	1.008692	1.01165495	1.01461	1.0175573
0.2	0.999755	1.005983	1.012076	1.018038	1.02387243	1.029584	1.0351756
0.3	0.999755	1.004562	1.008983	1.013055	1.01680963	1.020275	1.0234777
0.4	0.999755	0.999488	0.998921	0.998122	0.99713943	0.996013	0.9947744
0.5	0.999755	0.992637	0.98616	0.980203	0.9746773	0.969517	0.9646723
0.6	0.999755	0.985595	0.973884	0.963892	0.95516971	0.947419	0.9404372
0.7	0.999755	0.979366	0.963671	0.950967	0.94030811	0.931126	0.923056
0.8	0.999755	0.974366	0.955879	0.941476	0.92971965	0.919797	0.9112109
0.9	0.999755	0.970606	0.950243	0.934804	0.92243584	0.912137	0.9033131
1	0.999755	0.967903	0.946305	0.930234	0.91752207	0.90703	0.8980973
1.5	0.999755	0.962955	0.939327	0.922319	0.90915088	0.89844	0.8894139
2	0.999755	0.96237	0.938523	0.92142	0.90821191	0.897485	0.8884553

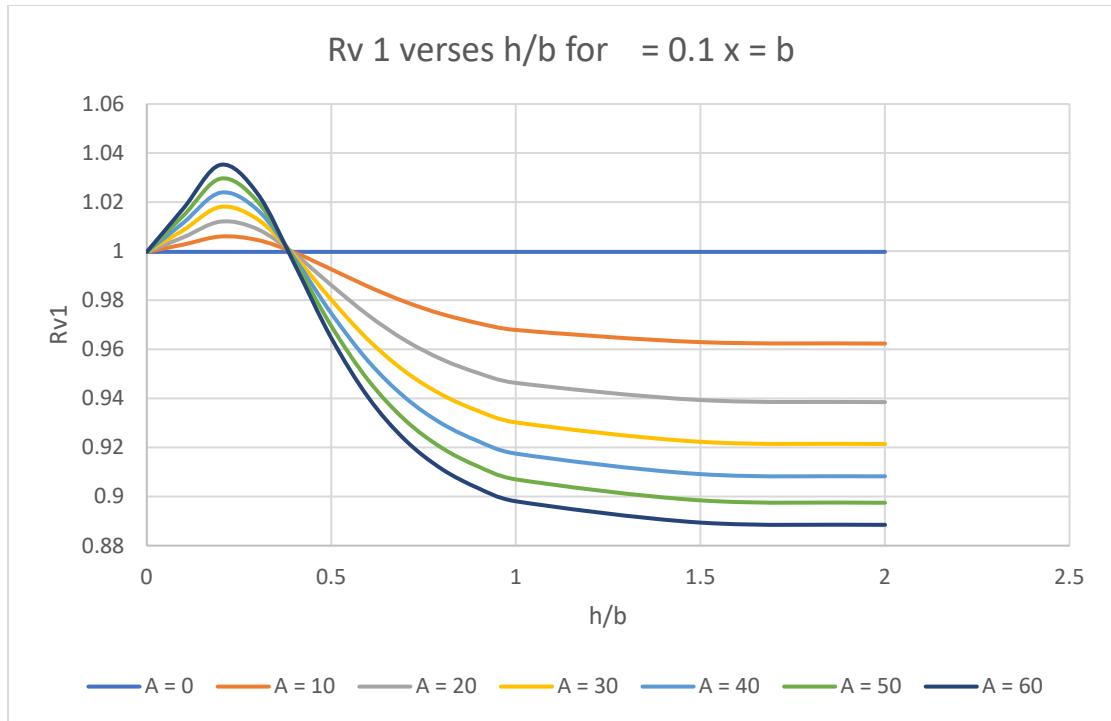
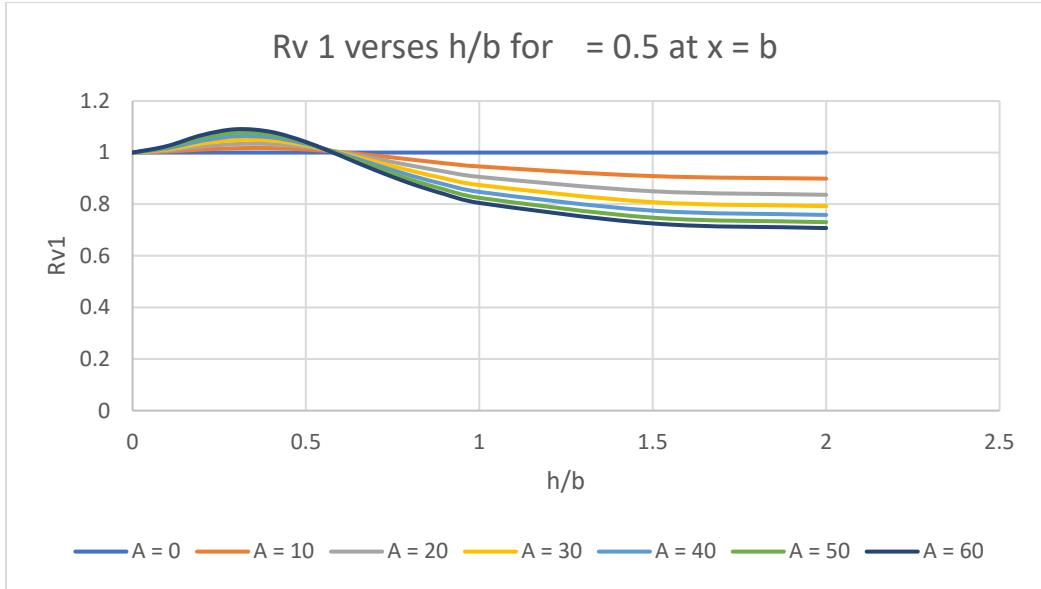


Table 27 Rv 1 verses h/b for  $\gamma = 0.5$

$h/b$	$A = 0$	$A = 10$	$A = 20$	$A = 30$	$A = 40$	$A = 50$	$A = 60$
1E 13	0.999909	0.999909	0.999909	0.999909	0.99990883	0.999909	0.9999088
0.1	0.999909	1.004158	1.008399	1.012631	1.01685509	1.02107	1.0252767
0.2	0.999909	1.011678	1.023267	1.034681	1.04592511	1.057003	1.0679183
0.3	0.999909	1.016911	1.033074	1.048467	1.06314793	1.077171	1.090583
0.4	0.999909	1.017324	1.032742	1.046481	1.05879199	1.069878	1.0799054
0.5	0.999909	1.012099	1.021396	1.028516	1.03396112	1.038097	1.0411934
0.6	0.999909	1.0018	1.001239	0.999091	0.99589769	0.992008	0.9876571
0.7	0.999909	0.988184	0.976304	0.964597	0.95322172	0.942249	0.9317086
0.8	0.999909	0.973319	0.950515	0.930416	0.91238486	0.896008	0.8809992
0.9	0.999909	0.958879	0.92658	0.899784	0.87679877	0.856632	0.8386487
1	0.999909	0.945903	0.905877	0.874031	0.84754037	0.824835	0.8049596
1.5	0.999909	0.908771	0.85026	0.807871	0.77485945	0.747915	0.7251991
2	0.999909	0.899056	0.836507	0.792136	0.75806586	0.730537	0.7075026



$$\frac{\sigma_y}{p} \text{ at } x = 0 = \frac{sft1}{p} = \frac{1}{\Gamma} \sum_{n=1}^{\infty} 2k_n A \frac{2 \frac{(1+\Gamma)}{\pi n} + \frac{\pi n}{(1+\Gamma)} \frac{C2}{0.5pb^2}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \sin \alpha_n b - 1 \dots \dots \dots 144$$

*Reaction at  $x = b + t$*

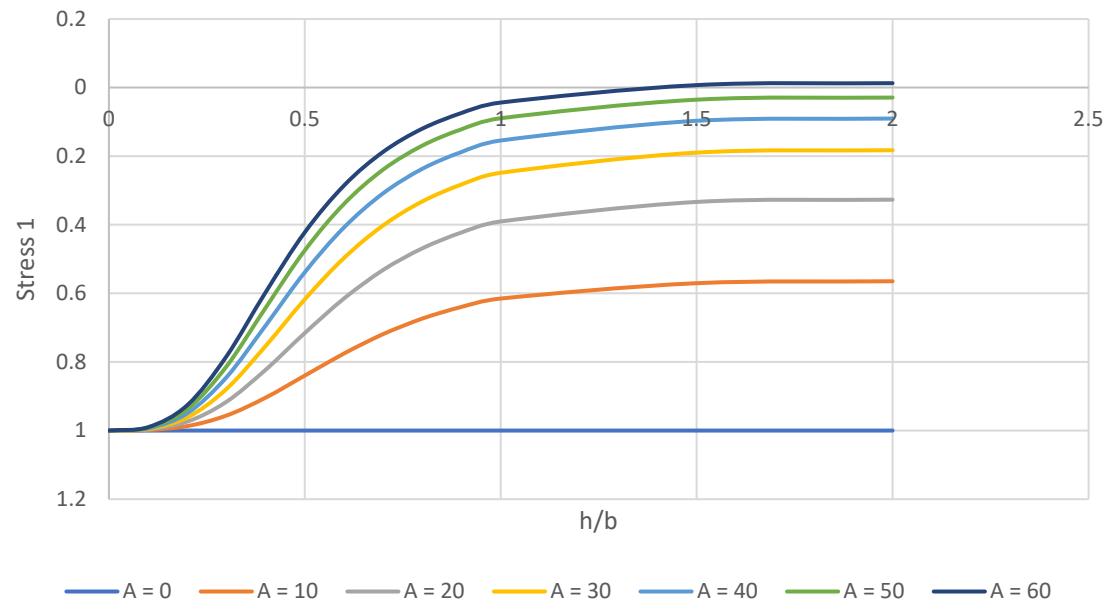
$$= \frac{sft1}{p}$$

$$= \frac{1}{\Gamma} \sum_{n=1}^{\infty} 2k_n A \frac{2 \frac{(1+\Gamma)}{\pi n} + \frac{\pi n}{(1+\Gamma)} \frac{C2}{0.5pb^2}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \sin \alpha_n b \cos \pi n + \frac{1}{\Gamma} \dots \dots \dots 145$$

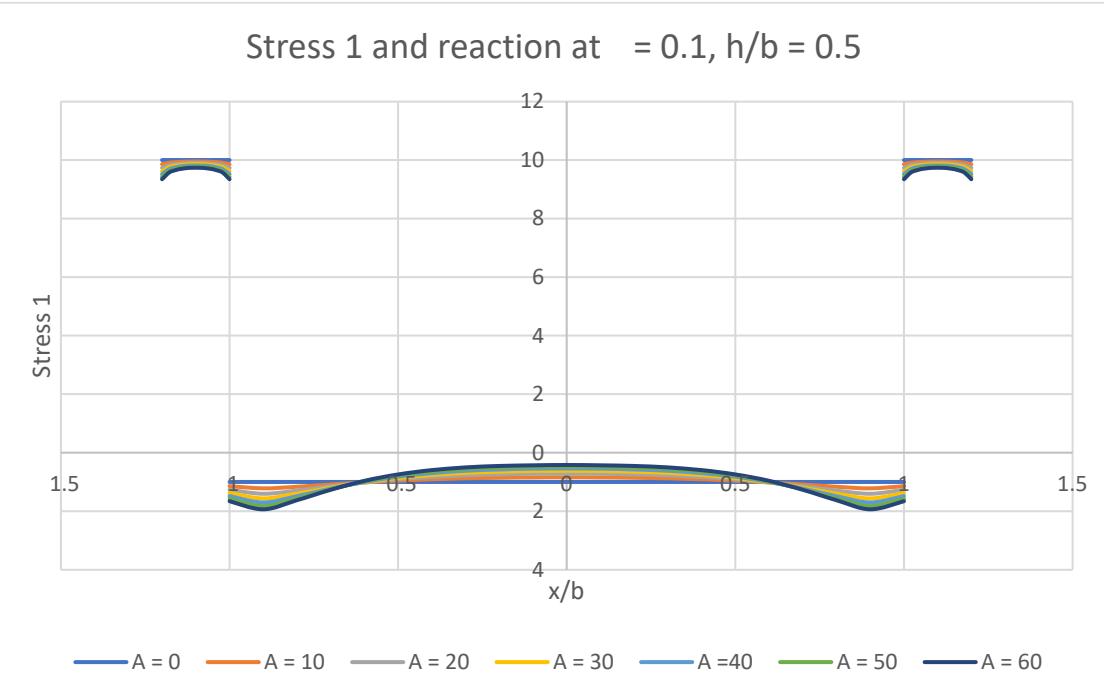
Table 28 Stress 1 at  $x = 0$  verses h/b for  
 $\Gamma = 0.1$

h/b	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	1	1	1	1	1	1	1
0.1	1	0.99834	0.99668	0.99502	0.9933774	0.99174	0.990099
0.2	1	0.98683	0.974	0.9615	0.9493263	0.93745	0.925877
0.3	1	0.95618	0.91592	0.87882	0.8445555	0.81282	0.783373
0.4	1	0.90391	0.82254	0.75306	0.6932617	0.64145	0.596278
0.5	1	0.83966	0.71528	0.61704	0.5382516	0.47424	0.421654
0.6	1	0.77527	0.61483	0.49664	0.4074426	0.33882	0.285181
0.7	1	0.71877	0.53181	0.4017	0.3081012	0.23908	0.18721
0.8	1	0.67351	0.46846	0.33181	0.2369526	0.16917	0.119707
0.9	1	0.63948	0.42256	0.28247	0.187685	0.12147	0.07416
1	1	0.61502	0.39043	0.24856	0.1542644	0.08942	0.04379
1.5	1	0.5702	0.33337	0.18957	0.0969685	0.03506	0.0073187
2	1	0.56491	0.32678	0.18286	0.0905168	0.02898	0.0129983

Stress 1 at  $x = 0$  verses  $h/b$  for  $\gamma = 0.1$



Stress 1 and reaction at  $\gamma = 0.1$ ,  $h/b = 0.5$



Stress 1 and reaction at  $\gamma = 0.1$ ,  $h/b = 1$

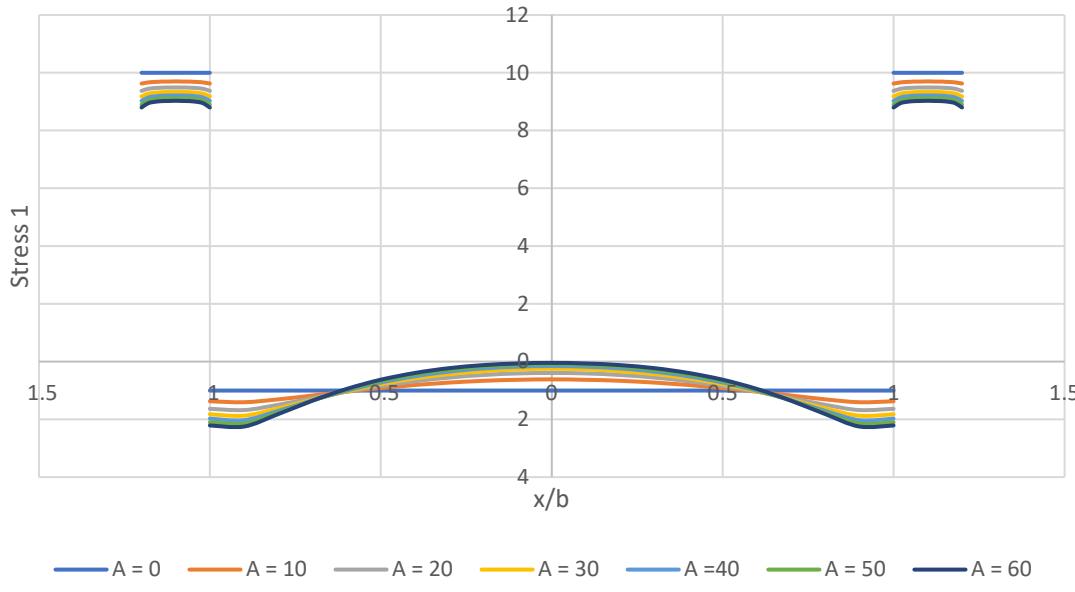
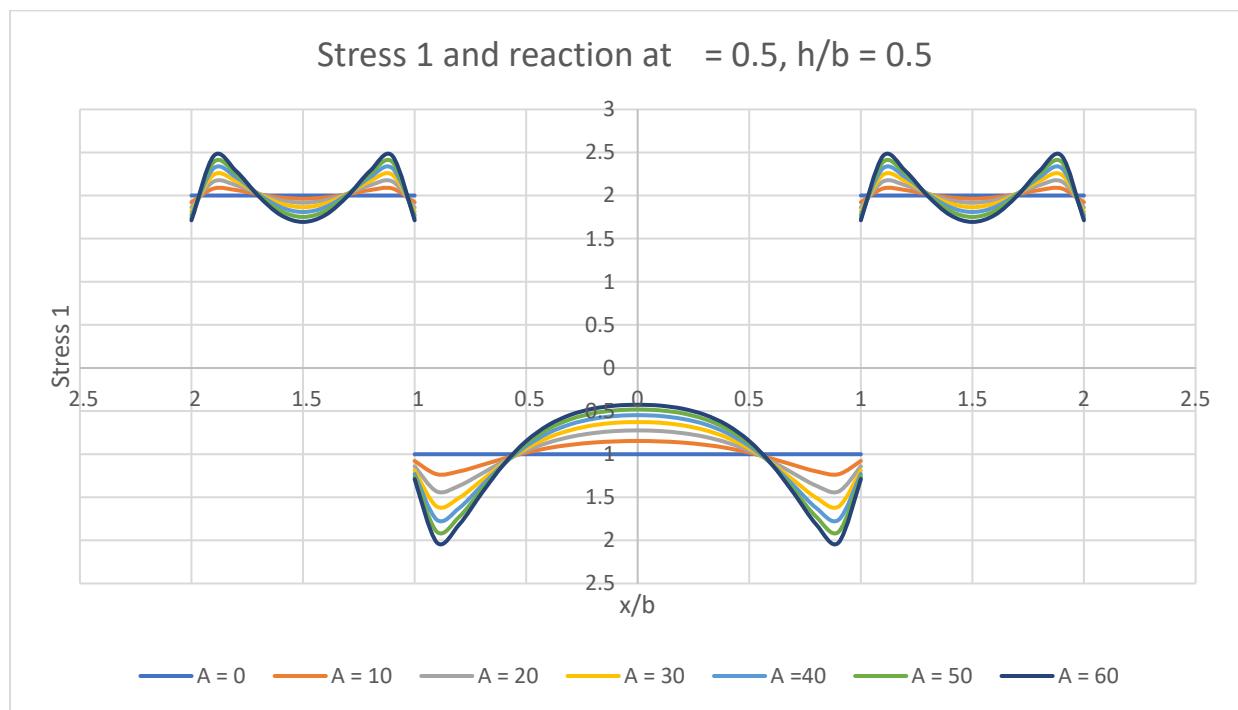
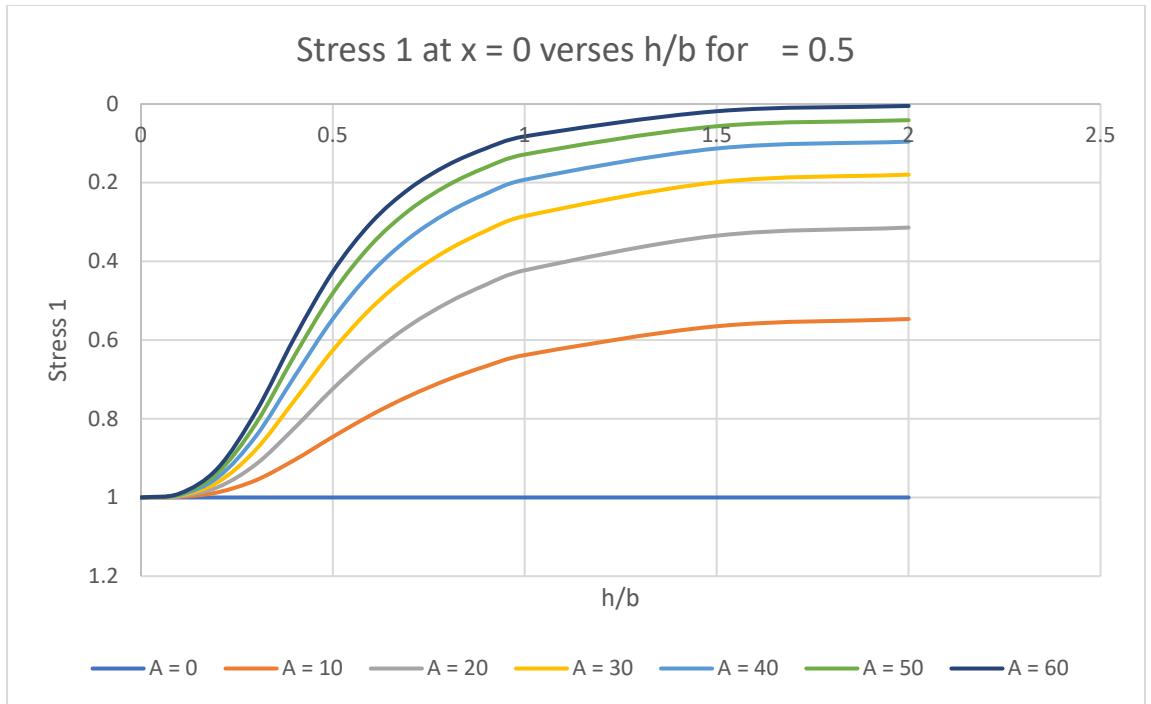
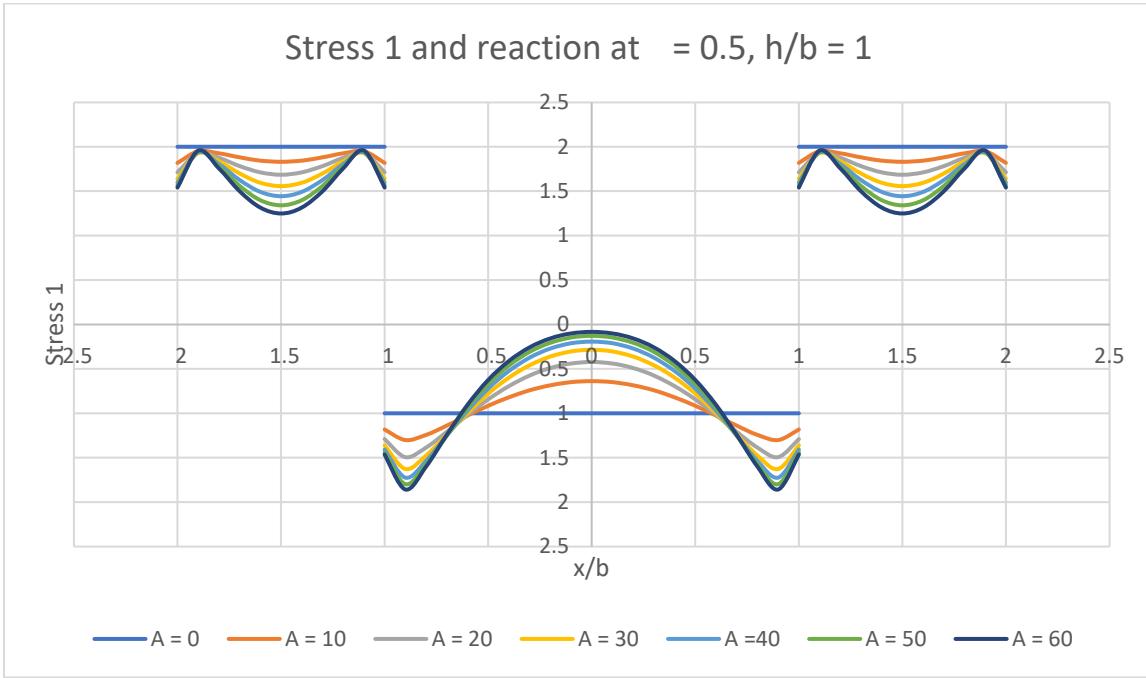


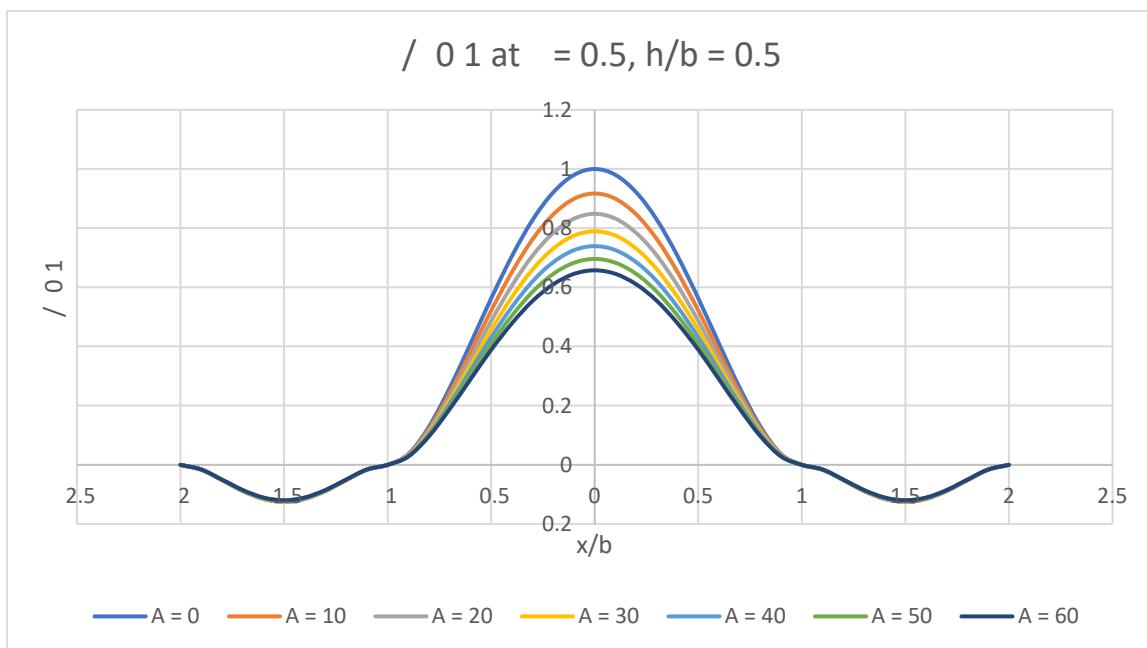
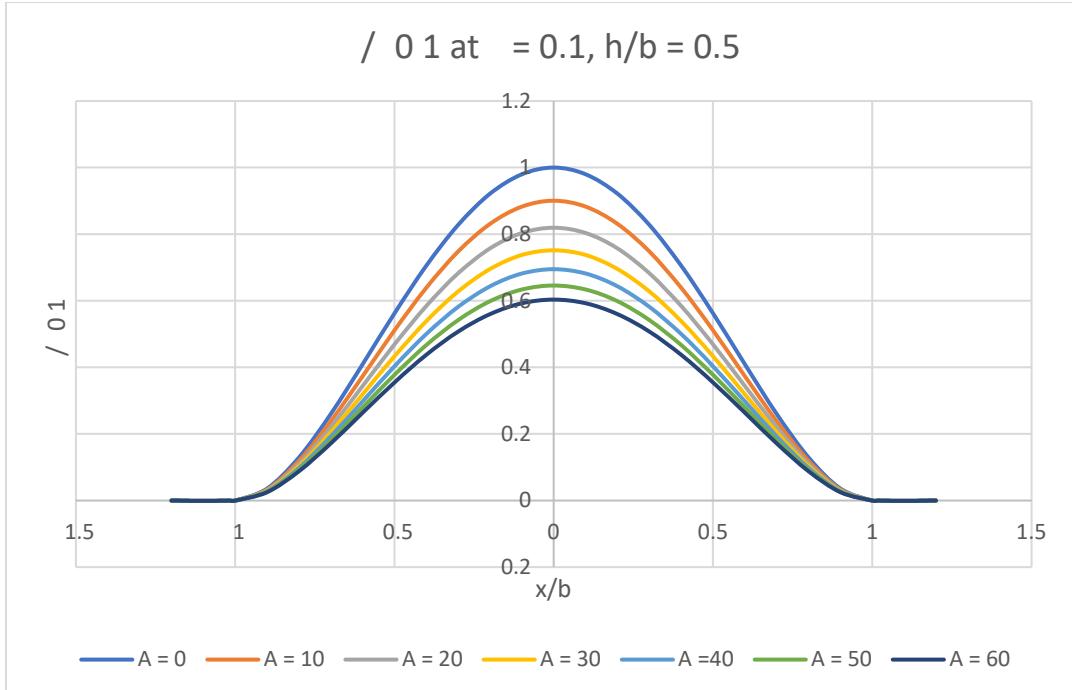
Table 29 Stress 1 at  $x = 0$  verses  $h/b$  for  $\gamma = 0.5$

$h/b$	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	1	1	1	1	1	1	1
0.1	1	0.99834	0.99668	0.99502	0.9933774	0.99174	0.990099
0.2	1	0.98682	0.97399	0.96149	0.9493129	0.93744	0.92586
0.3	1	0.95598	0.91549	0.87816	0.8436601	0.8117	0.782032
0.4	1	0.90474	0.82343	0.75351	0.6929952	0.64031	0.594181
0.5	1	0.84591	0.72394	0.62593	0.5461756	0.48058	0.426126
0.6	1	0.79022	0.63551	0.51852	0.4283021	0.3576	0.301461
0.7	1	0.74158	0.56288	0.43471	0.3402506	0.26914	0.214713
0.8	1	0.70038	0.50477	0.37068	0.2755015	0.20617	0.15468
0.9	1	0.66609	0.45877	0.32193	0.2277433	0.16095	0.112522
1	1	0.63802	0.42266	0.28486	0.1923269	0.12809	0.082419
1.5	1	0.56486	0.33469	0.19877	0.1130208	0.05658	0.018398
2	1	0.5466	0.31402	0.17937	0.0956799	0.0413	0.004973





The deflection does not take into account the moment of inertia of the elastic media, so it is conservative. But for a media such as soils where tension is not allowed it is accurate and useful.



Once  $D$  is found the stresses at  $q = 0$  are

$$a_n = -[c_n(1 + 2\alpha_n h) + 2\alpha_n^2 h^2]e^{-2\alpha_n h}$$

$$b_n = [2c_n - 1 + 2\alpha_n h]e^{-2\alpha_n h}$$

$$c_n = 1 + \frac{2\alpha_n^2 h^2 e^{-2\alpha_n h}}{(1 - 2\alpha_n h)e^{-2\alpha_n h} - 1}$$

$$d_n = a_n - b_n - c_n - 1$$

$$k_n = -\frac{a_n + c_n}{d_n}$$

## Rewrite for the excel sheet

$$\sigma_y = - \sum_{n=\varphi}^{\infty} D \cos \alpha_n x \left[ \left( a_{n1} + b_{n1} \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right) + \left( c_n + \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} \right] - p \left( 1 - \frac{y}{h} \right) \dots \dots \dots 153$$

$$a_{n1} = a_n e^{\alpha_n y} = -[c_n(1 + 2\alpha_n h) + 2\alpha_n^2 h^2] e^{-\alpha_n h}$$

$$b_{n1} = b_n e^{\alpha_n y} = [2c_n - 1 + 2\alpha_n h] e^{-\alpha_n h}$$

The stresses with  $q = 0$  are:

Now setting the rotation to be zero at  $x = \pm b$  for  $p = 0$  we have:

$$0 = \sum_{n=1}^{\infty} \frac{\frac{W_n}{q} + \frac{2 \sin \alpha_n b}{\Gamma(\pi n)}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \sin \alpha_n b$$

$$\frac{C2 \cdot 2}{0.5qb^2} = -\frac{\sum_{n=1}^{\infty} \frac{\frac{\Gamma W_n}{q} \sin \alpha_n b + 2 \frac{(\sin \alpha_n b)^2}{\pi n}}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A}}{\sum_{n=1}^{\infty} \frac{\frac{\pi n}{(1+\Gamma)} (\sin \alpha_n b)^2}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A}} \quad \dots \dots \dots \quad 156$$

Note: the maximum moment may not be at  $x = 0$  depending on  $A$

Table 30 Rm2+ verses h/b for  $\alpha = 0.1$

h/b	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	0.999951	0.999951	0.999951	0.999951	0.99995096	0.999951	0.999951
0.1	1.000115	0.998547	0.996984	0.995426	0.99387263	0.992325	0.9907813
0.2	1.000922	0.989317	0.977982	0.966907	0.95608366	0.945504	0.935159
0.3	1.002199	0.966875	0.933934	0.903146	0.87430978	0.847248	0.8218049
0.4	1.002979	0.930966	0.868214	0.813089	0.76431422	0.72088	0.6819768
0.5	1.001226	0.885941	0.793031	0.716717	0.65303264	0.599169	0.5530826
0.6	0.995996	0.838496	0.720961	0.630219	0.55827295	0.499993	0.4519402
0.7	0.987893	0.794486	0.659579	0.560616	0.48525627	0.426187	0.3788059
0.8	0.978185	0.757132	0.611111	0.50811	0.43198114	0.37371	0.3278726
0.9	0.968082	0.727237	0.574573	0.469914	0.39418369	0.337171	0.2929271
1	0.958444	0.704185	0.547764	0.442664	0.36772175	0.311941	0.2690511
1.5	0.926194	0.651872	0.492221	0.388759	0.31687964	0.264441	0.2247715
2	0.914313	0.639852	0.481194	0.378821	0.30792247	0.256327	0.217375

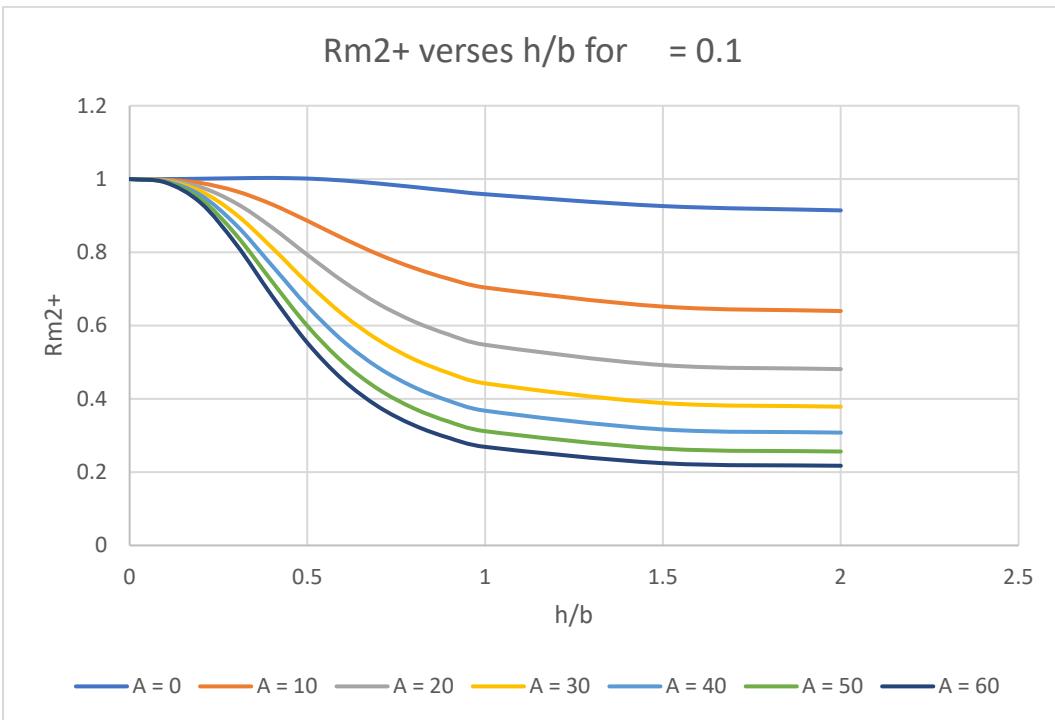


Table 31 Rm2+ verses h/b for  $\gamma = 0.5$

$h/b$	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	0.999909	0.999909	0.999909	0.999909	0.99990882	0.999909	0.9999088
0.1	1.000075	0.998537	0.997004	0.995477	0.99395379	0.992436	0.9909225
0.2	1.001236	0.990094	0.979211	0.968579	0.95818814	0.948031	0.9380988
0.3	1.00447	0.971484	0.940678	0.911845	0.88480226	0.859389	0.8354625
0.4	1.010034	0.944653	0.887224	0.836401	0.79112512	0.75055	0.7139909
0.5	1.015145	0.912496	0.828087	0.757544	0.69777974	0.646551	0.6021921
0.6	1.015705	0.876153	0.768512	0.683175	0.61401449	0.55694	0.5091207
0.7	1.009098	0.836346	0.710805	0.61582	0.54169997	0.482428	0.4340767
0.8	0.994937	0.794493	0.656503	0.556245	0.48045463	0.421386	0.374221
0.9	0.974435	0.75251	0.606878	0.50464	0.42934376	0.371864	0.3267393
1	0.949544	0.712205	0.562738	0.460747	0.38719796	0.331966	0.28918
1.5	0.818343	0.563711	0.420807	0.330314	0.2684521	0.223856	0.1904226
2	0.735459	0.493921	0.362498	0.280867	0.22582275	0.186559	0.1573764

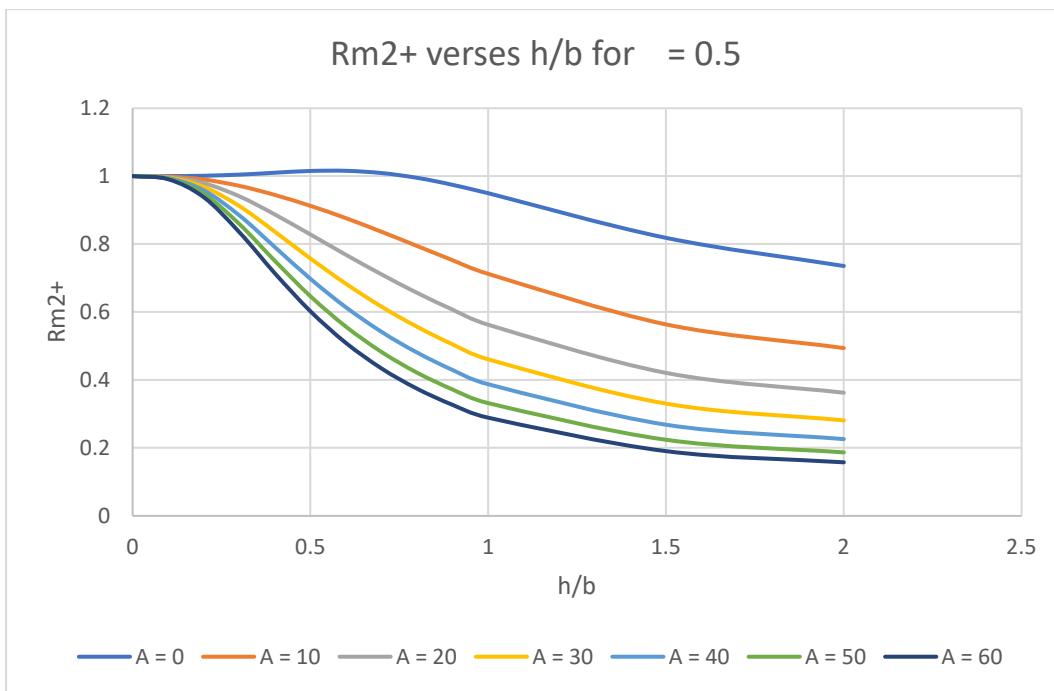


Table 32 Rm2 verses h/b for = 0.1

h/b	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	1.000025	1.000025	1.000025	1.000025	1.00002452	1.000025	1.0000245
0.1	0.999892	0.99915	0.998411	0.997673	0.99693662	0.996202	0.9954697
0.2	0.997645	0.991677	0.985812	0.980047	0.97437772	0.968803	0.9633198
0.3	0.99255	0.973146	0.954763	0.937318	0.92073191	0.904939	0.8898776
0.4	0.985821	0.945045	0.90863	0.875865	0.84619108	0.819159	0.7944068
0.5	0.978196	0.91188	0.856781	0.810136	0.77003524	0.735112	0.7043624
0.6	0.970059	0.87862	0.808015	0.751585	0.7052618	0.666417	0.6332733
0.7	0.961768	0.848841	0.767144	0.704899	0.65562992	0.615473	0.581979
0.8	0.953724	0.824182	0.735283	0.669987	0.61966203	0.579461	0.5464459
0.9	0.94628	0.804796	0.711509	0.644787	0.59431423	0.554543	0.5222171
1	0.939663	0.790049	0.694214	0.626931	0.57667906	0.537442	0.505764
1.5	0.919306	0.757311	0.658924	0.592072	0.54321474	0.505638	0.4756272
2	0.912214	0.749992	0.652074	0.585781	0.53744187	0.500319	0.4707014

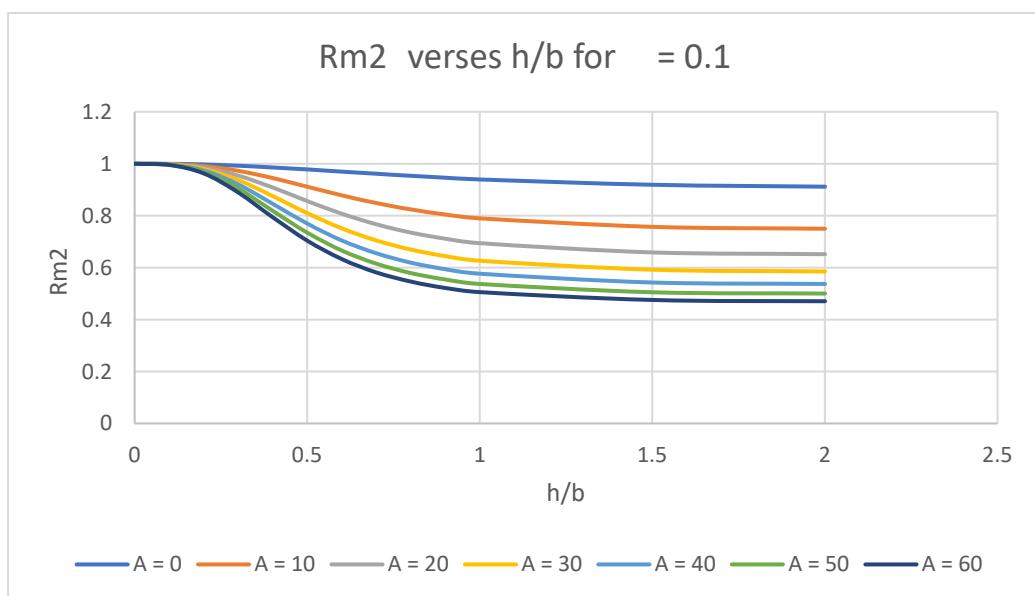
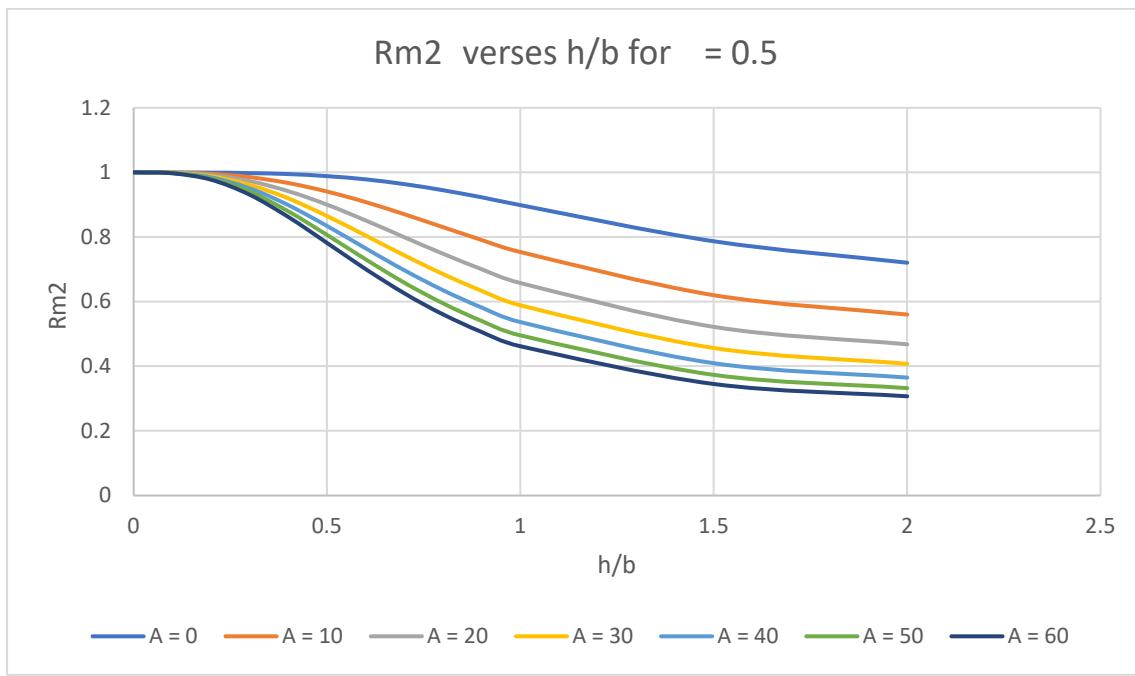


Table 33 Rm2 verses h/b for = 0.5

$h/b$	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	1.000046	1.000046	1.000046	1.000046	1.00004559	1.000046	1.0000456
0.1	0.999963	0.999498	0.999035	0.998573	0.99811294	0.997654	0.997196
0.2	0.999383	0.995526	0.991745	0.988037	0.9844002	0.980832	0.9773291
0.3	0.997824	0.985128	0.973116	0.961726	0.95090338	0.940601	0.9307767
0.4	0.994609	0.966916	0.941959	0.919302	0.89860103	0.879578	0.862007
0.5	0.988517	0.940873	0.900237	0.86502	0.83408927	0.806617	0.7819828
0.6	0.978367	0.908028	0.851372	0.804467	0.76478381	0.73062	0.7007833
0.7	0.963673	0.870403	0.799395	0.743092	0.69705725	0.658509	0.6256089
0.8	0.944797	0.830558	0.748089	0.685201	0.63529749	0.59449	0.56033
0.9	0.922733	0.790946	0.700263	0.633409	0.5816771	0.540194	0.5060076
1	0.898769	0.753447	0.657515	0.588755	0.53662586	0.495469	0.4619664
1.5	0.78691	0.620045	0.521875	0.456457	0.40929672	0.373411	0.34501
2	0.720646	0.559773	0.467866	0.407656	0.36472359	0.3323	0.3067786



*Rv2*, the stress and deflections are the same just *C2* is different

$$Rv2 = \frac{\int \sigma_y}{-qb} \\ = \sum_{n=1}^{\infty} \left( \frac{+ \frac{W_n}{q} \frac{(\pi n)^2}{(1+\Gamma)^2} + \frac{2}{\Gamma} \frac{\pi n}{(1+\Gamma)^2} \sin \alpha_n b - 2k_n A \frac{1}{\Gamma} \frac{C2}{0.5qb^2} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \right) \sin \alpha_n b^* \dots \dots \dots 159$$

And the stress at  $x = 0$  we have

Table 34 Rv 2 verses h/b for  $\gamma = 0.1$ ,  $b = b^*$

h/b	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	0.999755	0.999755	0.999755	0.999755	0.99975487	0.999755	0.9997549
0.1	0.985215	0.988204	0.991185	0.994158	0.99712304	1.00008	1.0030292
0.2	0.966849	0.973095	0.979205	0.985183	0.99103492	0.996763	1.002371
0.3	0.951827	0.956682	0.961149	0.965266	0.96906474	0.972574	0.9758186
0.4	0.941455	0.941291	0.940826	0.940124	0.93923742	0.938205	0.9370565
0.5	0.934249	0.927336	0.921055	0.915285	0.90993902	0.904952	0.9002726
0.6	0.92902	0.91524	0.903871	0.894194	0.88576276	0.878285	0.8715588
0.7	0.925027	0.905283	0.890127	0.87789	0.86764869	0.858845	0.8511223
0.8	0.92186	0.897451	0.879727	0.865955	0.85474153	0.845298	0.8371429
0.9	0.919299	0.891496	0.872124	0.857473	0.84576337	0.836033	0.8277131
1	0.917218	0.88707	0.866675	0.851533	0.83958133	0.829735	0.8213672
1.5	0.911498	0.87757	0.855806	0.840153	0.82804505	0.818203	0.8099158
2	0.909658	0.875525	0.853757	0.838151	0.82610163	0.816318	0.8080845

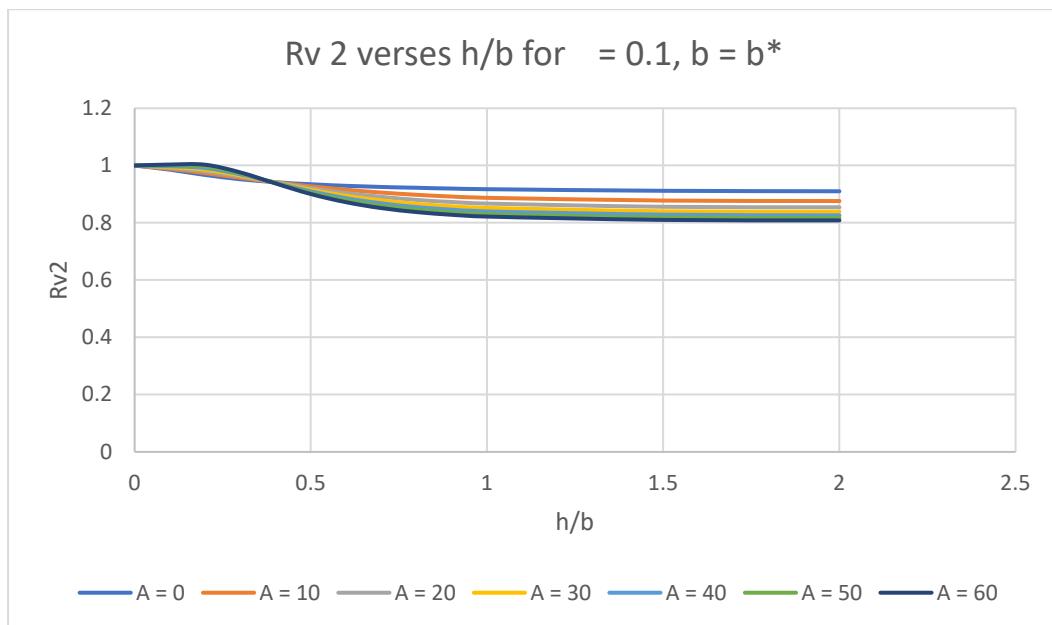
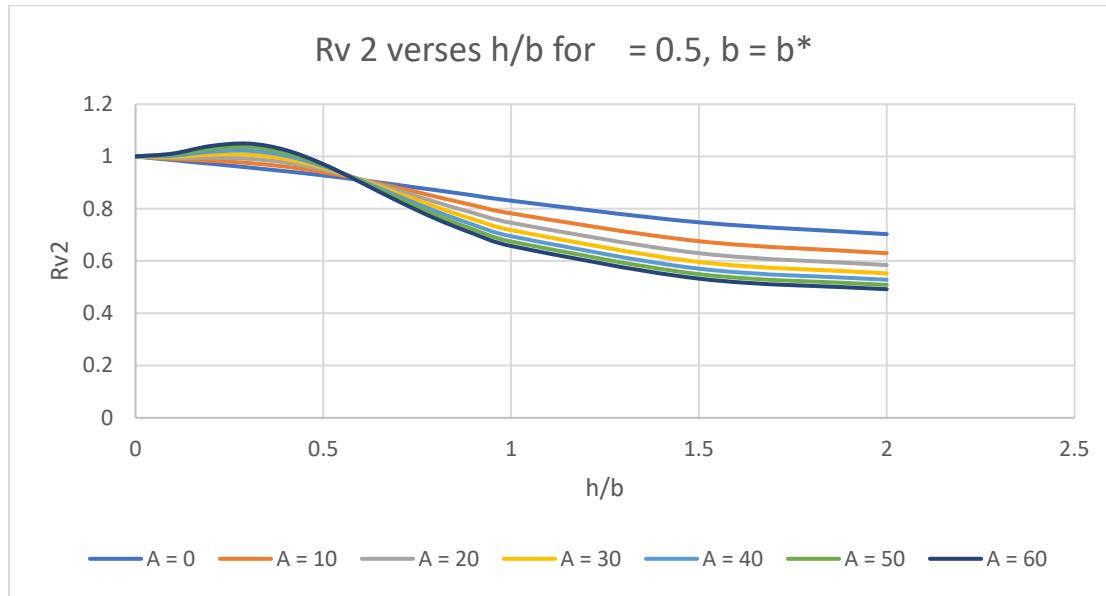


Table 35 Rv 2 verses h/b for  $\gamma = 0.5$ ,  $b = b^*$

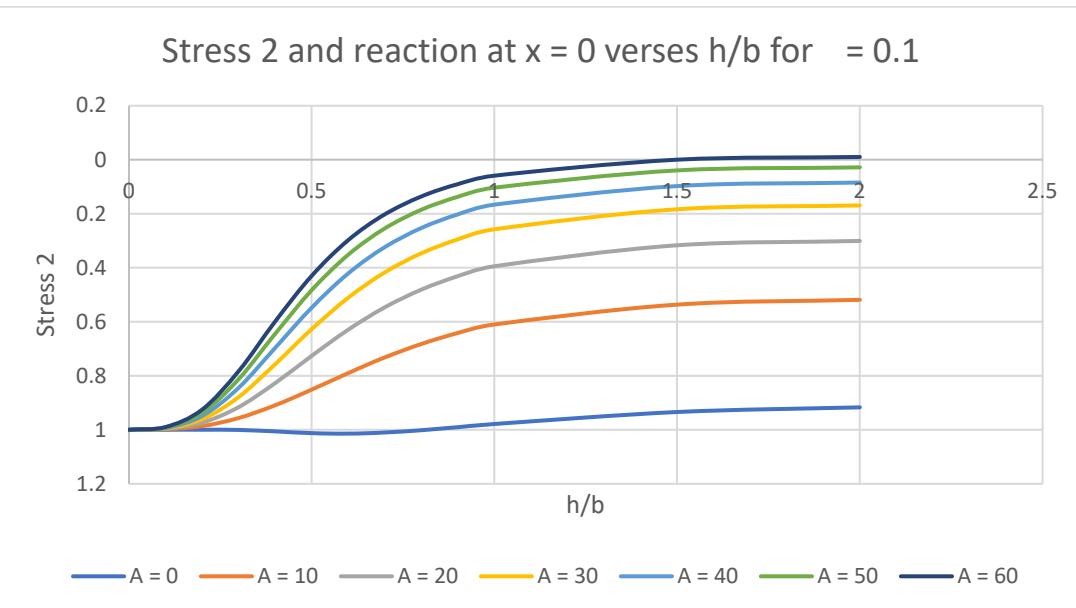
$h/b$	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	0.999909	0.999909	0.999909	0.999909	0.99990883	0.999909	0.9999088
0.1	0.985886	0.990138	0.994382	0.998618	1.00284491	1.007063	1.0112731
0.2	0.971833	0.98364	0.995266	1.006716	1.01799517	1.029108	1.0400582
0.3	0.957787	0.974905	0.991179	1.006678	1.02146154	1.035583	1.0490895
0.4	0.943192	0.960844	0.976478	0.990414	1.00290753	1.014163	1.0243483
0.5	0.927158	0.939756	0.949414	0.95686	0.96260191	0.967012	0.9703628
0.6	0.90952	0.912056	0.912083	0.91048	0.90779517	0.904384	0.9004858
0.7	0.890587	0.879939	0.869068	0.85831	0.84782816	0.837701	0.8279602
0.8	0.870787	0.846125	0.825072	0.806581	0.79003625	0.775042	0.7613247
0.9	0.850617	0.812949	0.783515	0.759239	0.73851278	0.720397	0.704293
1	0.830625	0.782014	0.746289	0.718062	0.69471377	0.674797	0.6574295
1.5	0.747966	0.675728	0.629668	0.596503	0.57081036	0.549934	0.5324004
2	0.702596	0.629614	0.584516	0.552628	0.52821315	0.508533	0.4921012



And the stress at  $x = b + t$  we have

Table 36 Stress 2 at x = 0 verses h/b for  $\epsilon = 0.1$

$h/b$	A = 0	A = 10	A = 20	A = 30	A = 40	A = 50	A = 60
1E 13	0.99932	0.99932	0.99932	0.99932	0.9993212	0.99932	0.999321
0.1	0.99938	0.99772	0.99606	0.99441	0.9927604	0.99112	0.989482
0.2	0.99934	0.98617	0.97334	0.96085	0.9486707	0.9368	0.925223
0.3	1.00008	0.95615	0.91579	0.87861	0.844263	0.81246	0.782946
0.4	1.00565	0.90896	0.82704	0.75706	0.6968191	0.6446	0.599061
0.5	1.01221	0.85111	0.72597	0.62697	0.5474656	0.48277	0.429533
0.6	1.01401	0.78973	0.62924	0.51071	0.4210125	0.35179	0.29751
0.7	1.00978	0.73185	0.54656	0.41718	0.3237566	0.25458	0.202333
0.8	1.00104	0.68192	0.48089	0.34642	0.2526878	0.18538	0.135972
0.9	0.98992	0.64139	0.43106	0.29473	0.2020917	0.13704	0.090283
1	0.97814	0.60975	0.39425	0.25766	0.1665082	0.10351	0.058921
1.5	0.93391	0.53617	0.31676	0.18334	0.097263	0.03959	4.009E 06
2	0.91639	0.51867	0.30092	0.16926	0.084728	0.02836	0.010134



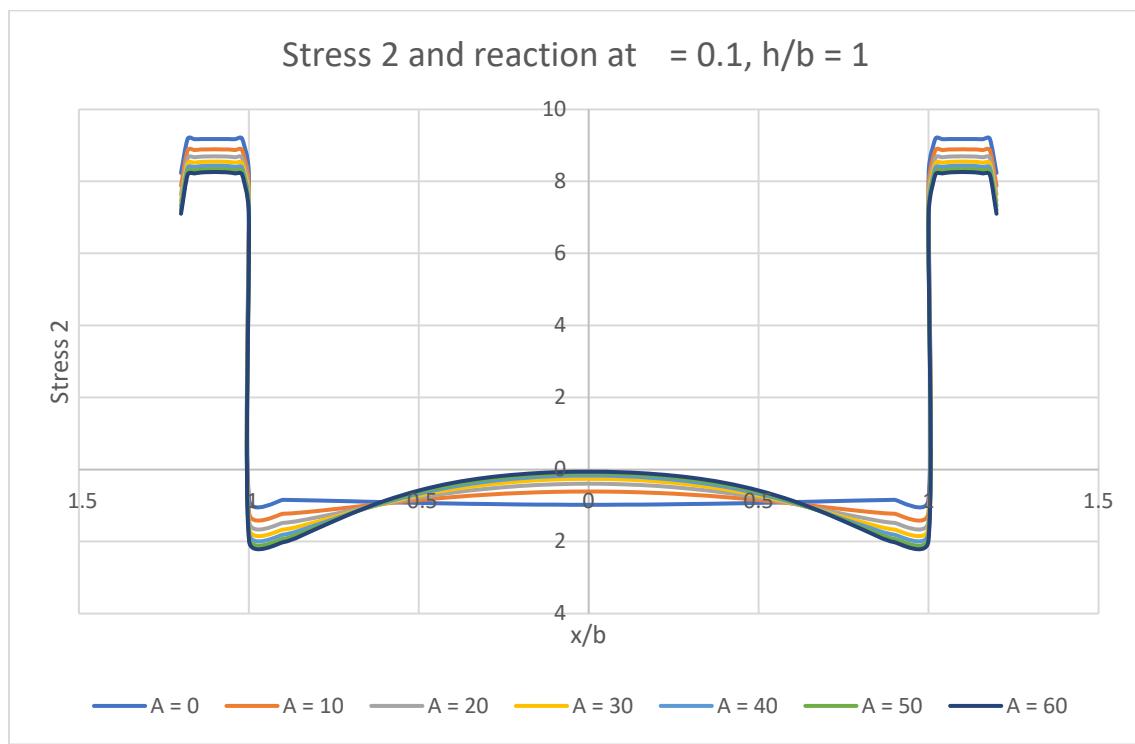
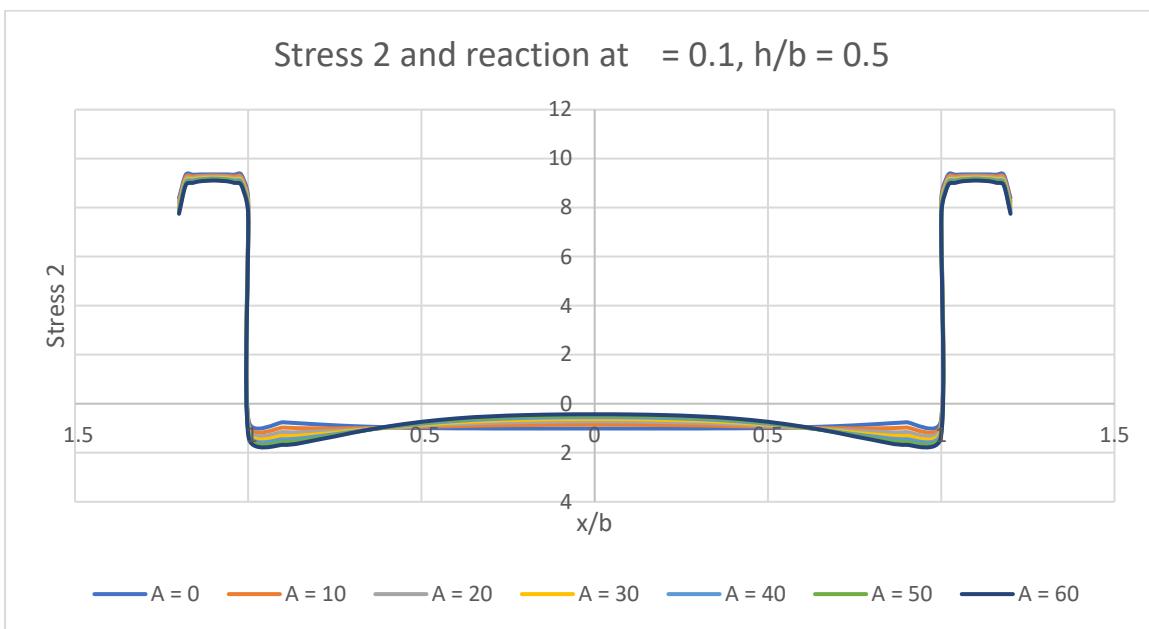
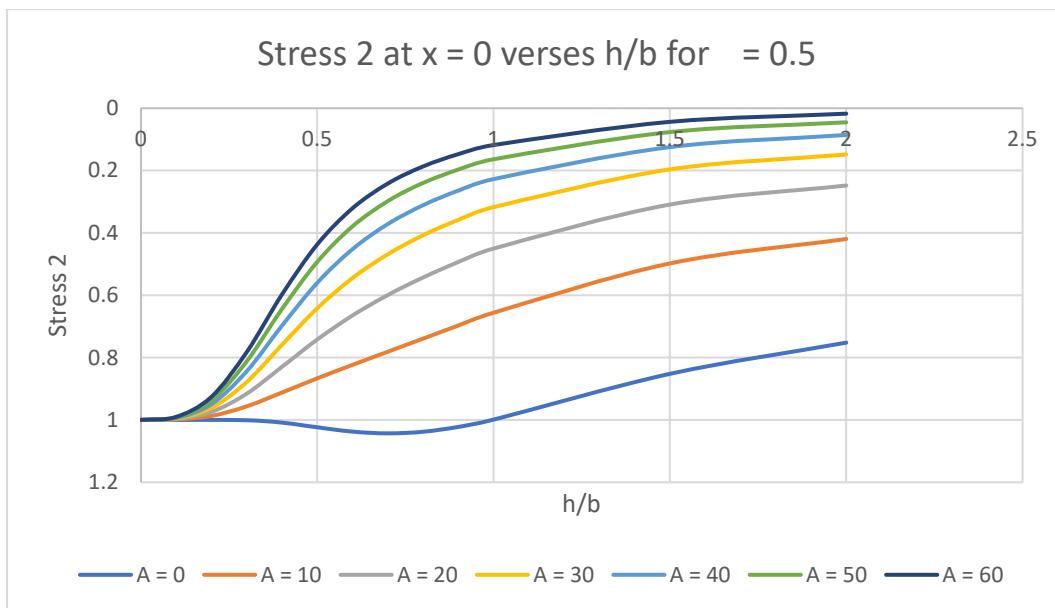
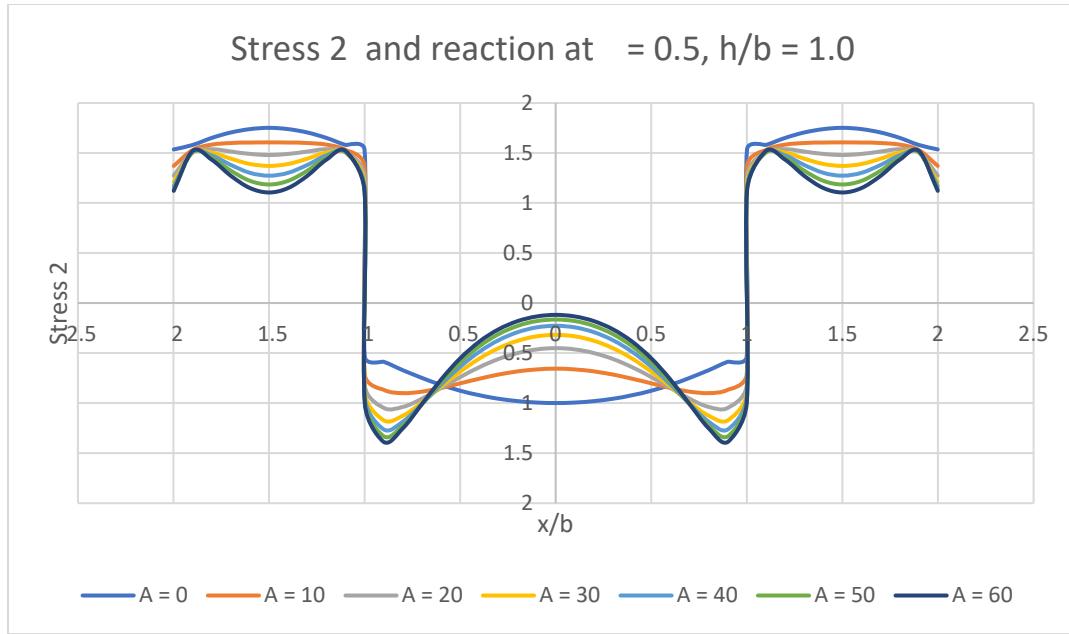
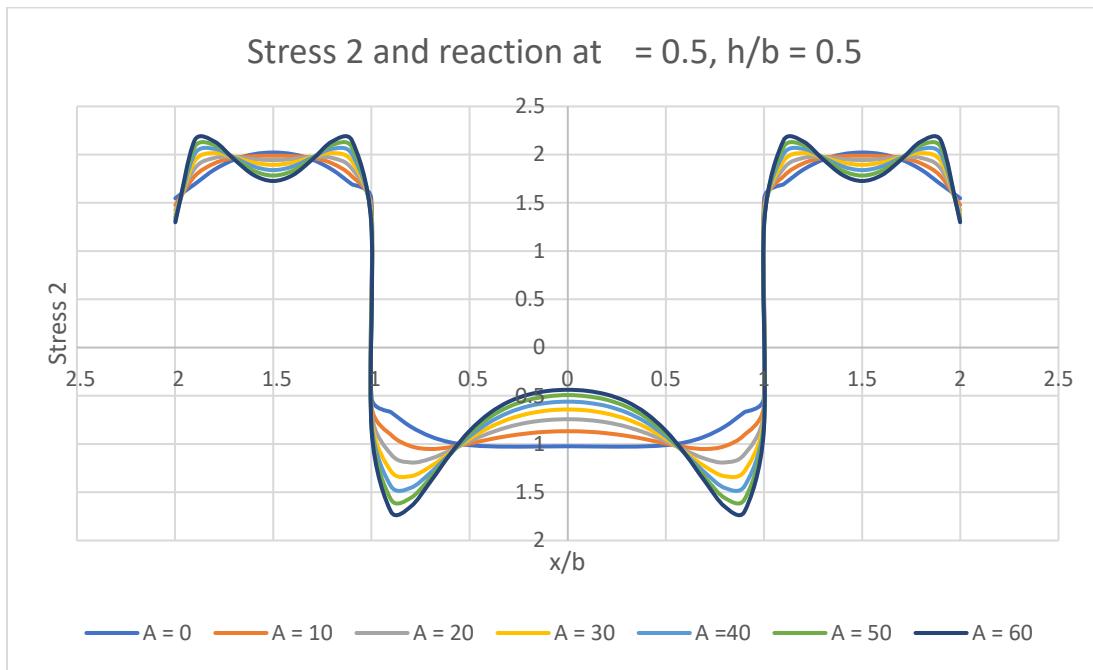


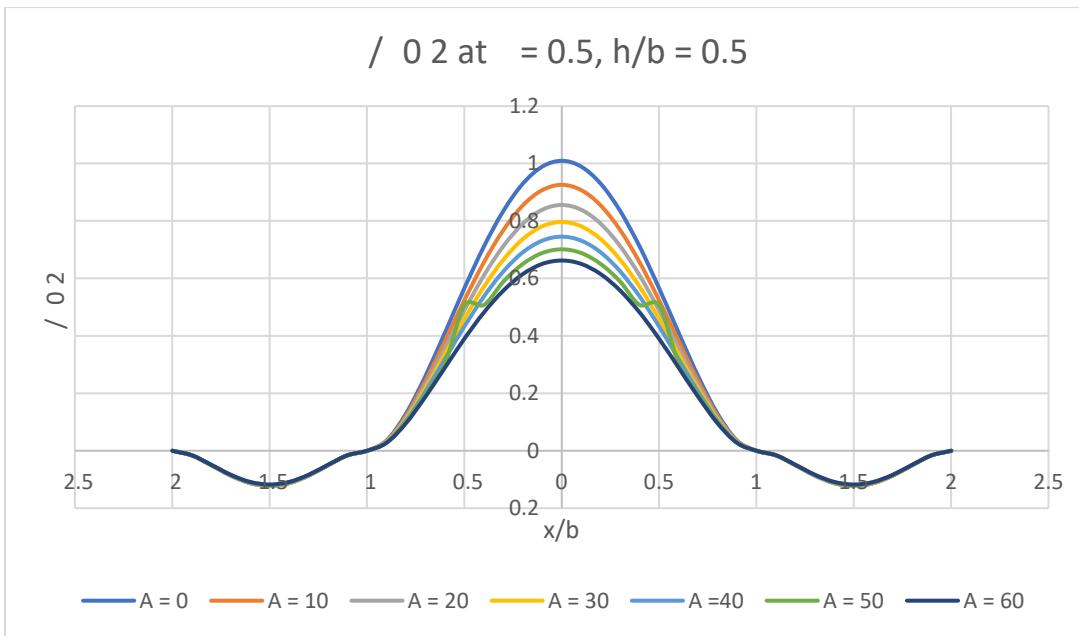
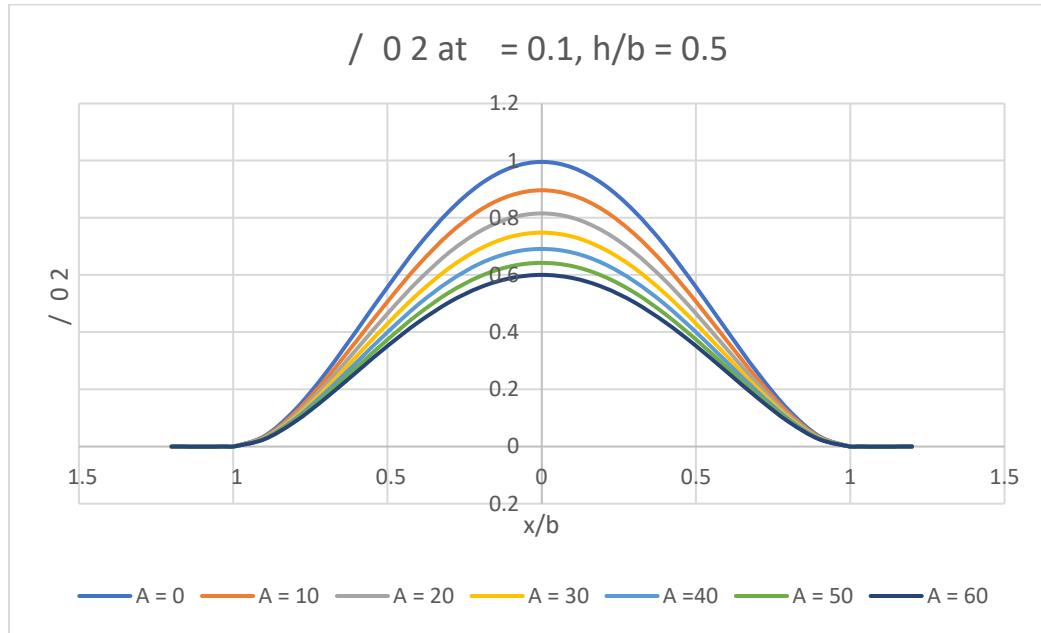
Table 37 Stress 2 at  $x = 0$  verses  $h/b$  for  $\gamma = 0.5$

$h/b$	$A = 0$	$A = 10$	$A = 20$	$A = 30$	$A = 40$	$A = 50$	$A = 60$
1E 13	0.99989	0.99989	0.99989	0.99989	0.9998898	0.99989	0.99989
0.1	0.99993	0.99826	0.9966	0.99495	0.9933039	0.99166	0.990025
0.2	0.99988	0.98671	0.97388	0.96138	0.9491991	0.93733	0.925749
0.3	1.00049	0.95634	0.91574	0.87831	0.8437173	0.81168	0.781942
0.4	1.00831	0.9119	0.8296	0.75884	0.6975885	0.64426	0.597577
0.5	1.0234	0.86624	0.74171	0.64156	0.5599918	0.49283	0.437022
0.6	1.03734	0.82317	0.66485	0.54483	0.4520182	0.37907	0.320973
0.7	1.04297	0.7812	0.59945	0.46853	0.3715531	0.29815	0.241633
0.8	1.03763	0.73901	0.54297	0.40772	0.3110309	0.24003	0.186802
0.9	1.02208	0.69685	0.49353	0.35826	0.2642978	0.19695	0.14753
1	0.99879	0.65584	0.45024	0.31747	0.2273597	0.16402	0.118324
1.5	0.85212	0.4977	0.30891	0.19644	0.12471	0.07687	0.043986
2	0.75185	0.41933	0.24809	0.14846	0.0861497	0.04535	0.017833





The deflection does not take into account the moment of inertia of the elastic media, so it is conservative. But for a media such as soils where tension is not allowed it is accurate and useful.



Once  $D$  is found the stresses with  $q = 0$  are:

$$\begin{aligned} \sigma_y = & -\sum_{n=\varphi}^{\infty} \cos \alpha_n x \left[ \left( a_n D + w_{n1} w_n + (b_n D + w_{n2} w_n) \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right) e^{\alpha_n y} \right. \\ & \left. + \left( c_n D + w_{n3} w_n + D \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} + \frac{2q \sin \alpha_n b}{\Gamma n \pi} \right] \quad -b < x < b \dots \dots 164 \end{aligned}$$

$$\begin{aligned}\sigma_x = k_0 \sum_{n=0}^{\infty} \cos \alpha_n x & \left[ \left( a_n D + w_{n1} w_n + (b_n D + w_{n2} w_n) \frac{\pi n}{(1+\Gamma)b} y \right) e^{\alpha_n y} \right. \\ & + \left( c_n D + w_{n3} w_n + D \frac{\pi n}{(1+\Gamma)b} y \right) e^{-\alpha_n y} - \frac{2q}{\Gamma} \frac{\sin \alpha_n b}{n\pi} \\ & + 2k_0 \sum_{n=0}^{\infty} \cos \alpha_n x [(b_n D + w_{n2} w_n) e^{\alpha_n y} - D e^{-\alpha_n y}] \quad -b < x < b \dots \dots 165\end{aligned}$$

$$\tau_{xy} = \sum_{n=\varphi}^{\infty} \sin \alpha_n x \left[ \left( a_n + b_n \right) D + w_{n1} w_n + w_{n2} w_n + \left( b_n D + w_{n2} w_n \right) \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right] e^{\alpha_n y} \\ + \left[ - (c_n - 1) D - w_{n3} w_n - D \frac{\pi n}{(1+\Gamma)} \frac{y}{b} \right] e^{-\alpha_n y} \quad -b < x < b \dots \dots 166$$

$$D = -\frac{s\alpha_n v_n}{\beta d_n} - \frac{W1_n}{d_n}$$

$$= -q \frac{2k_n A}{(a_n + c_n)} \left( \frac{\frac{W_n}{q} + \frac{2}{\Gamma} \frac{\sin \alpha_n b}{n\pi} + \frac{\pi n}{\Gamma(1+\Gamma)} \frac{C2}{0.5qb^2} \sin \alpha_n b}{\frac{(\pi n)^3}{(1+\Gamma)^3} + 2k_n A} \right) - \frac{W1_n}{d_n} \dots 167$$

Where:

$$W1_n = (w_{n1} - w_{n2} - w_{n3})w_n$$

$$a_n = -[c_n(1 + 2\alpha_n h) + 2\alpha_n^2 h^2]e^{-2\alpha_n h}$$

$$b_n = [2c_n - 1 + 2\alpha_n h]e^{-2\alpha_n h}$$

$$c_n = 1 + \frac{2\alpha_n^2 h^2 e^{-2\alpha_n h}}{(1 - 2\alpha_n h)e^{-2\alpha_n h} - 1}$$

$$d_n = a_n - b_n - c_n - 1$$

$$k_n = -\frac{a_n + c_n}{d_n}$$

$$W_n = -\frac{(a_n + c_n)(w_{n1} - w_{n2} - w_{n3})w_n}{(a_n - b_n - c_n - 1)} + (w_{n1} + w_{n3})w_n$$

$$w_{n1} = -w_{n3}(1 + 2\alpha_n h)e^{-2\alpha_n h} - (1 + \alpha_n h)e^{-\alpha_n h}$$

$$w_{n2} = 2w_{n3}e^{-2\alpha_n h} + e^{-\alpha_n h}$$

$$w_{n3} = \frac{\alpha_n h e^{-\alpha_n h}}{(1 - 2\alpha_n h)e^{-2\alpha_n h} - 1}$$

Rewrite for the excel sheet

$$\sigma_y = - \sum_{n=\varphi}^{\infty} \cos \alpha_n x \left[ \left( a_{n1} D + w_{n11} w_n + (b_{n1} D + w_{n21} w_n) \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) + \left( c_n D + w_{n3} w_n + D \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} + \frac{2q \sin \alpha_n b}{\Gamma} \frac{y}{n\pi} \right] \quad -b < x < b \dots \dots 168$$

$$\begin{aligned} \sigma_x = k_0 \sum_{n=\varphi}^{\infty} \cos \alpha_n x & \left[ \left( a_{n1} D + w_{n11} w_n + (b_{n1} D + w_{n21} w_n) \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) + \left( c_n D + w_{n3} w_n + D \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} - \frac{2q \sin \alpha_n b}{\Gamma} \frac{y}{n\pi} \right] \\ & + 2k_0 \sum_{n=\varphi}^{\infty} \cos \alpha_n x [(b_{n1} D + w_{n21} w_n) - D e^{-\alpha_n y}] \quad -b < x < b \dots \dots 169 \end{aligned}$$

$$\begin{aligned} \tau_{xy} = \sum_{n=\varphi}^{\infty} \sin \alpha_n x & \left[ \left( (a_{n1} + b_{n1}) D + w_{n11} w_n + w_{n21} w_n + (b_{n1} D + w_{n21} w_n) \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) e^{\alpha_n y} \right. \\ & \left. + \left( -(c_n - 1) D - w_{n3} w_n - D \frac{\pi n}{(1 + \Gamma)} \frac{y}{b} \right) e^{-\alpha_n y} \right] \quad -b < x < b \dots \dots 170 \end{aligned}$$

$$a_{n1} = a_n e^{\alpha_n h} = -[c_n(1 + 2\alpha_n h) + 2\alpha_n^2 h^2] e^{-\alpha_n h}$$

$$b_{n1} = b_n e^{\alpha_n h} = [2c_n - 1 + 2\alpha_n h] e^{-\alpha_n h}$$

$$w_{n11} = w_{n1} e^{\alpha_n h} = -w_{n3}(1 + 2\alpha_n h)e^{-\alpha_n h} - (1 + \alpha_n h)$$

$$w_{n21} = w_{n2} e^{\alpha_n h} = 2w_{n3} e^{-\alpha_n h} + 1$$

### Example 6:

In example 2 we used a 30 ft wall to approximate an infinite wall. When redoing the calculation with  $h/b = 15$  and  $A = 141$   $Rm_- = 0.2846$  is close to the example = 0.28453 but  $Rv = 0.59241$  verses 0.592369. For  $A = 20$   $Rm_- = 0.6484456$  is close to the example  $Rm_- = 0.648373$  but  $Rv = 0.833056$  verses 0.833077. Part of the reason is in example 1,  $n$  max was 20,000 where in the posted excel sheet  $n$  max was 5,000 and  $h$  has an effect.

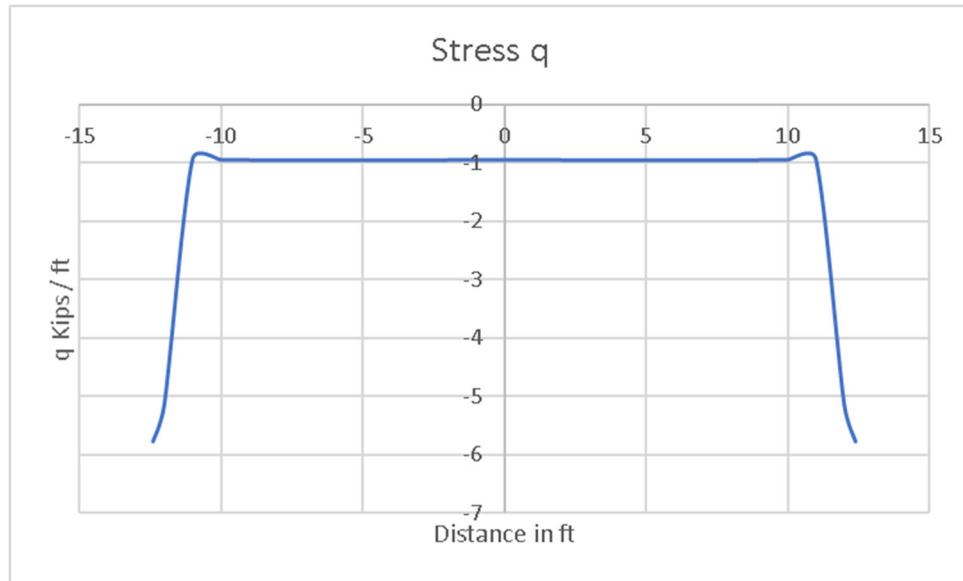
### Example 7:

A concrete culvert has a top deck slab 15 ft span fixed end 12" thick with 10 ft of soils on top. The load is 100 psf live load and dead load of soils + beam =  $120 \times 10 + 150 = 1350$  psf we use  $s = 12"$   $= 0.3$   $1/\beta = E_s/(1-v^2) = 1111$  psi  $E_c = E = 57000$   $3000 = 3,122,018.58$  psi. So,  $A = 0.900769$ ,  $h/b = 1.3333$  and  $t/b = 0.5$ .  $Rm_1 = 0.9770204$   $Rm_2 = 0.8005702$ . So factored negative moment =  $1.6 * (1.350 * 0.9770204 + 0.100 * 0.8005702) * 15 * 15 / 12 = 41.97$  Kips ft.  $Rm_1 + = 0.9639418$   $Rm_2 + = 0.8281587$ . The factored positive moment =  $1.6 * (1.350 * 0.9639418 + 0.100 * 0.8281587) * 15 * 15 / 24 = 20.76$  Kips ft. The shear at  $x = 0.867b$   $Rv_1 = 0.8526281$   $Rv_2 = 0.6817198$ . The factored shear =  $1.6 * (1.350 * 0.8526281 + 0.100 * 0.6817198) * 15 / 2 = 14.63$  kips and the top deck ready to be designed provide adding the axial load. We note that all the stresses are in compression in the region  $b < x < b$  and  $0 < y < 9.75$  ft. And because we assumed  $t/b = 0.5$  for  $x > b$  the deflection is zero then for  $x > b$ , all stresses are in compression and for  $b < x < b$  is all compression then this solution is applicable for soil.

### Example 8:

An existing floor sitting on a steel beam W14x30. The span is 25 ft  $q = 1.6$  kip/ft was designed with ASD the old methods. It is a fixed end beam. Thus,  $M_{max} = 1.6 * 25^2 / 12 = 83.33$  kip ft. The required section modulus for a compact beam is =  $83.33 * 12 / 24 = 41.67 < 42$  in $^3$  The properties of the beam  $A = 8.85$  in $^2$   $S_x = 42$  in $^3$  and  $I = 291$  in $^4$ . The beam sits on HSS6x6 column.

In a renovation the floor was elevated 6 inches by the architect request so a 6-inch plate one inch thick was used to tack weld to the steel beam usually it is a T section. We will demonstrate the load is not uniform anymore and arching reduction takes place. We have  $A = 5799$   $t/b = 0.02$   $b = 12.5$   $v = 0.3$ .  $E_s = E = 29,000,000$  psi and they cancel each other in  $A$ . We find that  $Rm_{2-} = 0.9679295$  a 3.2% reduction. See graph for the new distribution of stress, so it is not uniform anymore.



### Example 9:

A glulam beam is fixed at the ends. The beam is 5.5" x 24" spans 40 ft to support a floor load  $q = 468$  lbs/ft and the weight of the beam  $p = 32.08$  lbs/ft. Thus, the moment is  $(q + p)L^2/12 = 66,677.78$  ft-lbs. The required section modulus for  $F_b = 2400$  psi is  $333.39 \text{ in}^3$   $F_v = 165$  psi and the required area =  $81.83 \text{ in}^2$  the shear a 2 ft from the support or  $0.9b$  is  $9001.44$  lbs this is with no arching. The section modulus of the beam is  $528 \text{ in}^3$  and the area =  $132 \text{ in}^2$ , the beam was chosen to give 0.51-inch deflection at the center, so the beam is OK using conventional methods. Now we consider arching so we take 17 inches for  $h$  the mass media and 7 inch is the beam under the mass media. Giving  $I = 157.21 \text{ in}^4$  and  $E$  cancels  $E_s$  so  $A = 241819.2$  with  $s = 5.5$ -inch  $b = 240$  inches. We assume the column is a HSS3x3 with a 4"x 4" plate for a 20003.2 lbs reaction. Thus  $\Gamma = t/b = 2/240 = 0.008333$  and  $h/b = 17/240 = 0.0708$ . This gives  $Rm1 = 0.1232561$  and  $Rv1 = 0.2922825$  taken at  $x = 0.9708b$   $Rm2 = 0.1230694$  and  $Rv2 = 0.2919907$ . The section modulus for a 7-inch beam is  $44.92 \text{ in}^3$  and the area =  $38.5 \text{ in}^2$ . Thus, the moment =  $(32.03 * 0.1232561 + 468 * 0.1230694) * 40^2 / 12 = 8205.91$  lbs ft we get the required new section modulus of  $41.03 < 44.92 \text{ in}^3$ . For the shear at  $x = 0.9708b$  is  $V = (32.03 * 0.2922825 + 468 * 0.2919907) * 40/2 = 2920.27$  lbs we get the required area =  $26.55 \text{ in}^2 < 38.5$ . The strength is fine. If we look at the stress at  $x = 19.97$  ft,  $y = 0$  and is also close to  $y = 17$  inch  $x 1/p = 106.948$  and  $x 2/q = 356.929$ . Or  $f_b = (32.03 * 106.948 + 468 * 356.929)/(12 * 5.5) = 2582.85$  psi  $> 2400$  psi this is also on top of the elastic media so the  $f_b$  of the beam without arching =  $66,677.78 * 12/528 = 1515.24$  psi. In here  $v$  was taken as 0.3 so  $K_0 1 = 0.3$  and  $K_0 2 = 1.0$ . If we look at  $K_0 2 = 0.3$  for a rough surface contact, we have  $x 1/p = 106.948$  and  $x 2/q = 107.079$ . Or  $f_b = (32.03 * 106.948 + 468 * 107.079)/(12 * 5.5) = 811.188$  psi  $< 1515.24$  psi and the beam appears stronger. Even if we make the beam 10-inch high and the elastic media 14-inch high for maximum stress,  $A = 82944$  the stress at  $x = 19.97$ ,  $y = 0$  and is also close to  $y = 17$  inch  $x 1/p = 111.087$  and  $x 2/q = 370.808$ . Or  $f_b = (32.03 * 111.087 + 468 * 370.808)/(12 * 5.5) = 2683$  psi this is with  $K_0 2 = 1.0$ . It is rare to have a smooth surface and  $K_0 2$  is not 1.0. If we look at  $K_0 2 = 0.3$  for a rough surface contact, we have  $x 1/p = 111.087$  and  $x 2/q = 111.242$ . Or  $f_b = (32.03 * 111.087 + 468 * 111.242)/(12 * 5.5) = 842.72$  psi and the beam appear stronger.

**Note: to calculate the reserve stress in a beam due to arching it is case by case because every beam is different.**

**Other solutions:**

If solve for the coefficients  $A$ ,  $B$ ,  $C$  and  $D$  by setting  $\sigma_x$  at  $x = t+b$  equal zero instead of setting the shear at  $y = 0$  equal to zero and proceed with the derivation we can obtain the solution of Figure 4. If this is not possible than subtracting  $\sigma_x$  at  $x = t+b$  is required. It is expected there will be more deflections and arching. Also, for unequaled span one can use the shorter span as an approximation to find the solution the domain of the Fourier series must be revised.

**Future Work:**

The solution for seismic moment will introduce a Fourier sine series in the deflection and the entire solution must be revised. It is left for future articles to cover other conditions such as a point load for live load and moments in the beam. Also, if add a beam at  $y = h$  we can solve the problem for a wide flange beam where the flanges are the beams and the web is the mass media.

**Conclusion:**

It is evident from the charts that  $h$  is proportional to the arching reduction obtained. This article is worthy of or a candidate for a Ph. D. dissertation. The future research would involve systems' optimization with arching for any beam. The dividends of this research will produce great benefits for structural designs, not just for the Army Corps of Engineers, but for all Federal agencies, including the Forest Service, Aerospace Engineering (Air force), and Naval Architectures (Navy). The application in civil engineering includes culverts with ribs, tunneling with ribs, bridges with ribs, masonry buildings, concrete buildings, steel buildings and timber structures. The application in Aerospace and Naval Architectures is in reducing the weight of materials.

Reducing construction materials reduces energy consumption which reduces construction costs. In shoring and excavation systems, a 50% reduction from the empirical formula due to arching gives a 30% ( $\sqrt{.5}$ ) reduction in materials (i.e. wood lagging over the entire face of the wall). This results in, approximately, a 15% reduction in total cost. Preliminary designs for deep culverts show 20% reduction in construction cost due to utilizing arching. If there is a reduction of 20% due to arching, as realized in the above charts for a moderate  $h$  for live or dead loads, then an average of 10% ( $\sqrt{.8}$ ) reduction in materials can be achieved for most applications.

**Can we say we caution most buildings in the world are safe because arching was never considered provided, they are designed with some code?**

**Acknowledgements:**

Glory be to God. Thank you, Lord Jesus Christ, for giving me the mathematical solution in my meditation and help me afterward to understand the physics.

## **Appendix 1.-References**

1. M. S. Aggour and C. B. Brown, "Analytical Determination of Earth Pressure Due to Compaction", Proceedings of the Third International Conference on Numerical Methods in Geomechanics/ AACHEN/ 2-6 April 1979, pp. 1167-1174.
2. Armento, William, "Design and Construction of Deep Retained Excavations, "ASCE/SEAONC, Continuing Education Seminars, November 1970.
3. W. D. Liam Finn, "Boundary Value Problems of Soil Mechanics", ASCE SM & FE #5 Journal of the Soil Mechanics and Foundations Division, September 1963, p. 3648
4. Richard L. Handy "The Arch In Soil Arching" ASCE Journal of Geotechnical Engineering, March 1985, p302.
5. La Croix, Y. and Jackson, W., "Design and Construction of Support Temporary Excavations in Urban Environment", 3rd Ohio Soil Seminar, October 1972.
6. U. S. Department of Commerce, National Technical Information Service, PB-257 212, "Lateral Support Systems and Underpinning Vol II & III: Construction Methods" D. T. Goldberg, et al. Goldberg-Zoino & Associate, Inc., Newton Upper Falls, Mass., April, 1976, p. 29, Sect. 2.32.2, Vol. III and p. 49, Vol. II.
7. White, E. E., "Underpinning", Foundation Engineering, ed G. A. Leonards, McGraw-Hill, pp. 826-893, 1962.